

Introduction to Cryogenics for accelerators

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Préambule

Reference

Great thanks to predecessors for this type of exercice, particularly to Ph. Lebrun and his “legacy” of slides

Disclaimer

Being an engineer and new in this domain as “teacher”, I will try to share with you some information with emphasis on “applied cases” with a “pragmatic approach” rather than only a theoretical one.

There are plenty of books, previous CAS courses with lot's of formulas and various equations. I leave it to you to check bibliography if this is what you are looking for!

Contents

- Introduction
- Cryogenic fluids
- Heat transfer & thermal insulation
- Cryogenic distribution & cooling schemes
- Refrigeration & liquefaction
- Various complements
- Concluding remarks, references

- **cryogenics**, that branch of physics which deals with the production of very low temperatures and their effects on matter

Oxford English Dictionary

2nd edition, Oxford University Press (1989)

- **cryogenics**, the science and technology of temperatures below 120 K

New International Dictionary of Refrigeration

3rd edition, IIF-IIR Paris (1975)

Temperature in Celsius (C): unit defined with 0 C (ice) and 100 C (vapour)

Temperature in Kelvin (K): 1 K = 1 C, but 0 K = -273.15 C (absolut zero)

Densification, liquéfaction & séparation des gaz

LNG



130 000 m³ LNG carrier
with double hull

Air separation by cryogenic
distillation

Up to 4500 t/day LOX

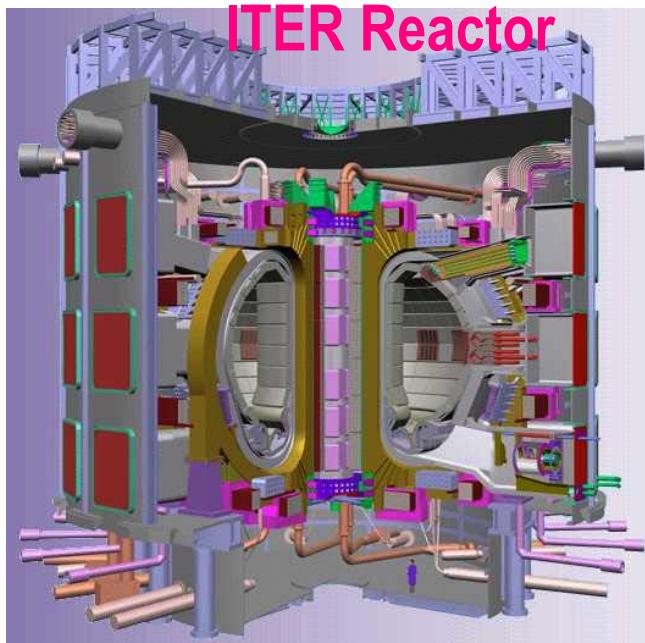


Rocket fuels



Ariane 5
25 t LHY, 130 t LOX

Cooling of superconducting devices



Main reasons to superconducting

For accelerators in high energy physics

- Compactness through higher fields

$$E_{\text{beam}} \approx 0.3 \cdot \mathbf{B} \cdot r$$

[Gev] [T] [m]

$$E_{\text{beam}} \approx \mathbf{E} \cdot L$$

[Gev] [MV/m] [m]

Capital Cost

At design stage, working at highest possible temperature is always considered, but often not selected to maximise beam energy ...

Cryogenic systems takes longer to recover from failures than conventional ones !
(but there is work on it!)

- Saving operating energy

Electromagnets:

Resistive: $P_{\text{input}} \approx E_{\text{beam}}$

Superconducting: $P_{\text{input}} \approx P_{\text{ref}}$

Operating Cost

Acceleration cavities

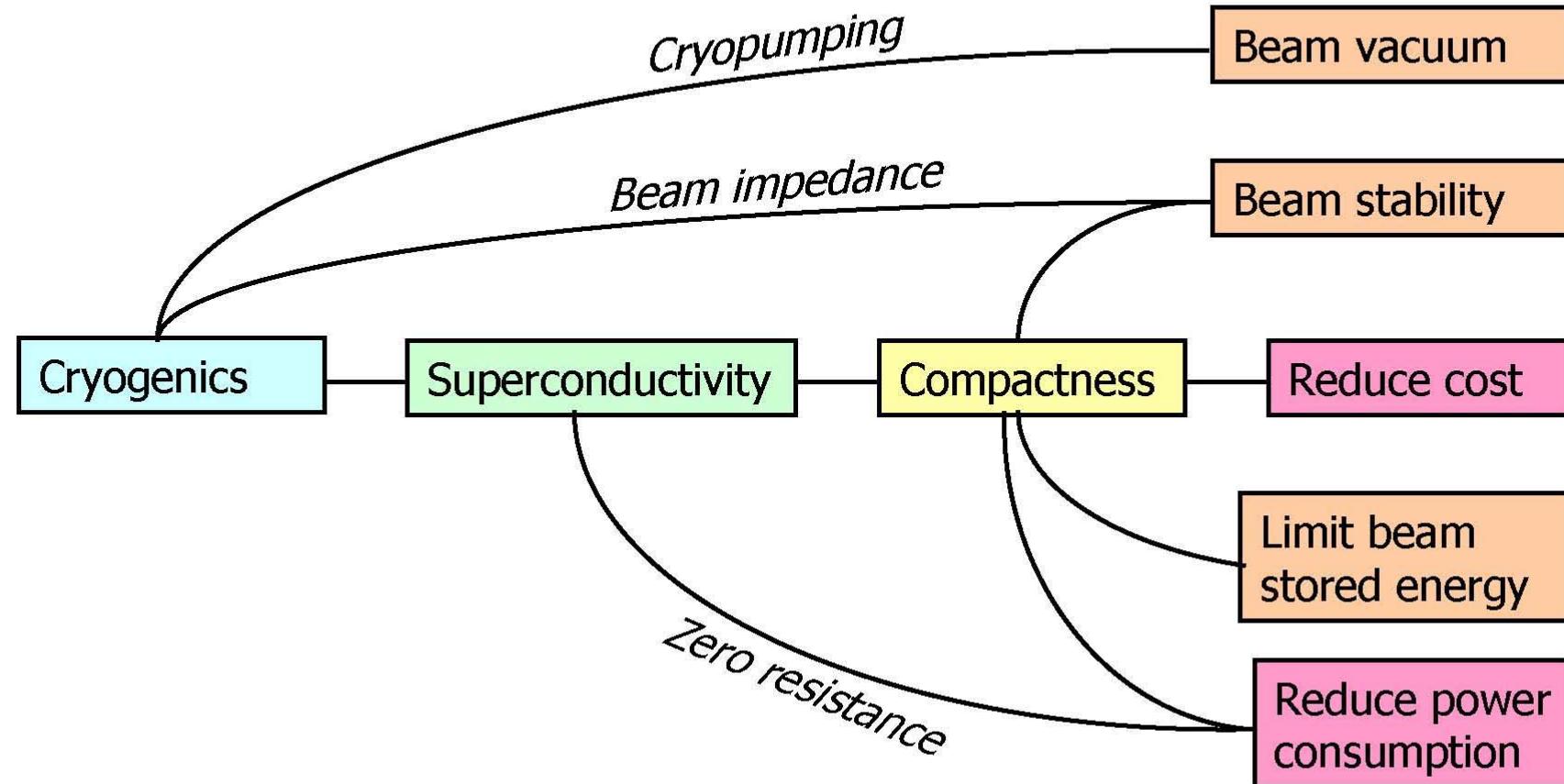
$$P_{\text{input}} \approx R_s \cdot L \cdot \mathbf{E}^2 / w$$

$$R_s \approx R_{\text{BCS}} + R_o$$

$$R_{\text{BCS}} \approx (1/T) \exp(-B T_c/T)$$



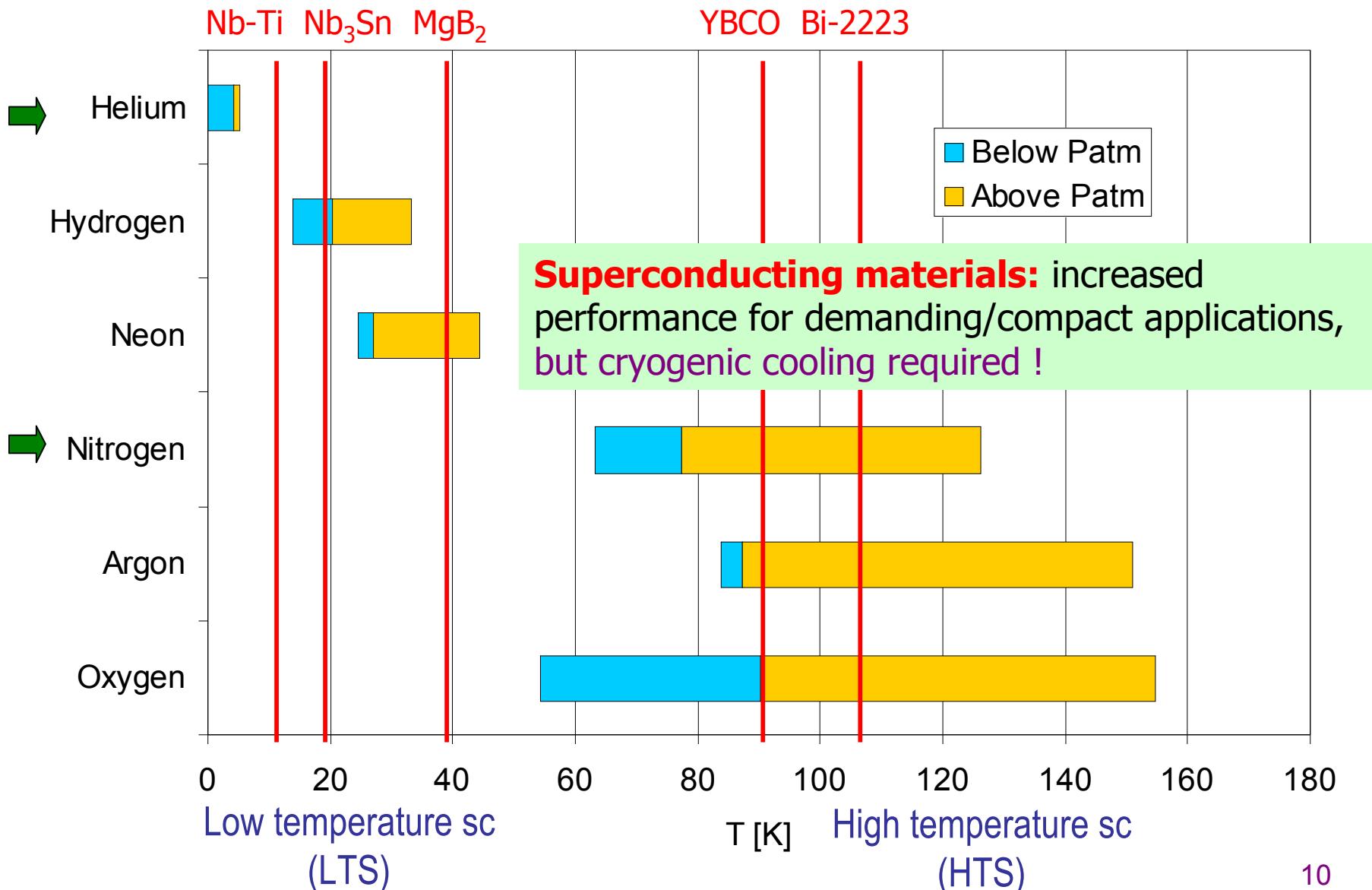
Rationale for superconductivity & cryogenics in particle accelerators



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Useful range of cryogens, and potential applications



Characteristic temperatures of cryogens

Cryogen	Triple point [K]	Normal boiling point [K]	Critical point [K]
Methane	90.7	111.6	190.5
Oxygen	54.4	90.2	154.6
Argon	83.8	87.3	150.9
Nitrogen	63.1	77.3	126.2
Neon	24.6	27.1	44.4
Hydrogen	13.8	20.4	33.2
Helium	2.2 (*)	4.2	5.2

(*): λ point

Properties of cryogens compared to water

Property	He	N ₂	H ₂ O
Normal boiling point	[K]	4.2	77
Critical temperature	[K]	5.2	126
Critical pressure	[bar]	2.3	34
Liq./Vap. density (*)		7.4	175
Heat of vaporization (*)	[J.g ⁻¹]	20.4	199
Liquid viscosity (*)	[μ Pa.s]	3.3	152

(*) at normal boiling point

Vaporization of normal boiling cryogens under 1 W applied heat load

Power \approx m' . Latent_Heat

[W] [g/s] [J/g]

Cryogen	[mg.s ⁻¹]	[l.h ⁻¹] (liquid)	[l.min ⁻¹] (gas NTP)
Helium	48	1.38	16.4
Nitrogen	5	0.02	0.24

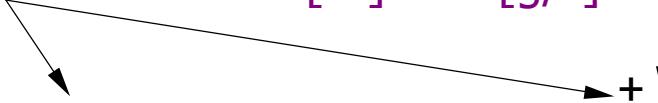
Amount of cryogens required to cool down 1 kg iron

$$\text{Power} \approx m' \cdot \text{Latent_Heat}$$

[W] [g/s] [J/g]

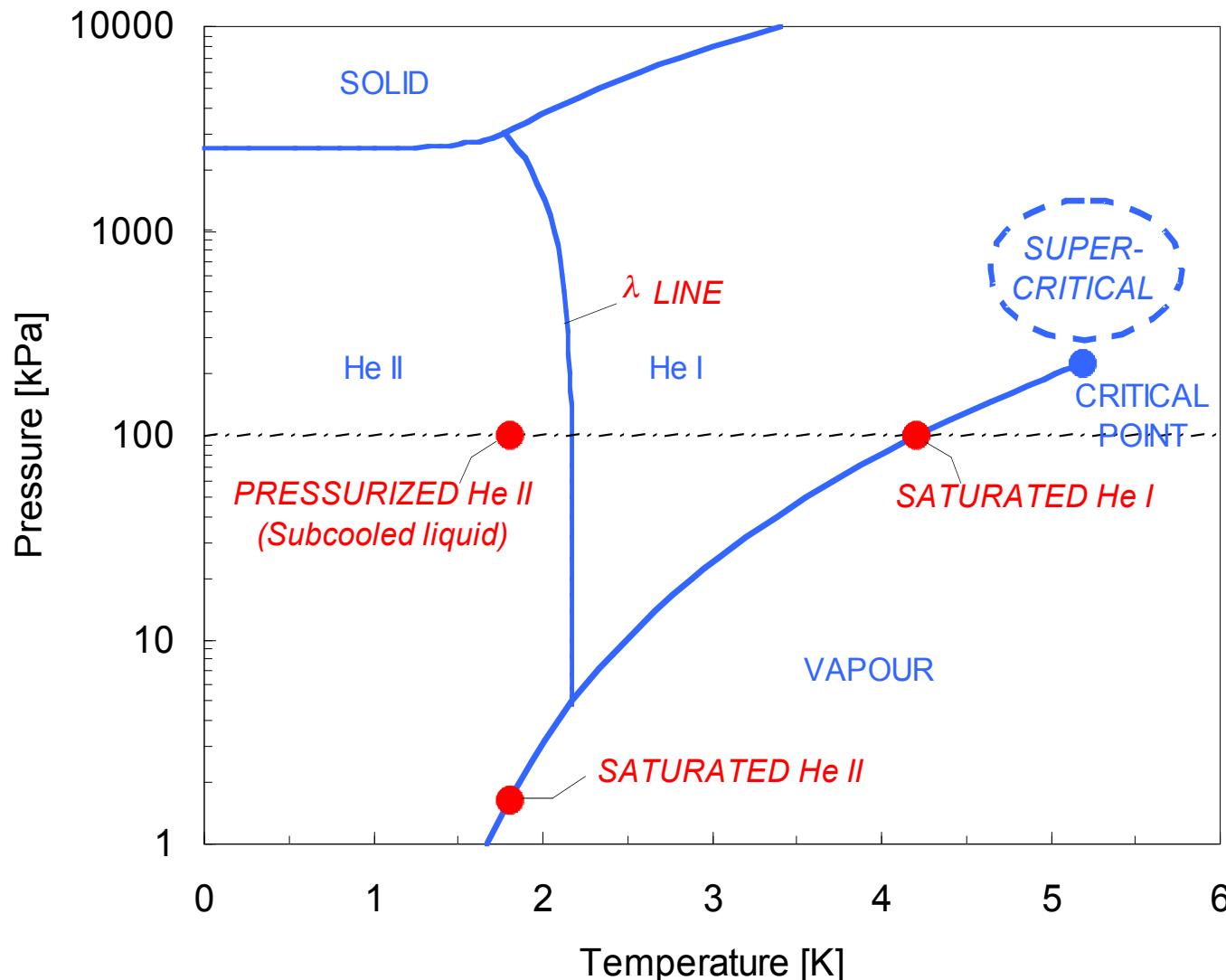
$$\text{Power} \approx m' \cdot \text{Specific_Heat} \cdot \Delta T$$

[W] [g/s] [J/g.K] [K]



Using	Latent heat only	Latent heat and enthalpy of gas
LHe from 290 to 4.2 K	29.5 litre	0.75 liter
LHe from 77 to 4.2 K	1.46 litre	0.12 litre
LN2 from 290 to 77 K	0.45 litre	0.29 litre

Phase diagram of helium



Helium as a cooling fluid

Phase domain	Advantages	Drawbacks
Saturated He I	Fixed temperature High heat transfer	Two-phase flow Boiling crisis
Supercritical	Monophase Negative J-T effect	Non-isothermal Density wave instability
He II	Low temperature High conductivity Low viscosity	Second-law cost Subatmospheric

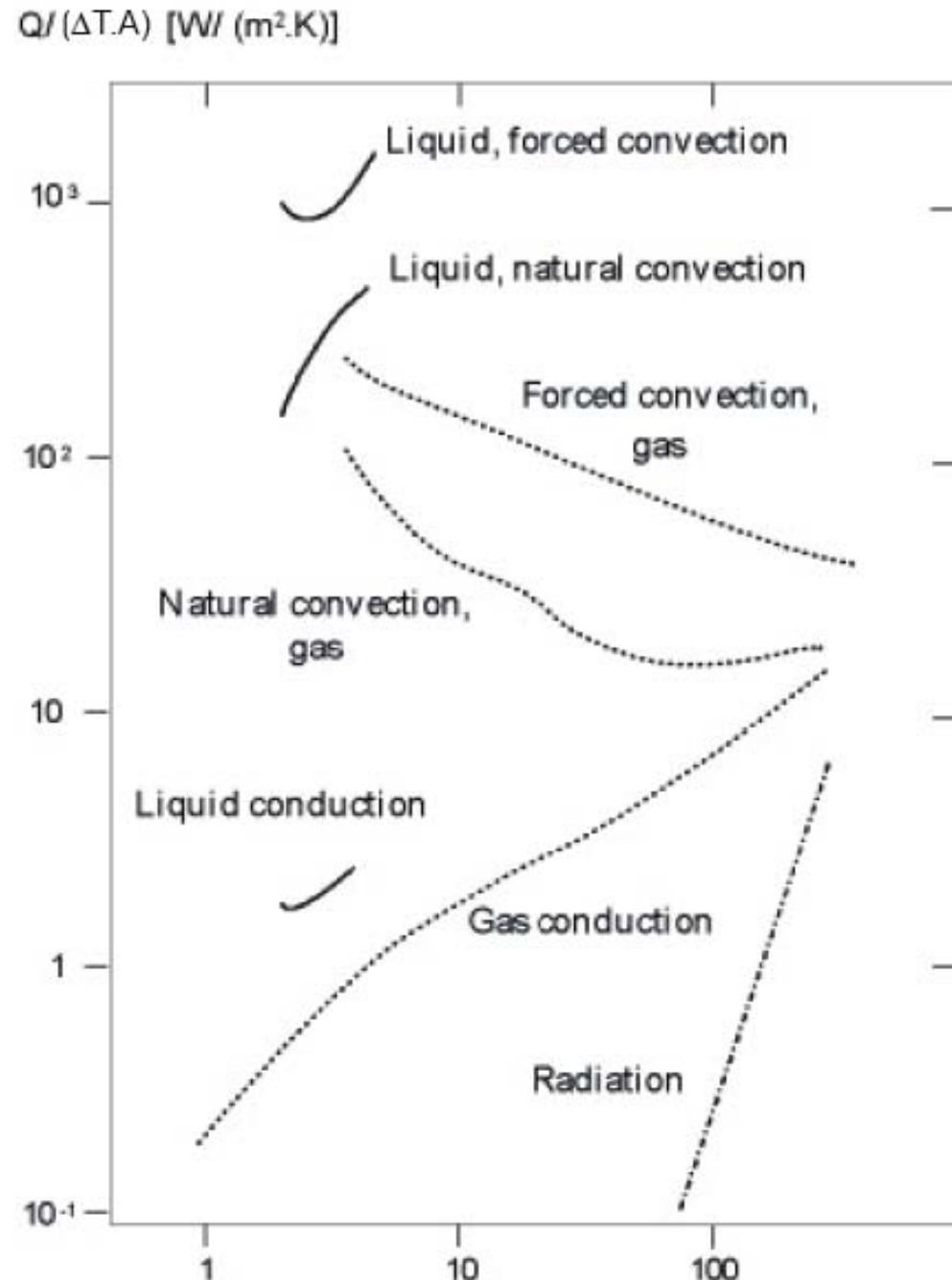
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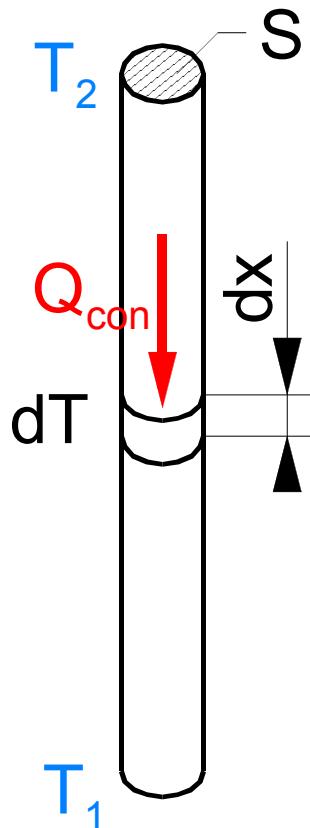
Typical heat transfer coefficients at cryogenic temperatures

3 mechanisms involved:

- Conduction
- Radiation
- Convection



Heat conduction in solids



Fourier's law:

$$Q_{\text{con}} = k(T) \cdot S \cdot \frac{dT}{dx}$$

$k(T)$: thermal conductivity [W/m.K]

Integral form:

$$Q_{\text{con}} = \frac{S}{L} \cdot \int_{T_1}^{T_2} k(T) \cdot dT$$

$\int k(T) \cdot dT$: thermal conductivity integral [W/m]

Thermal conductivity integrals for standard construction materials are tabulated

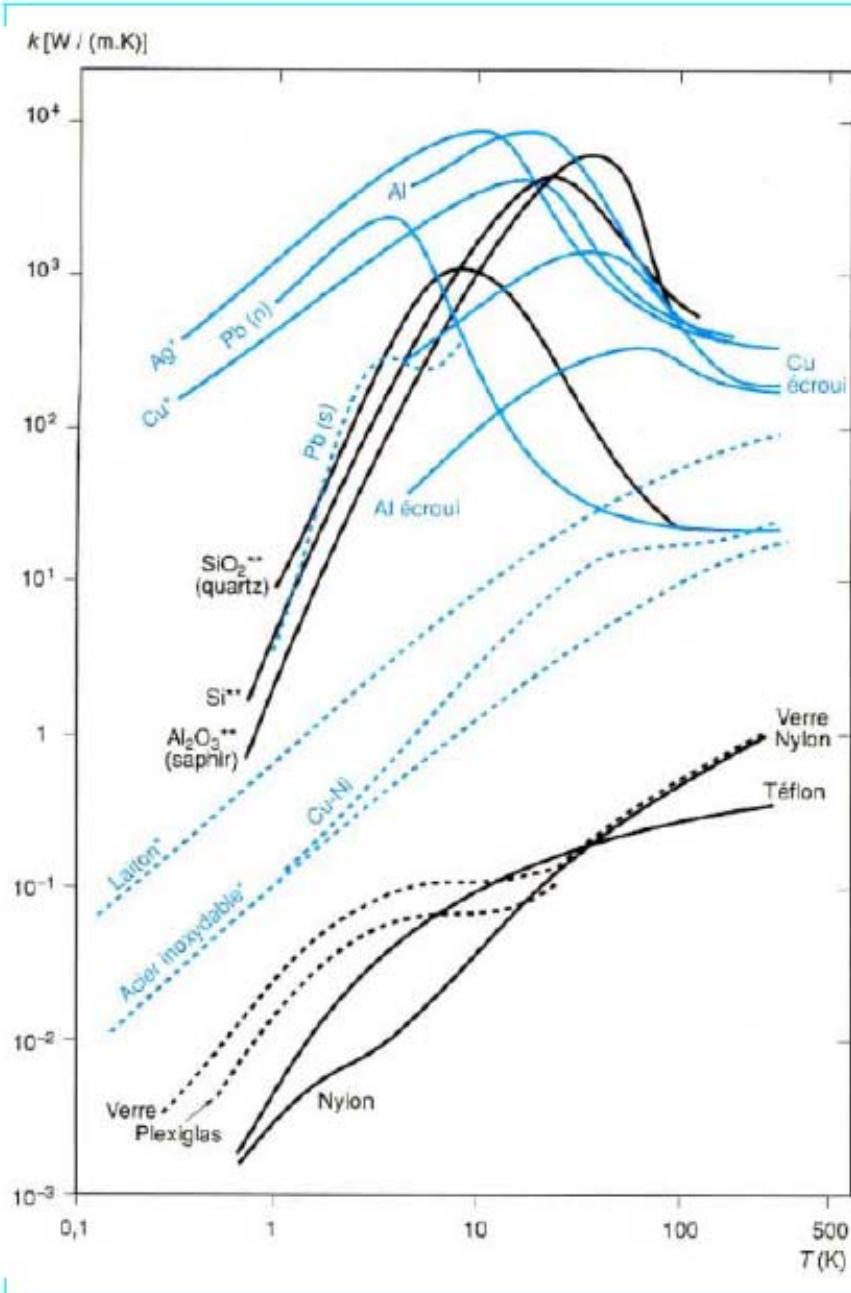
Risks associated with "optimisation":

- small section S : towards limit for material resistance
- long length L : towards limits for mechanical stability
- insulators (large) K : difficulties with transfer of forces

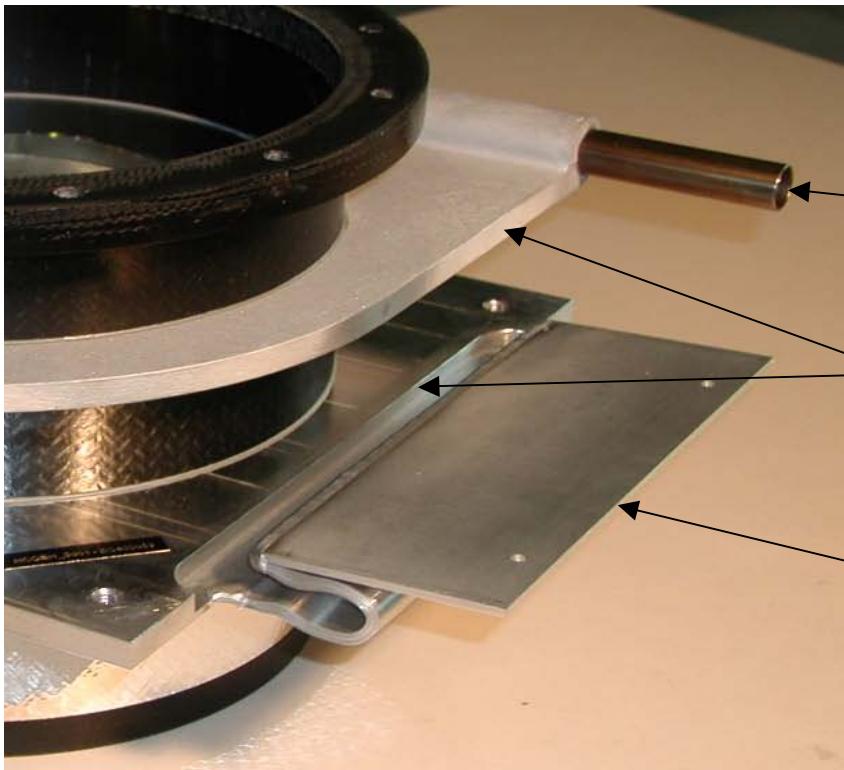
Thermal conductivity integrals, selection of materials [W/m]

From vanishingly low temperature up to	20 K	80 K	290 K
OFHC copper	11000	60600	152000
DHP copper	395	5890	46100
1100 aluminium	2740	23300	72100
2024 aluminium alloy	160	2420	22900
AISI 304 stainless steel	16.3	349	3060
G-10 glass-epoxy composite	2	18	153

Thermal conductivity of materials at cryogenic temperatures



Non-metallic composite support post with heat intercepts for LHC magnets



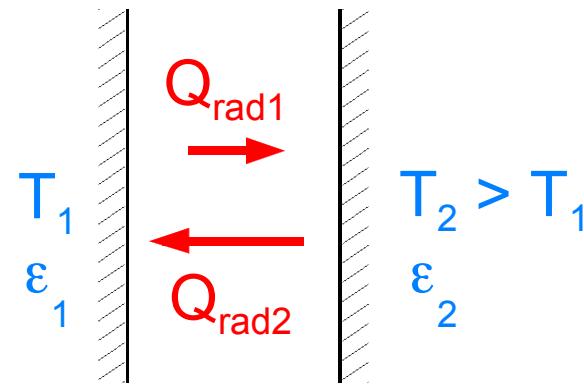
5 K cooling line (SC He)

Aluminium intercept plates glued to G-10 column

Aluminium strips to thermal shield at 50-75 K

Thermal radiation

- Wien's law
 - Maximum of black body power spectrum
 $\lambda_{max} \cdot T = 2898 \text{ [\mu m.K]}$
- Stefan-Boltzmann's law
 - Black body
 - "Gray"body
 - "Gray" surfaces at T_1 and T_2



$$Q_{rad} = \sigma A T^4$$

$$\sigma = 5.67 \times 10^{-8} \text{ W/m}^2 \cdot \text{K}^4$$

(Stefan Boltzmann's constant)

$$Q_{rad} = \varepsilon \sigma A T^4$$

ε emissivity of surface

$$Q_{rad} = E \sigma A (T_1^4 - T_2^4)$$

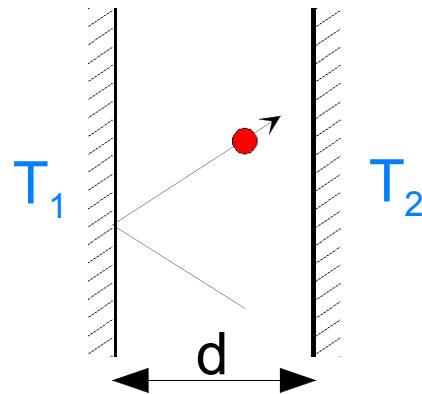
E function of $\varepsilon_1, \varepsilon_2$, geometry

$E \cdot T^4$

Best would be to have a reflective (high E) "parasol" to intercept T4 ...

Emissivity of technical materials at low temperatures

	Radiation from 290 K Surface at 77 K	Radiation from 77 K Surface at 4.2 K
Stainless steel, as found	0.34	0.12
Stainless steel, mech. polished	0.12	0.07
Stainless steel, electropolished	0.10	0.07
Stainless steel + Al foil	0.05	0.01
Aluminium, as found	0.12	0.07
Aluminium, mech. polished	0.10	0.06
Aluminium, electropolished	0.08	0.04
Copper, as found	0.12	0.06
Copper, mech. Polished	0.06	0.02



Residual gas conduction

$\lambda_{molecule}$: mean free path of gas molecules

- Viscous regime
 - At high gas pressure $\lambda_{molecule} \ll d$
 - Classical conduction $Q_{res} = k(T) A dT/dx$
 - Thermal conductivity $k(T)$ independant of pressure
- Molecular regime
 - At low gas pressure $\lambda_{molecule} \gg d$
 - Kennard's law $Q_{res} = A \alpha(T) \Omega P (T_2 - T_1)$
 - Conduction heat transfer proportional to pressure, independant of spacing between surfaces
 Ω depends on gas species
 - Accommodation coefficient $\alpha(T)$ depends on gas species, T_1 , T_2 , and geometry of facing surfaces

Multi-layer insulation (MLI)

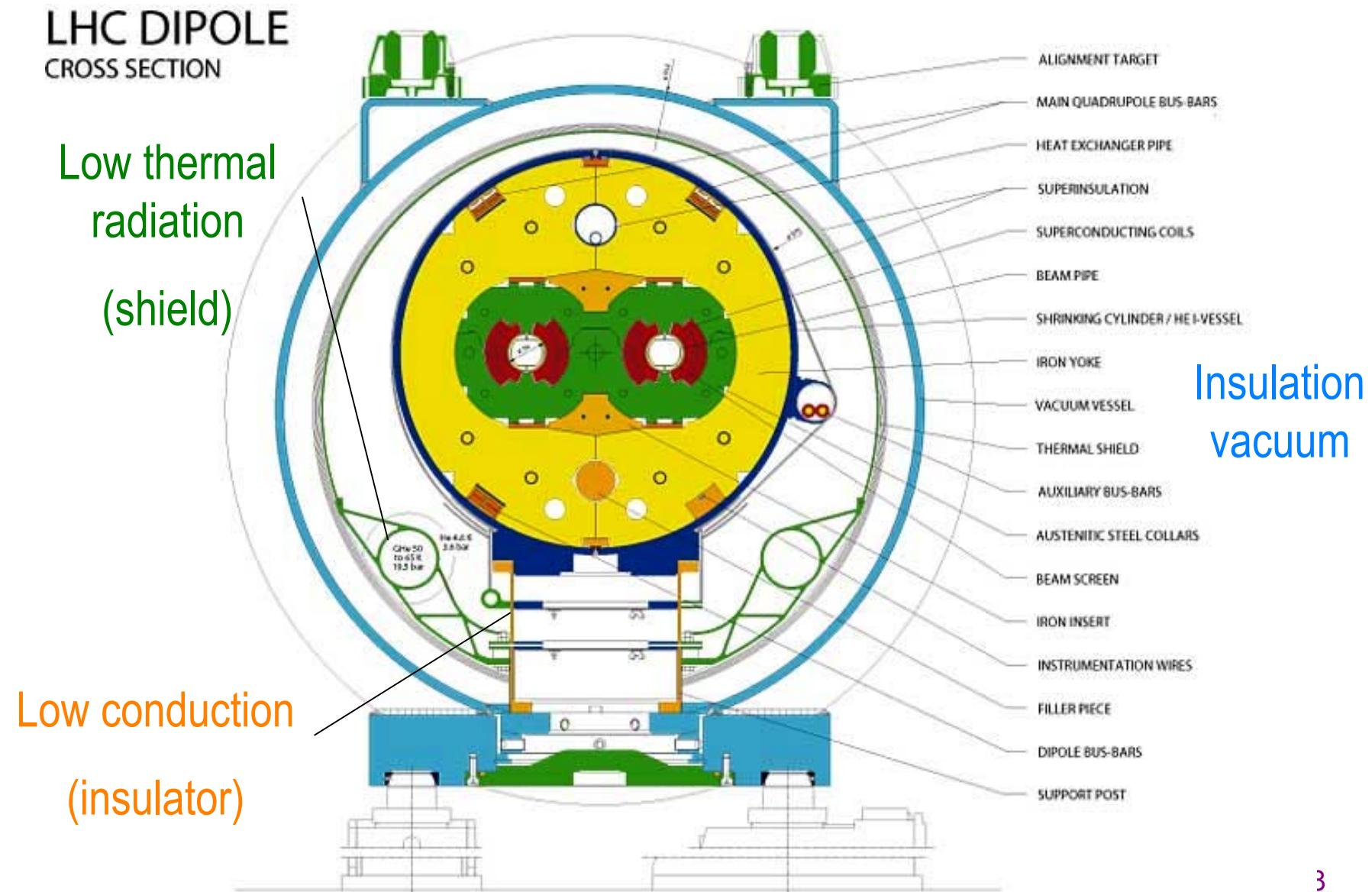


- Complex system involving three heat transfer processes
 - $Q_{MLI} = Q_{rad} + Q_{sol} + Q_{res}$
 - With n reflective layers of equal emissivity, $Q_{rad} \sim 1/(n+1)$
 - Due to parasitic contacts between layers, Q_{sol} increases with layer density
 - Q_{res} due to residual gas trapped between layers, scales as $1/n$ in molecular regime
 - Non-linear behaviour requires layer-to-layer modeling
- In practice
 - Typical data available from (abundant) literature
 - Measure performance on test samples

Typical heat fluxes at vanishingly low temperature between flat plates [W/m²]

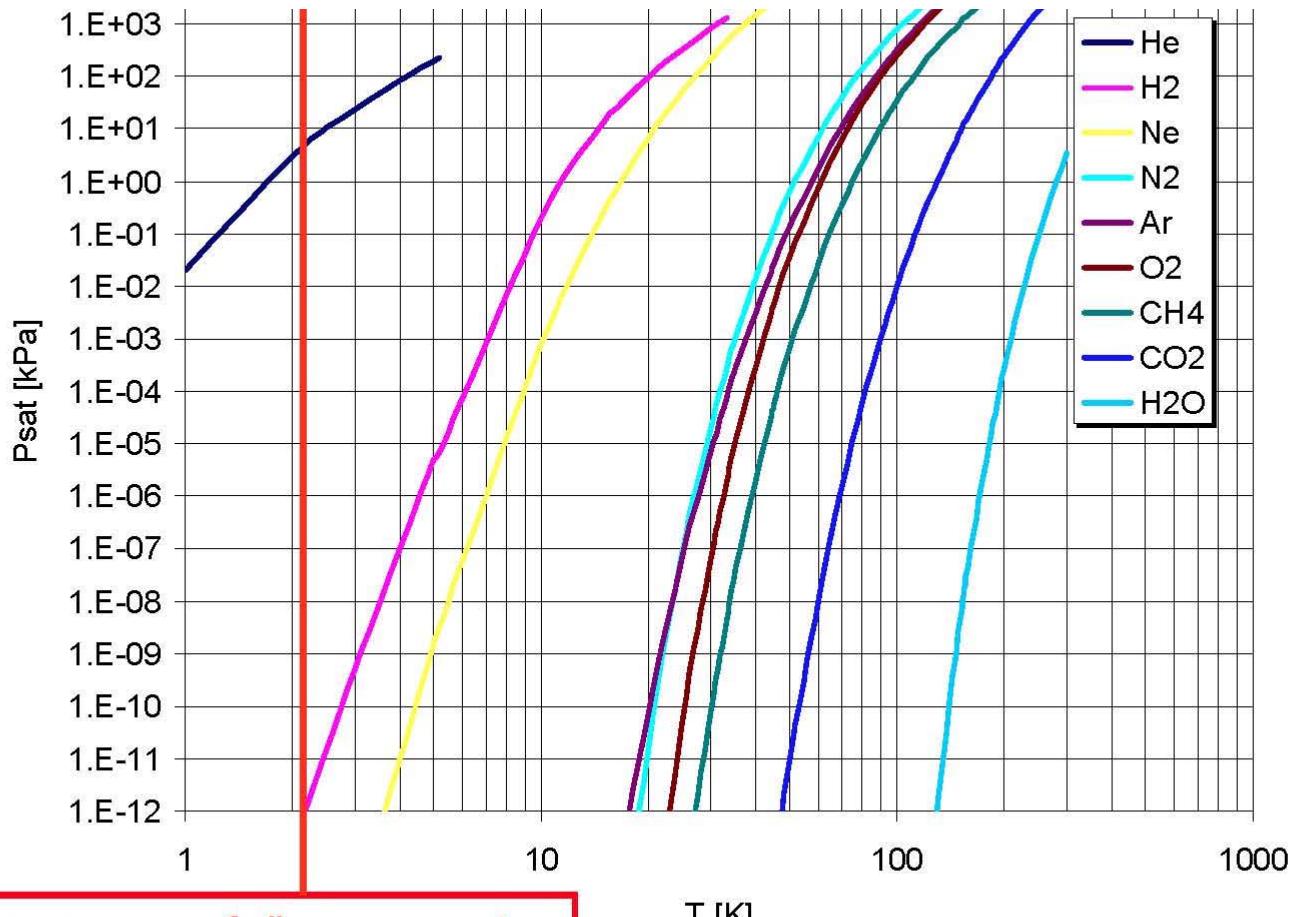
Black-body radiation from 290 K	401	Thermal shields
Black-body radiation from 80 K	2.3	
Gas conduction (100 mPa He) from 290 K	19	
Gas conduction (1 mPa He) from 290 K	0.19	Degraded vacuum
Gas conduction (100 mPa He) from 80 K	6.8	
Gas conduction (1 mPa He) from 80 K	0.07	
MLI (30 layers) from 290 K, pressure below 1 mPa	1-1.5	
MLI (10 layers) from 80 K, pressure below 1 mPa	0.05	Super isolation
MLI (10 layers) from 80 K, pressure 100 mPa	1-2	

Cross section of a LHC dipole



Cryopumping maintains good vacuum

Vapour pressure at cryogenic temperatures

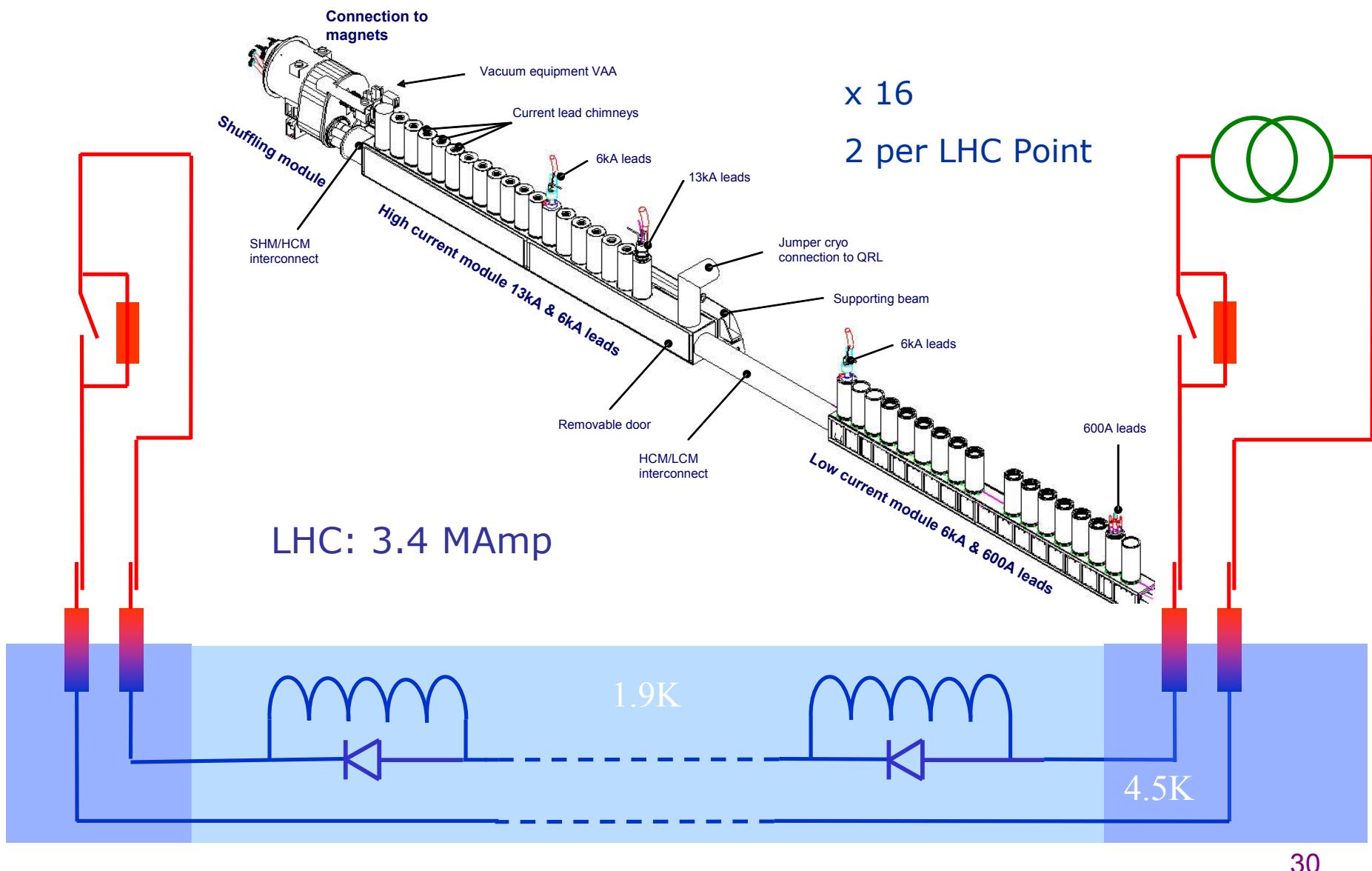


Saturation pressure of all gases except helium vanish at cryogenic temperature

T [K]

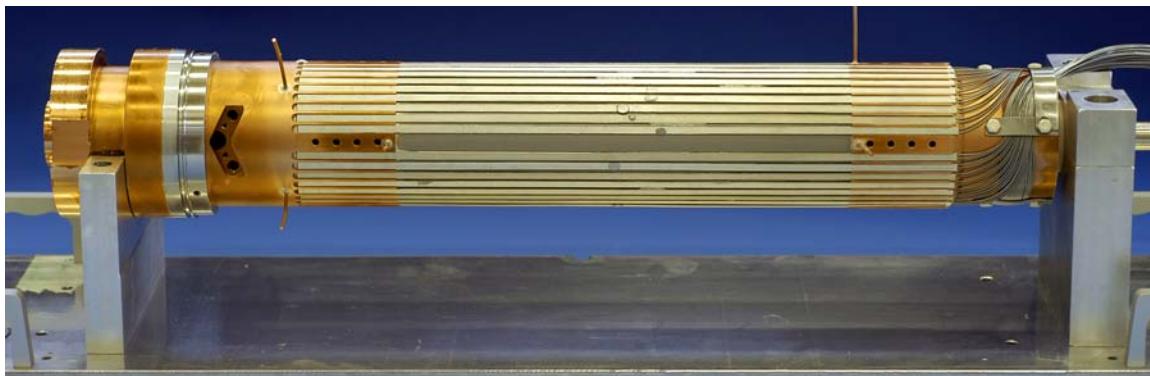
Cryopumping maintains good vacuum

Electrical Feed Box for current leads

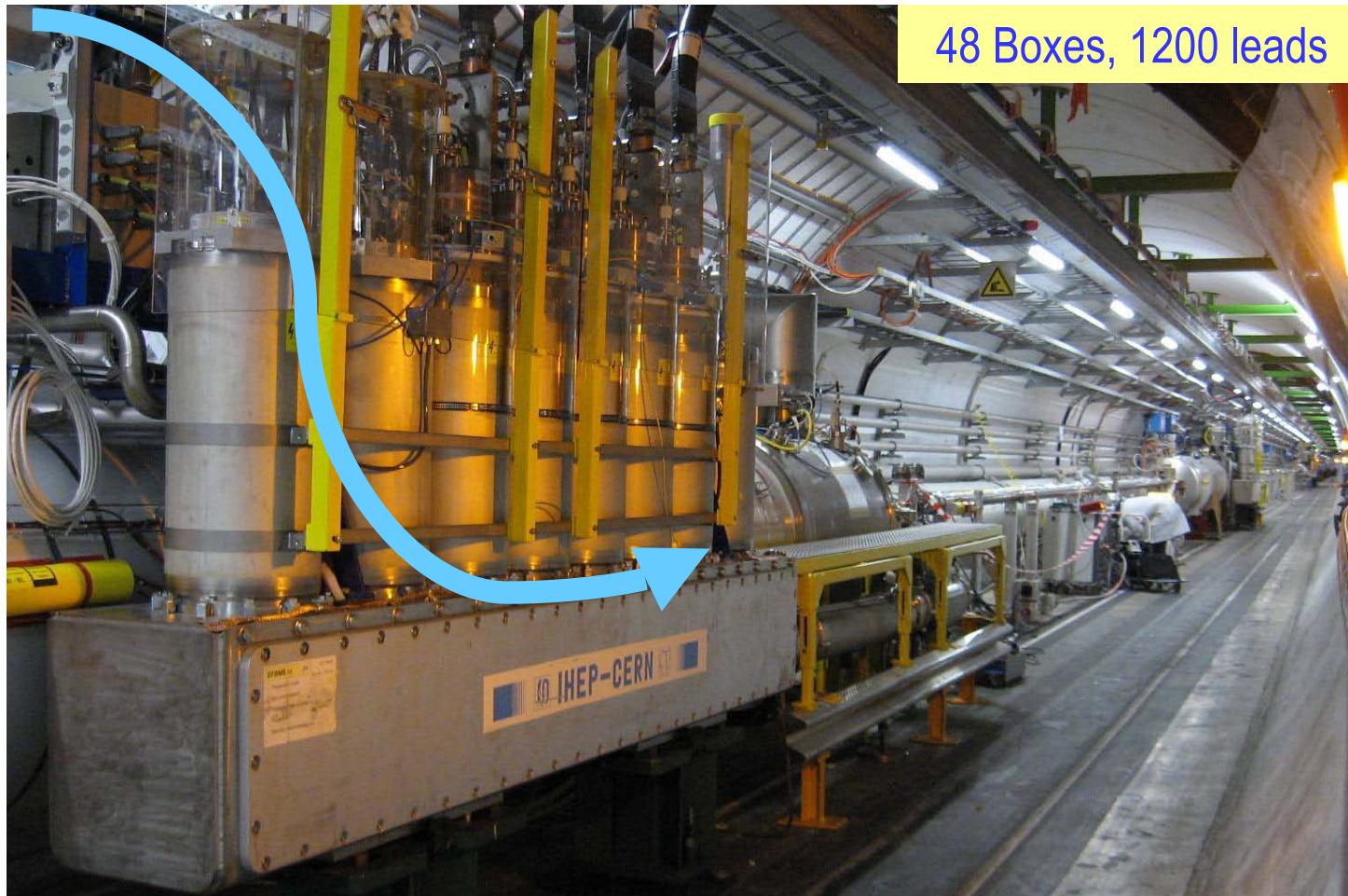


Beating the WFL law: HTS current leads

- The WFL law essentially states that good electrical conductors are also good thermal conductors
- Current leads need good electrical conductors with low thermal conductivity
- Superconductors are bad thermal conductors with zero resistivity
- Build current lead with superconductor up to temperature as high as possible, i.e. HTS



Electrical feed boxes for current leads



HTS vs. normal conducting current leads

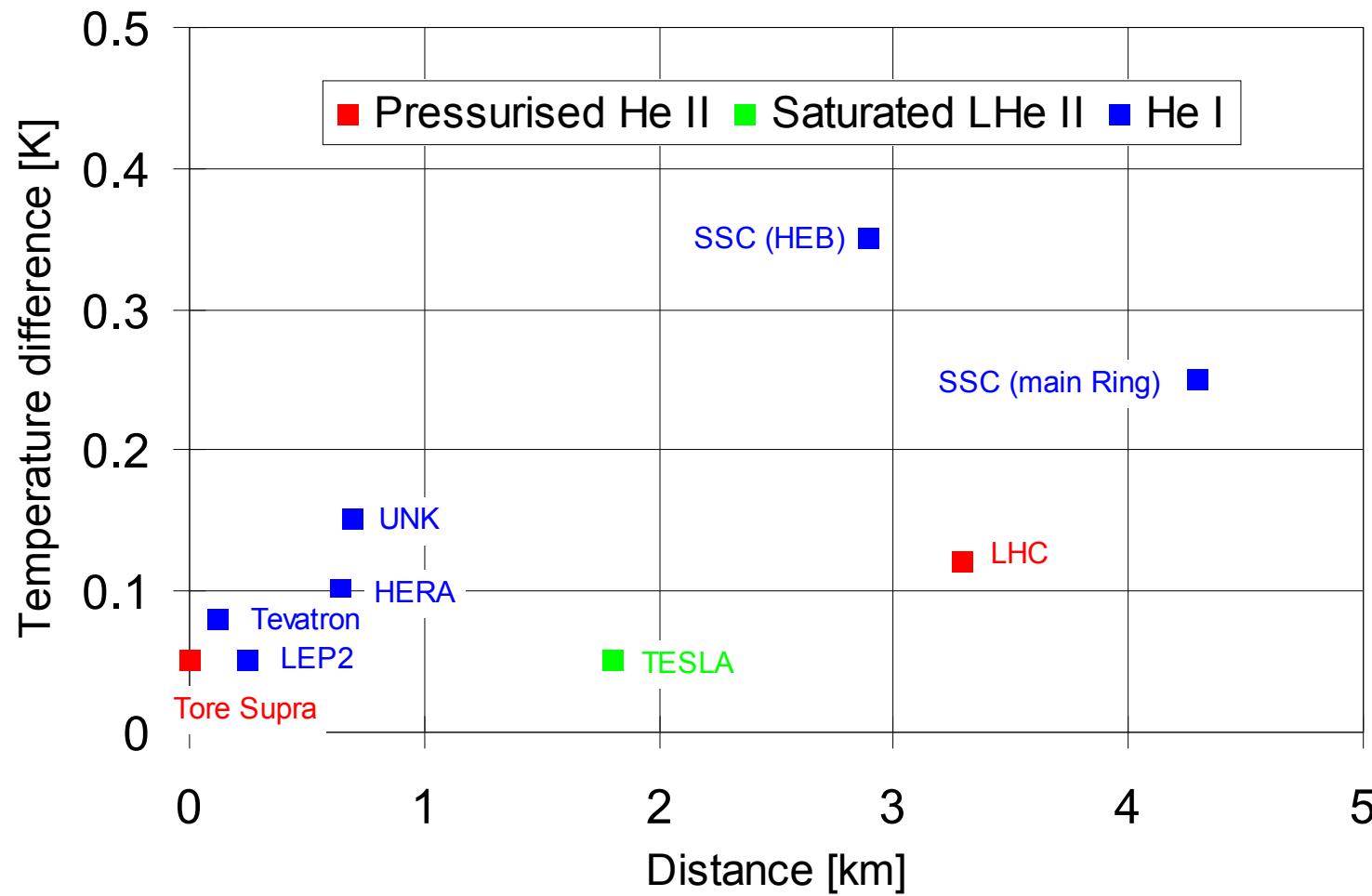
Type	Resistive	HTS (4 to 50 K) Resistive (above)
Heat into LHe	[W/kA]	1.1
Total exergy consumption	[W/kA]	430
Electrical power from grid	[W/kA]	1430

For LHC, using HTS allowed to save the equivalent of 1 large 18kW@4.5K refrigerator !

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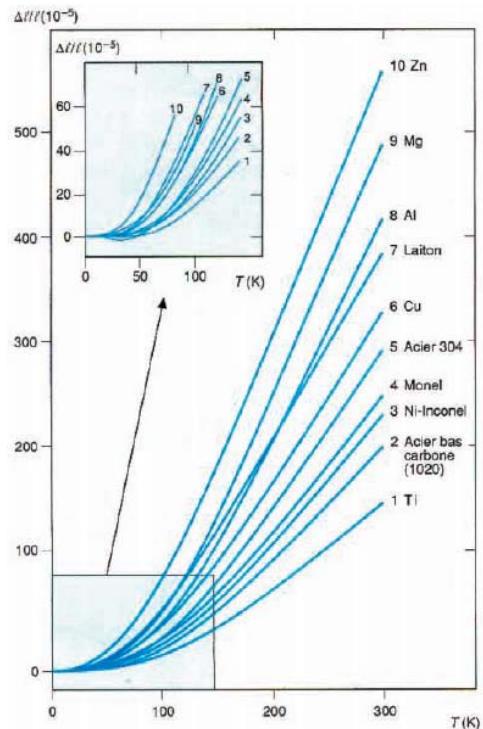
Transport of refrigeration in large distributed cryogenic systems



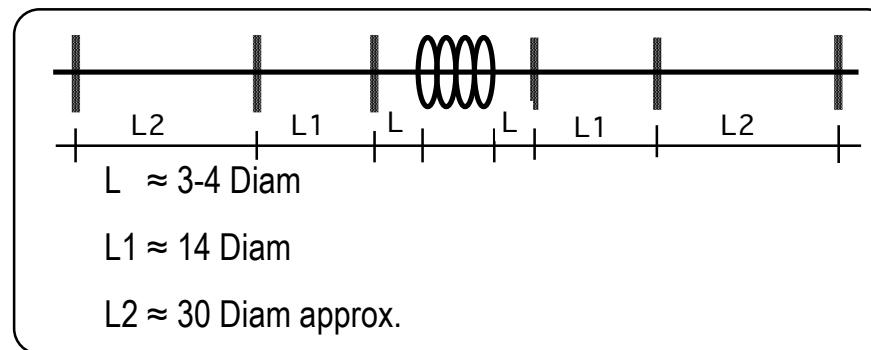
Cryogenic distribution scheme: design issues

- Monophase vs. two-phase
 - temperature control
 - hydrostatic head & flow instabilities
- Pumps vs. no pumps
 - efficiency & cost
 - reliability & safety
- LN₂
 - cooldown and/or normal operation
 - capital & operating costs of additional fluid
 - safety in underground areas (ODH)
- Lumped vs. distributed cryoplants
- Separate cryoline vs. integrated piping
- Number of active components (valves, actuators)
- Redundancy of configuration

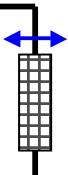
Thermal contraction for cryo lines



3 mm / m of thermal contraction
=> Compensation required !

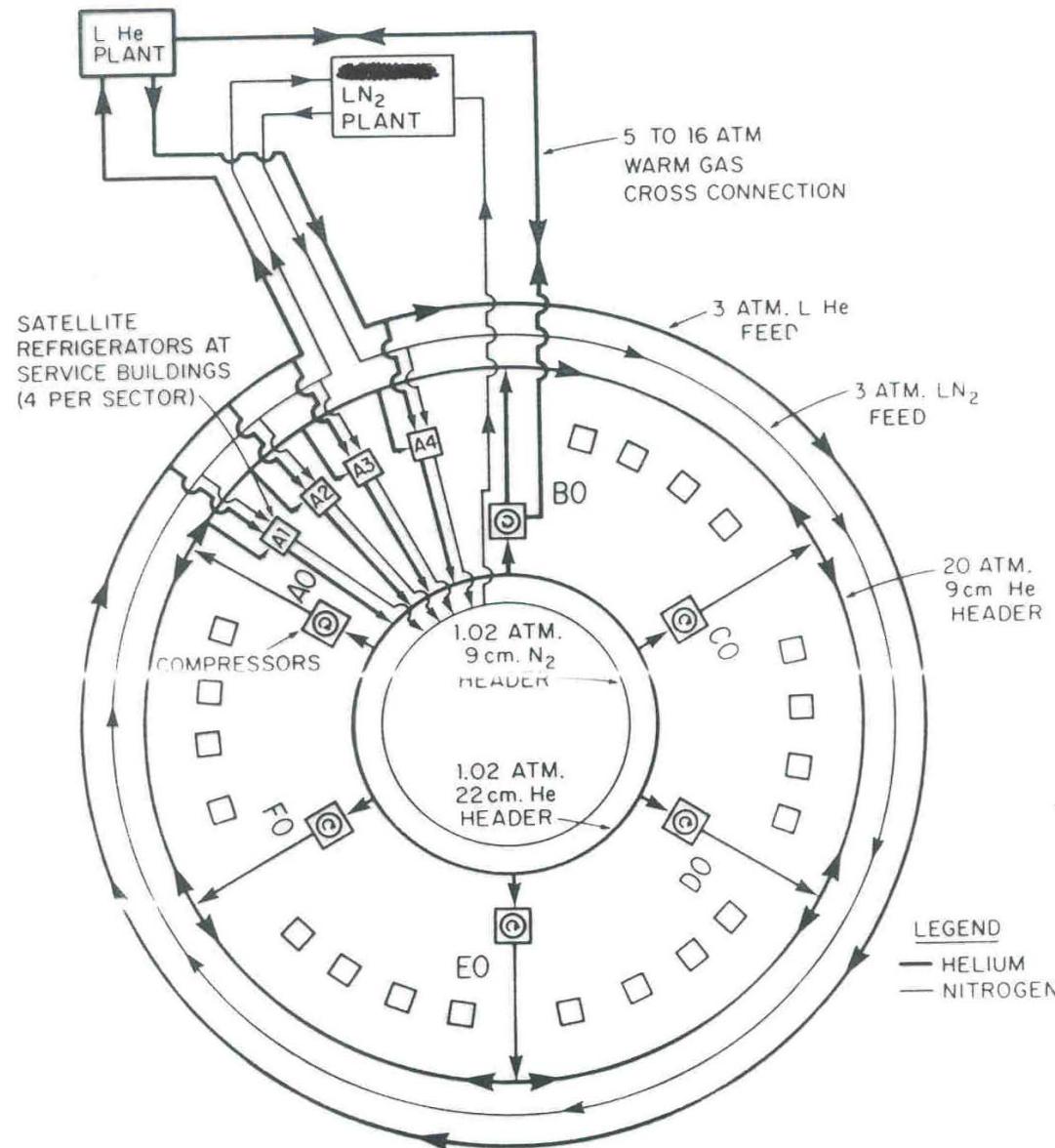


Thermal compensation



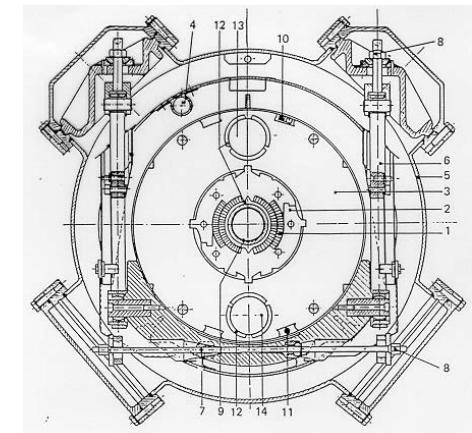
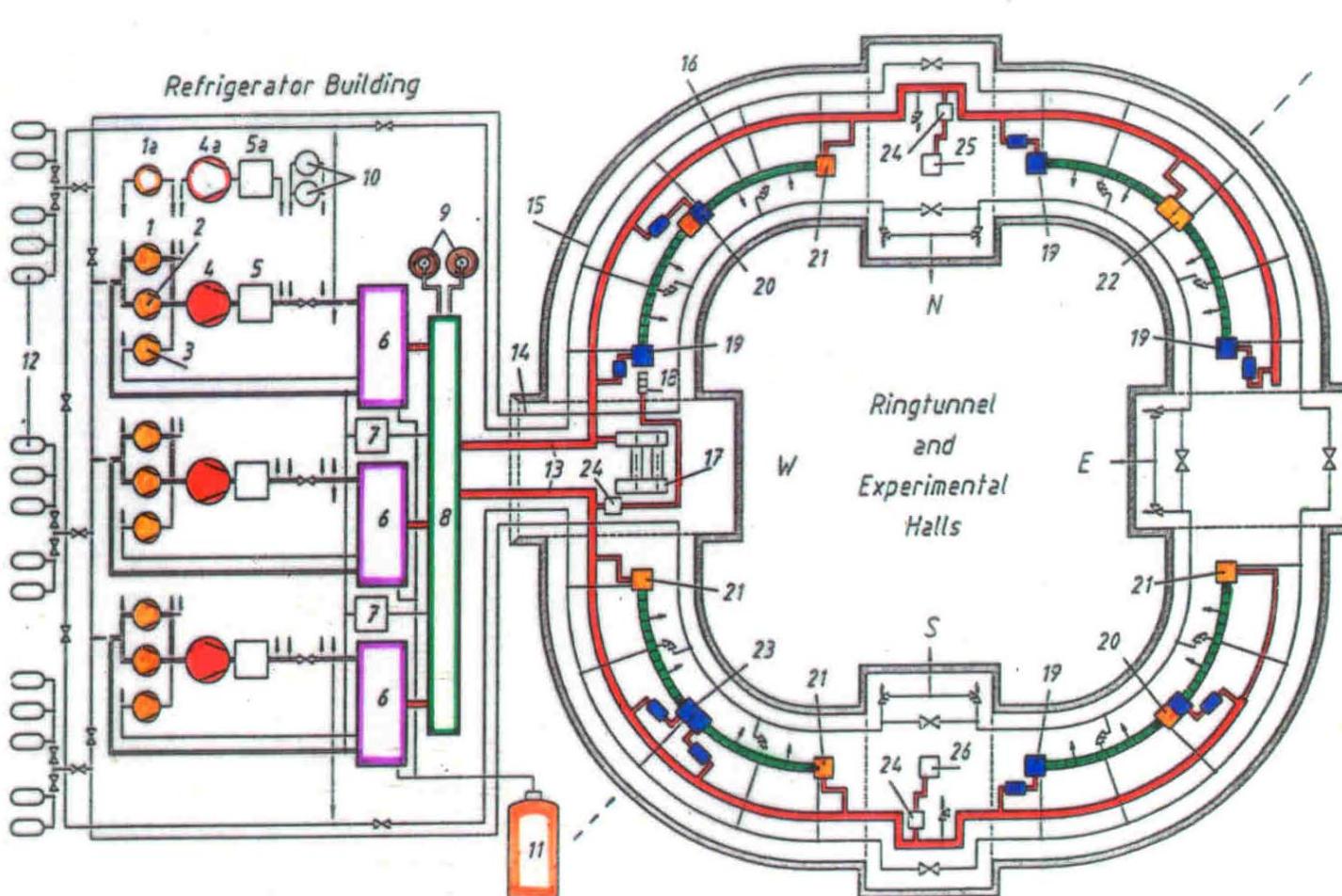
This is **THE** delicate part in the design of a cryogenic line, as thermal performance can only be considered once the line withstand mechanical forces !!!

Tevatron distribution scheme



Central helium liquefier,
separate ring cryoline
and satellite
refrigerators

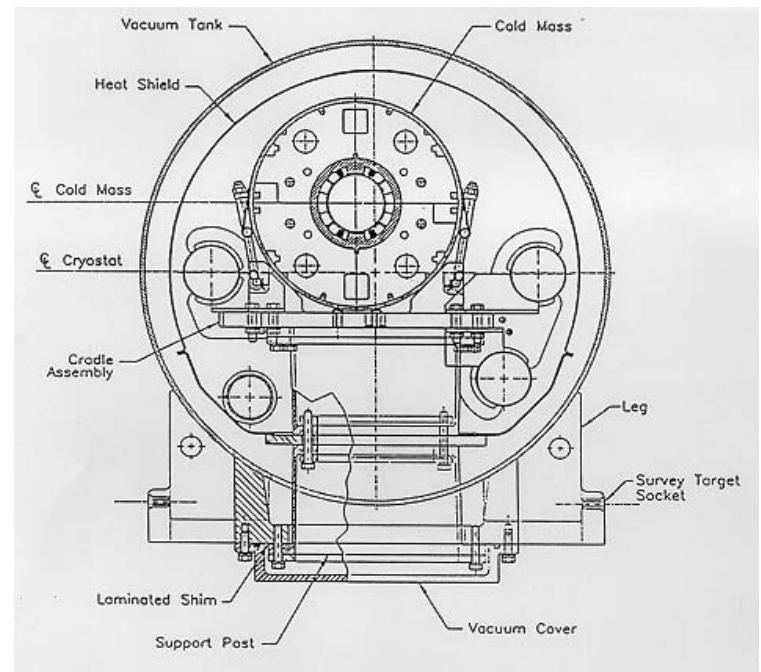
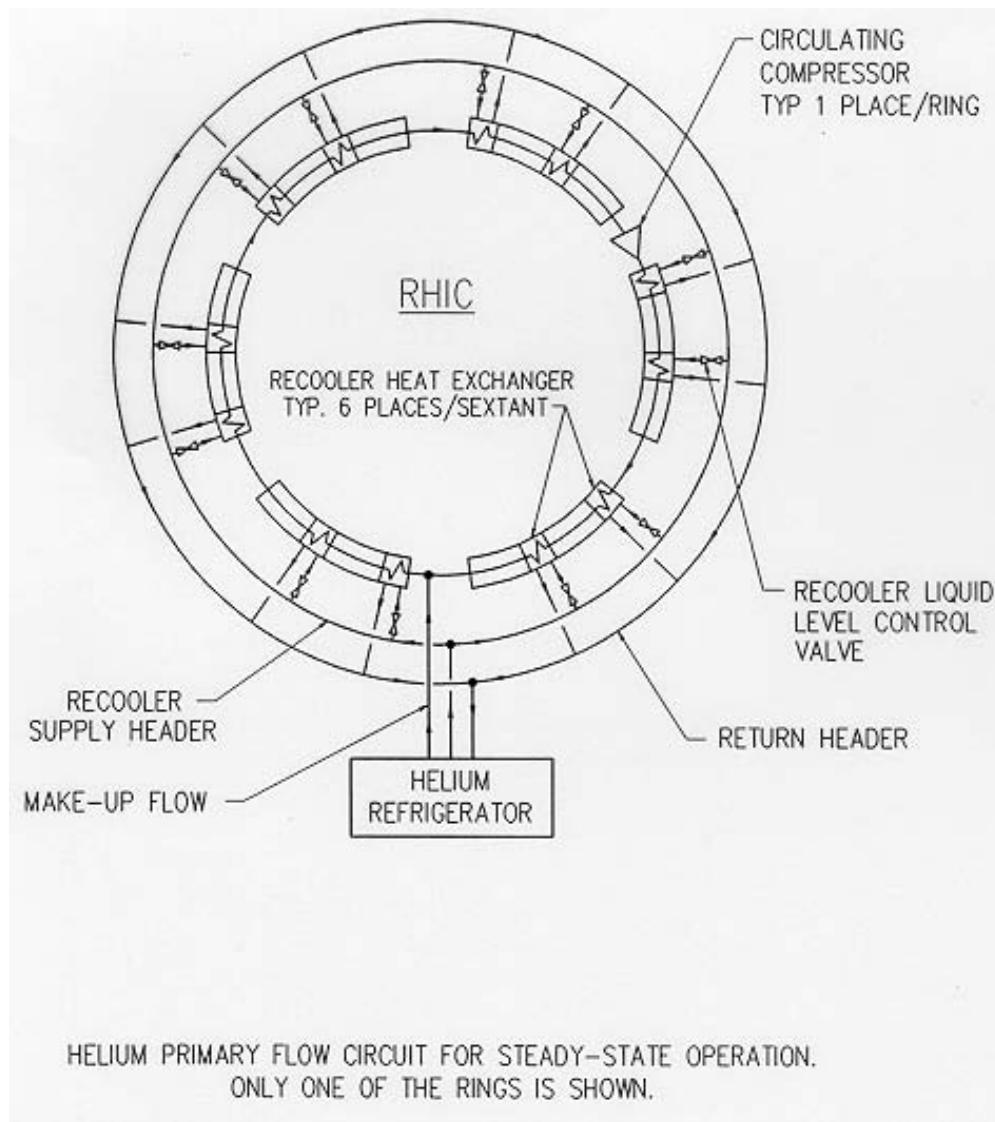
HERA distribution scheme



Central
cryoplant and
separate ring
cryoline

Refrigeration 4.3 K	6775 W	total mass flow	0.871 kg/s
Refrigeration 40/80 K	20000 W	Primary power	2845 kW
Current lead flow	20.5×10^{-3} kg/s	Specif. power consumption	281 W (300 K)/W (4.3 K)

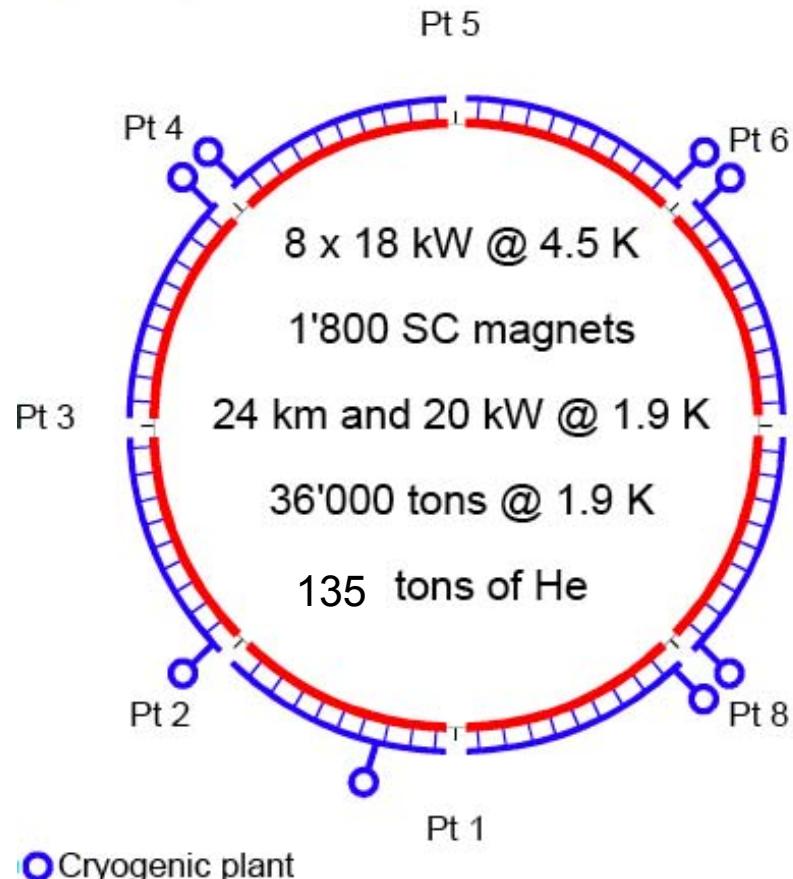
RHIC distribution scheme



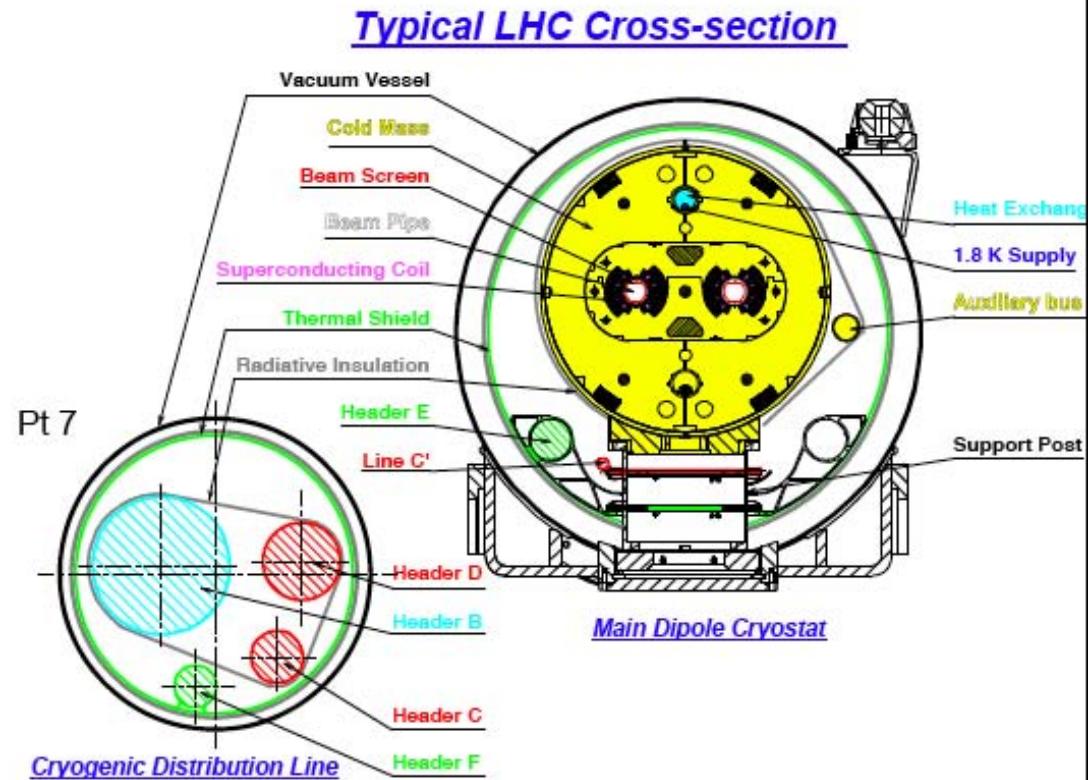
Central cryoplant and
piping integrated in
magnet cryostat



LHC distribution scheme



○ Cryogenic plant



Cryoplants at five points,
separate ring cryoline,
107 m long strings

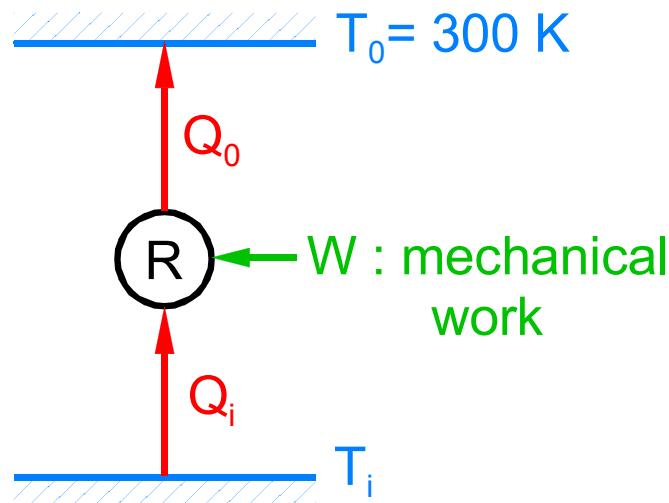
Superconducting Linac (Tesla_based)



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Thermodynamics of cryogenic refrigeration



First principle [Joule]

$$Q_0 = Q_i + W$$

Second principle [Clausius]

$$\frac{Q_0}{T_0} \geq \frac{Q_i}{T_i}$$

(= for reversible process)

Hence, $W \geq T_0 \cdot \frac{Q_i}{T_i} - Q_i$ which can be written in three different ways:

① $W \geq T_0 \cdot \ddot{A}S_i - Q_i$ introducing **entropy S** as

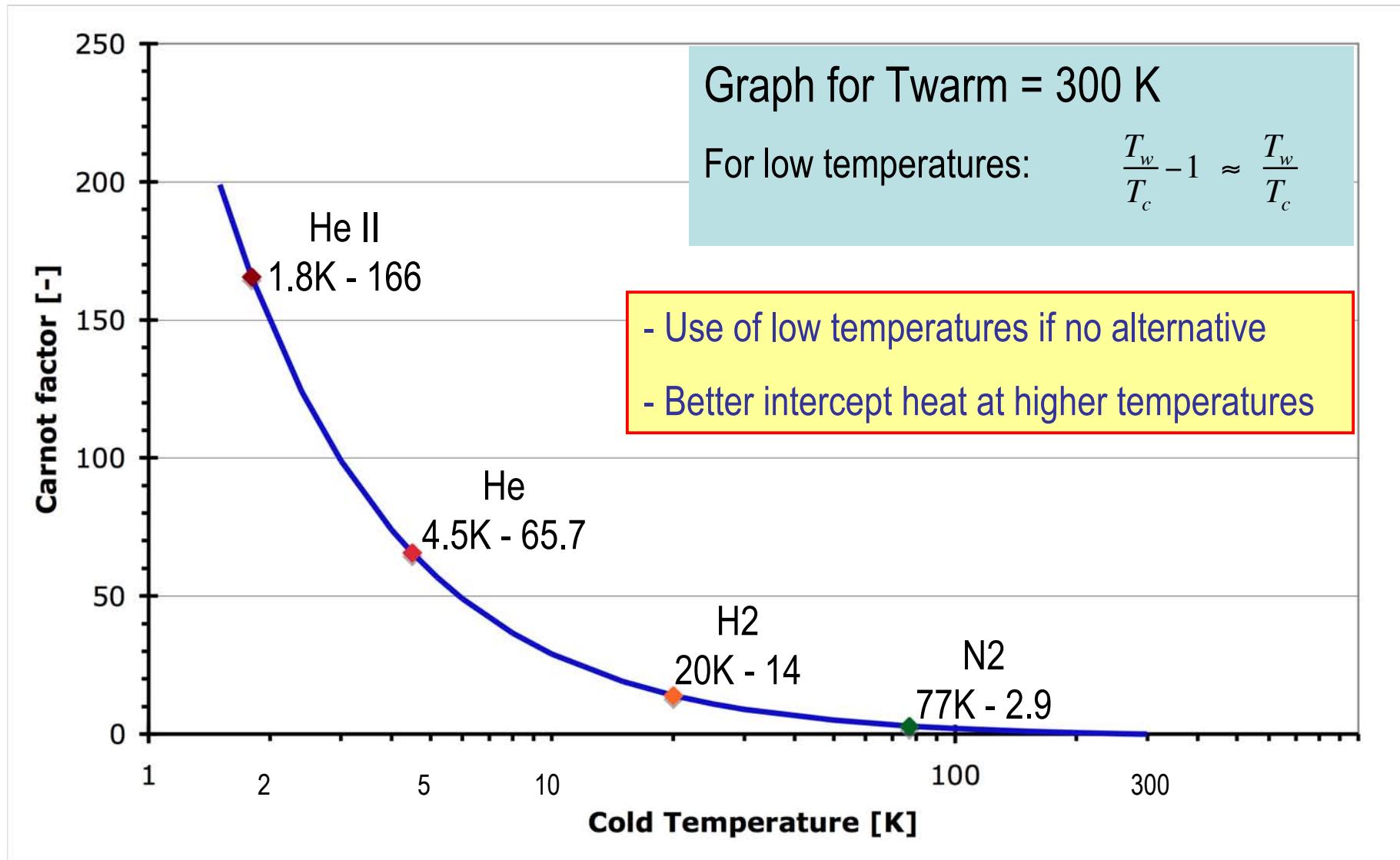
$$\ddot{A}S_i = \frac{Q_i}{T_i}$$

② $W \geq Q_i \left[\left(\frac{T_0}{T_i} - 1 \right) \right]$ **Carnot factor**

③ $W \geq \ddot{A}E_i$ introducing **exergy E** as

$$\ddot{A}E_i = Q_i \cdot \left(\frac{T_0}{T_i} - 1 \right)$$

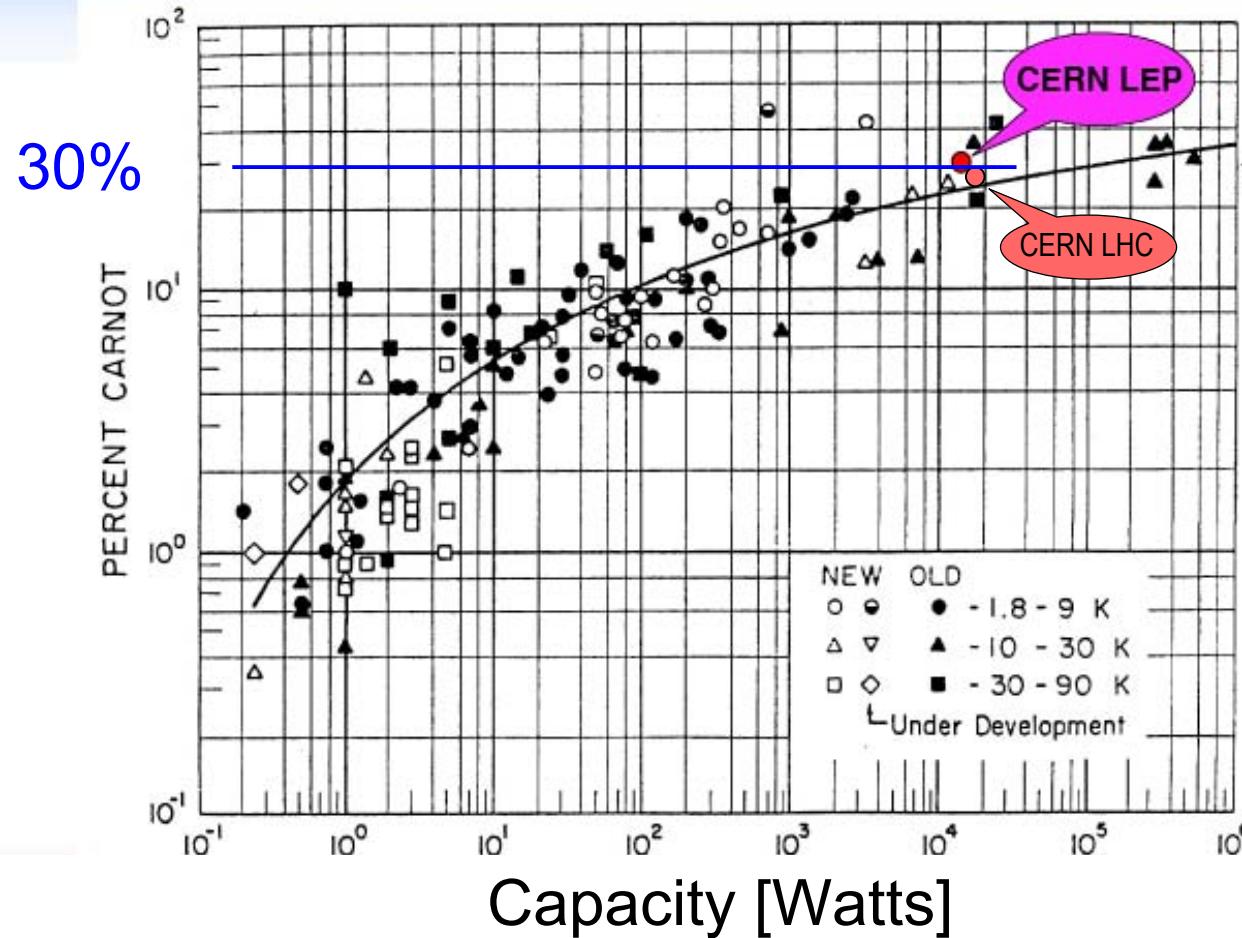
The Carnot Factor



Helium refrigerators

Power Input \approx Power@cold x Carnot / %w.r.tCarnot

LE DIAGRAMME DE STROBRIDGE



The efficiency w.r.t Carnot does not depend on the temperature, but rather on the size

The largest possible, the best !

Minimum refrigeration work

Consider the extraction of 1 W at 4.5 K, rejected at 300 K
The minimum refrigeration work (equation 2) is:

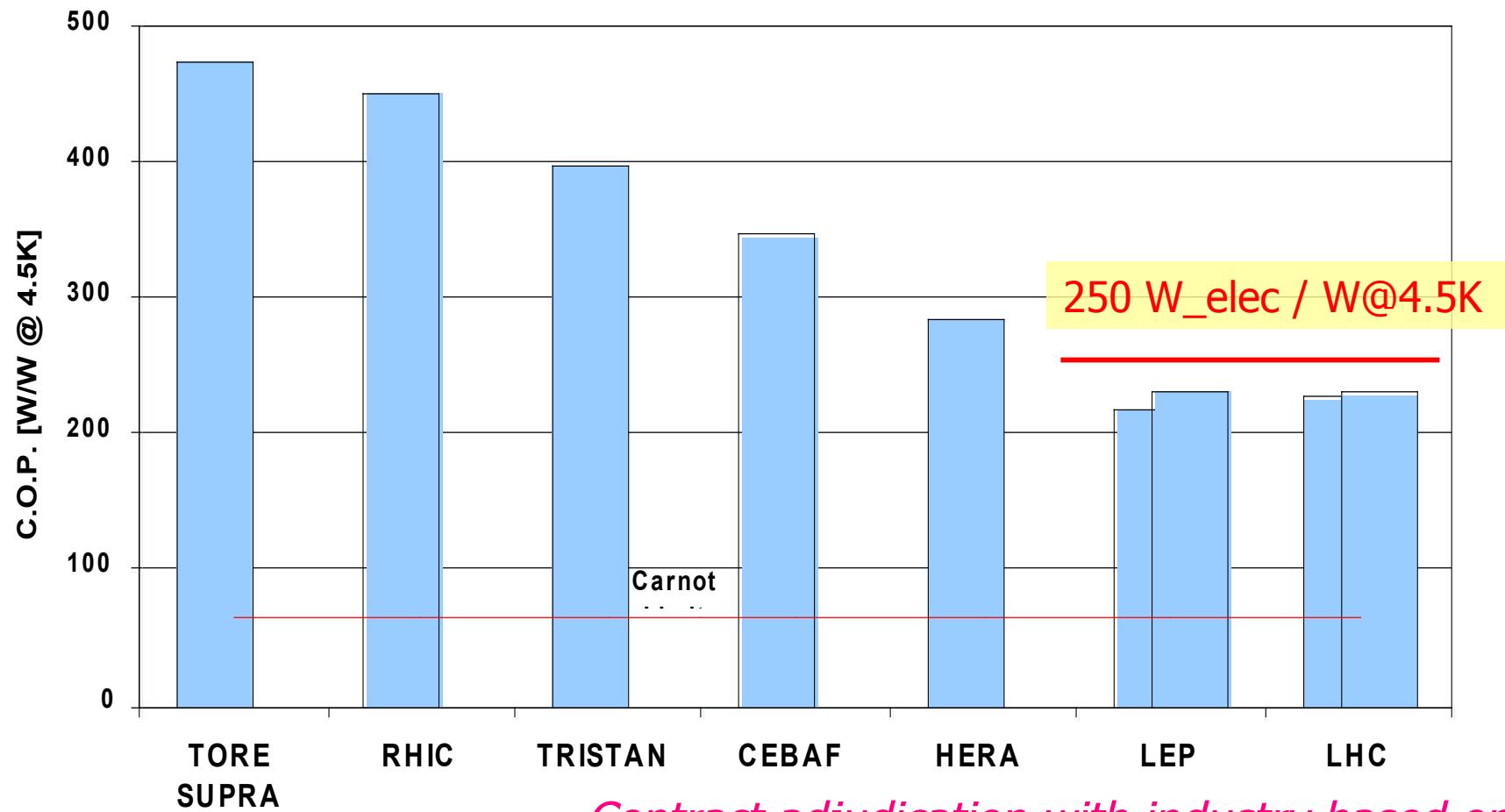
$$W_{\min} = Q_i \cdot \left(\frac{T_0}{T_i} - 1 \right) = 1 \cdot \left(\frac{300}{4.5} - 1 \right) = 65.7 \text{ W}$$

In practice, the most efficient helium refrigerators have an efficiency of about 30% w.r. to the Carnot limit.

$$\Rightarrow W_{\text{real}} = \frac{W_{\min}}{\zeta} = \frac{65.7}{0.3} = 220 \text{ W}$$

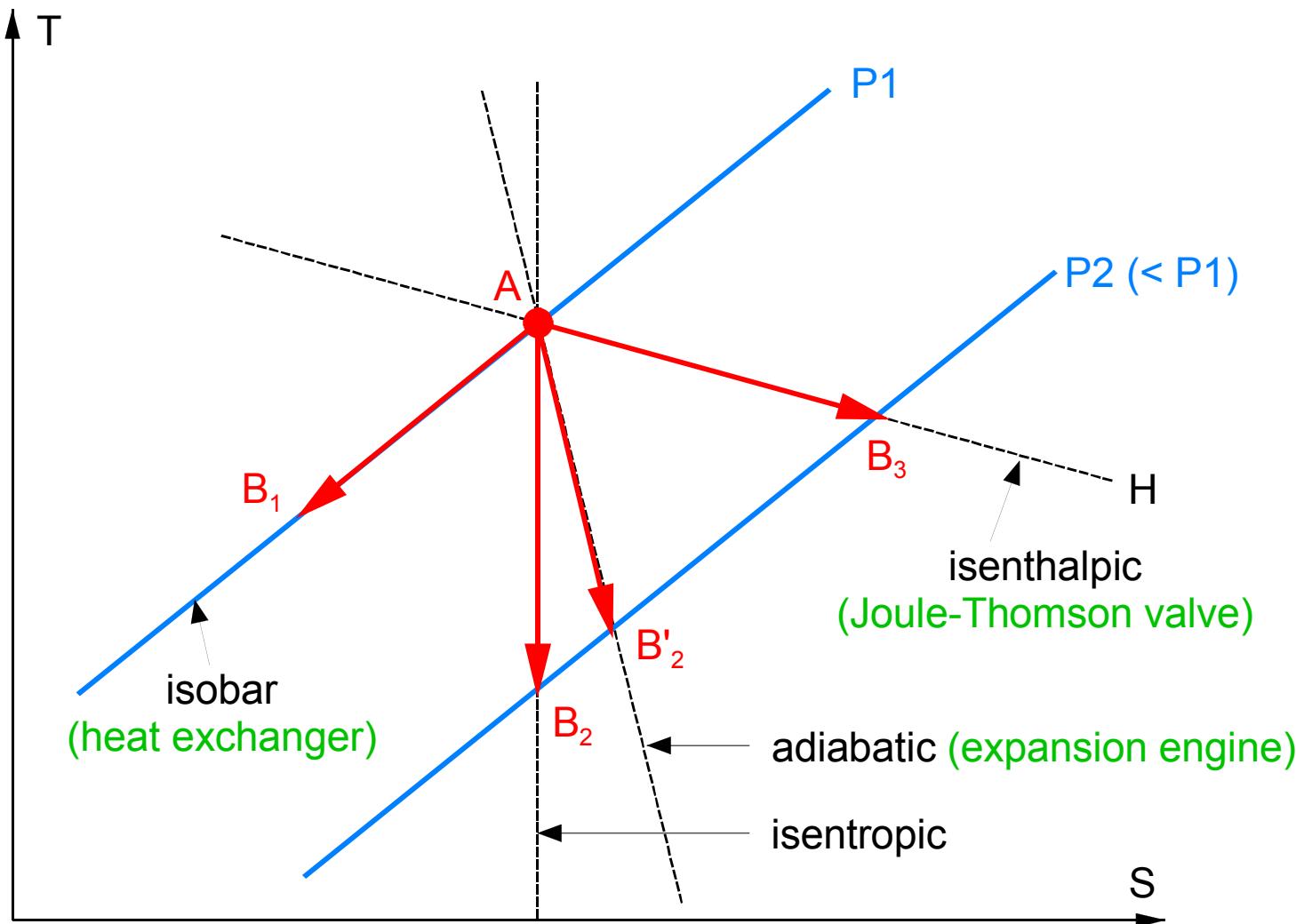
C.O.P. of large cryogenic helium refrigerators

Time (left to right) is not the only factor for improvement

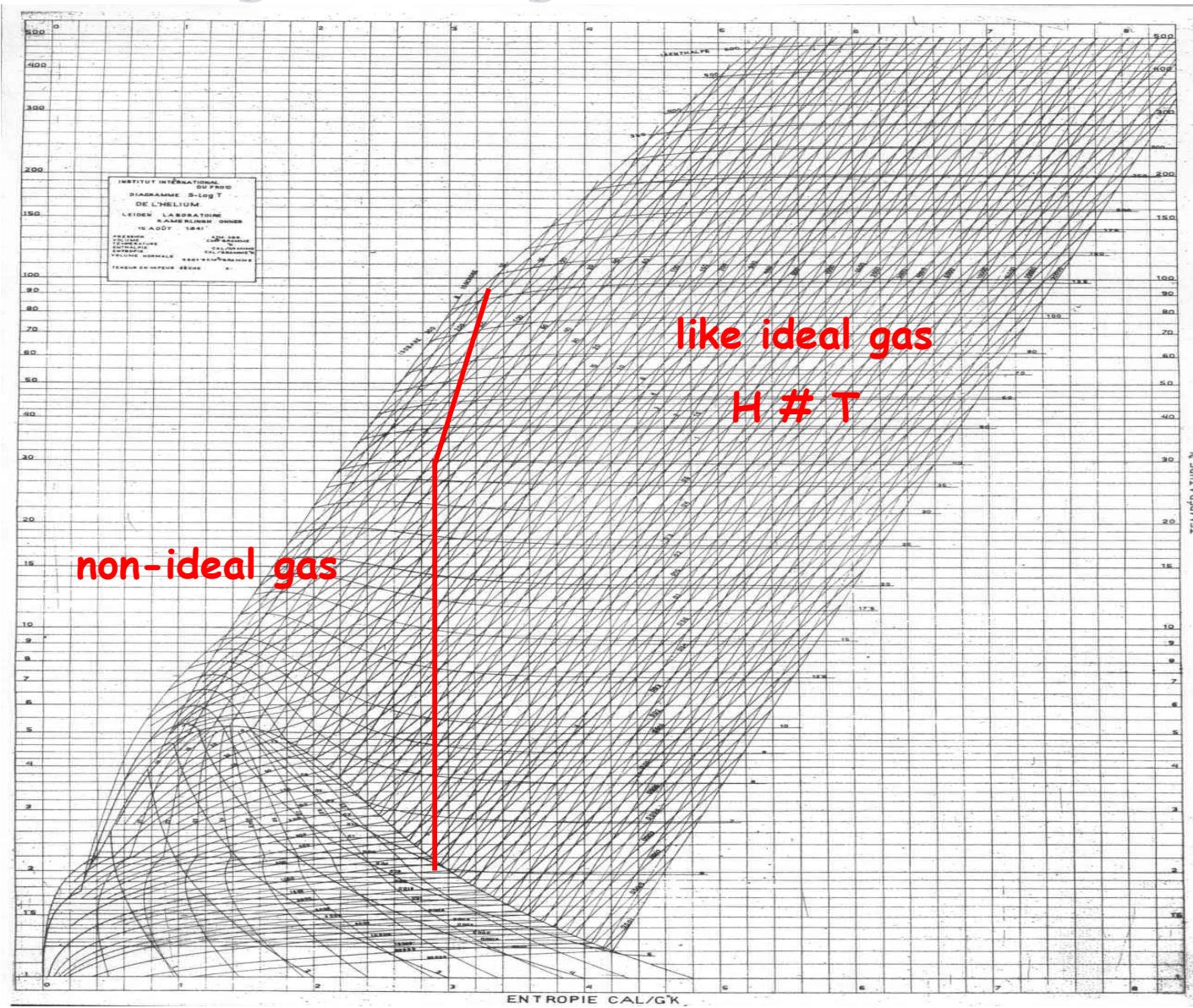


*Contract adjudication with industry based on
Capital+Operation(10yrs) costs*

Elementary cooling processes on T-S diagram



Log T-s Diagram for Helium

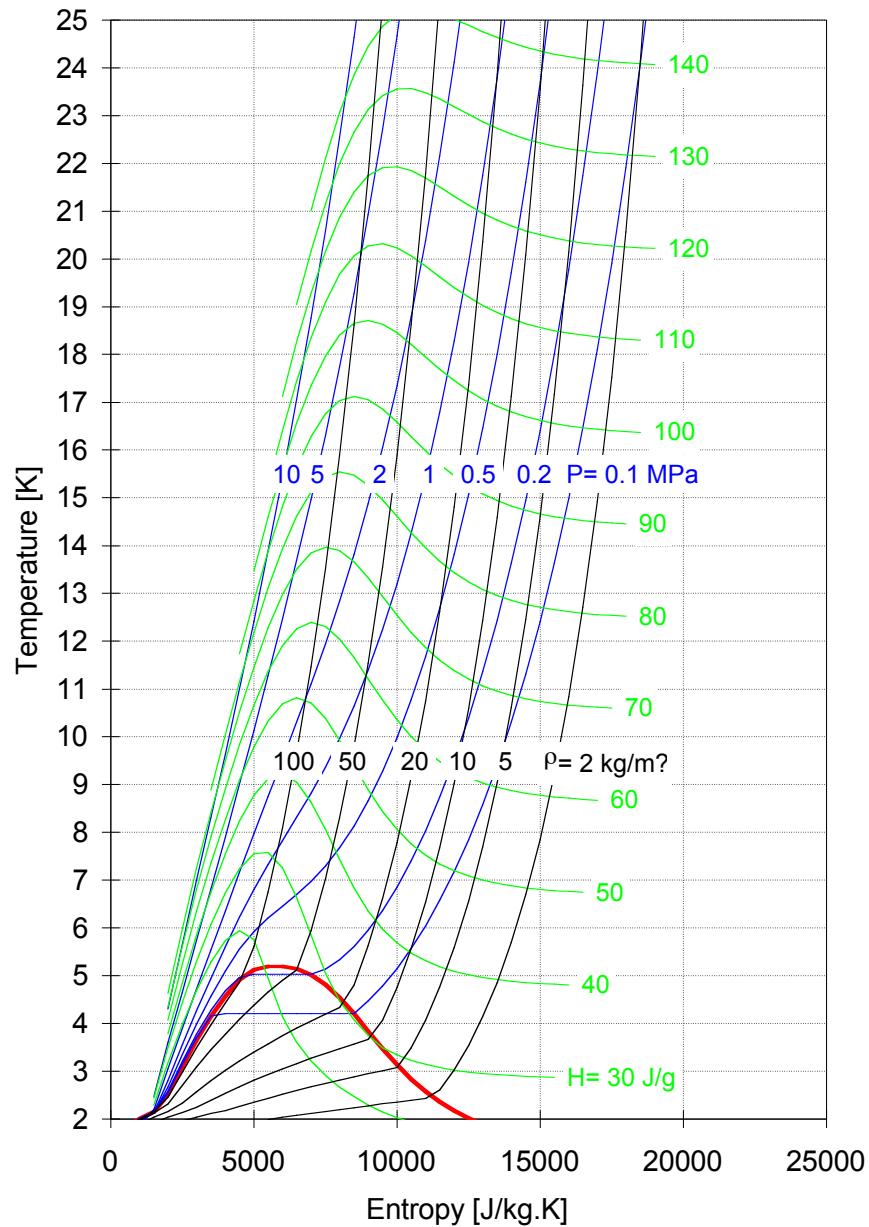


Maximum Joule-Thomson inversion temperatures

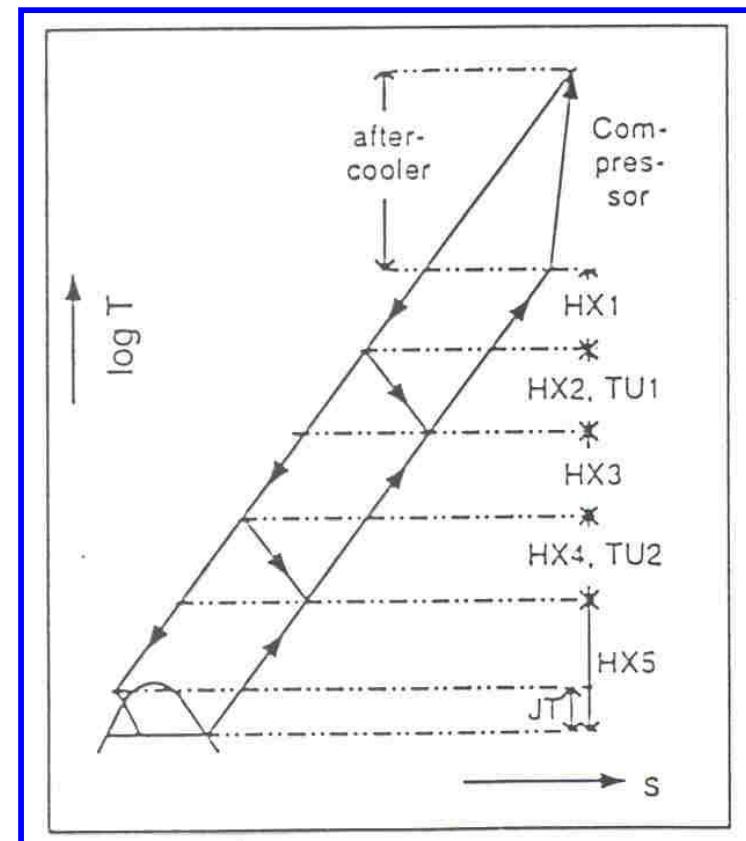
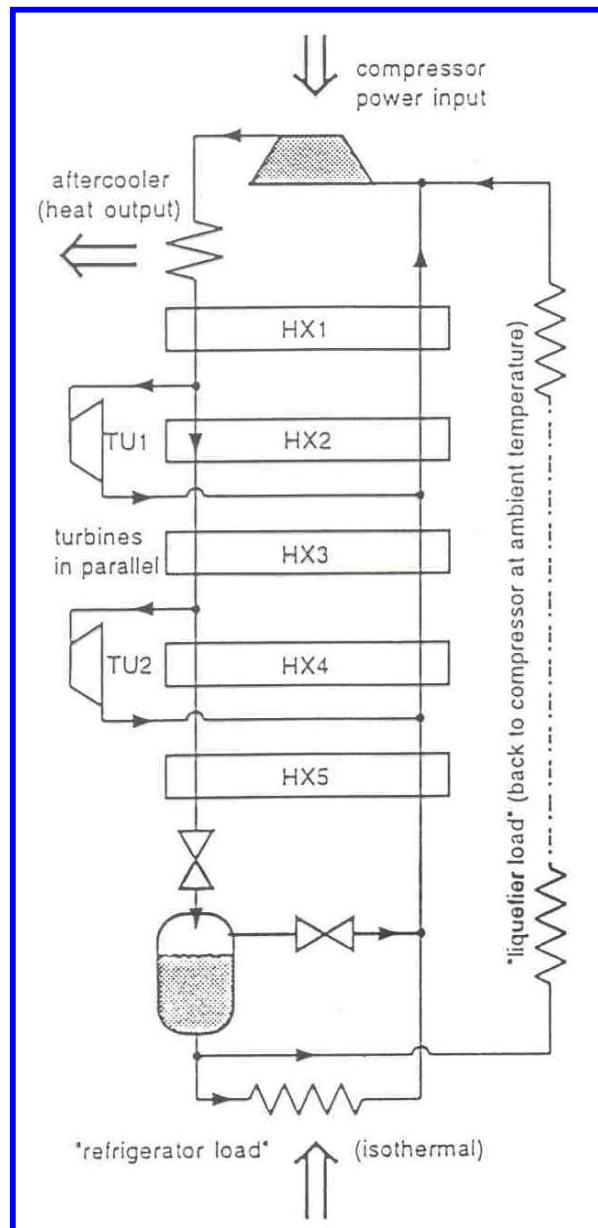
Cryogen	Maximum inversion temperature [K]
Helium	43
Hydrogen	202
Neon	260
Air	603
Nitrogen	623
Oxygen	761

While air can be cooled down and liquefied by JT expansion from room temperature, helium and hydrogen need precooling down to below inversion temperature by heat exchange or work-extracting expansion (e.g. in turbines)

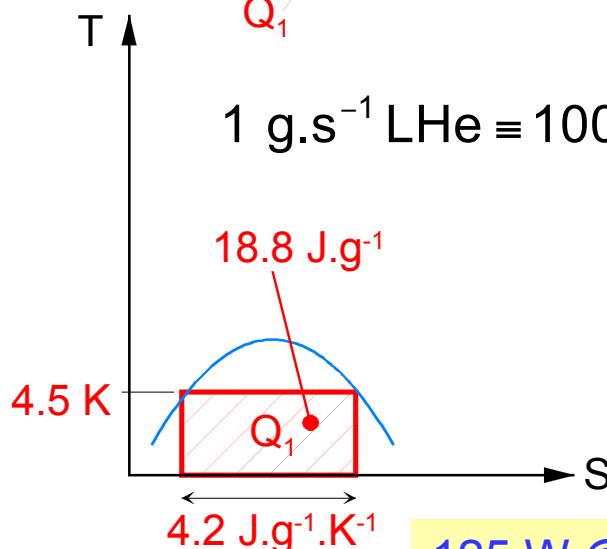
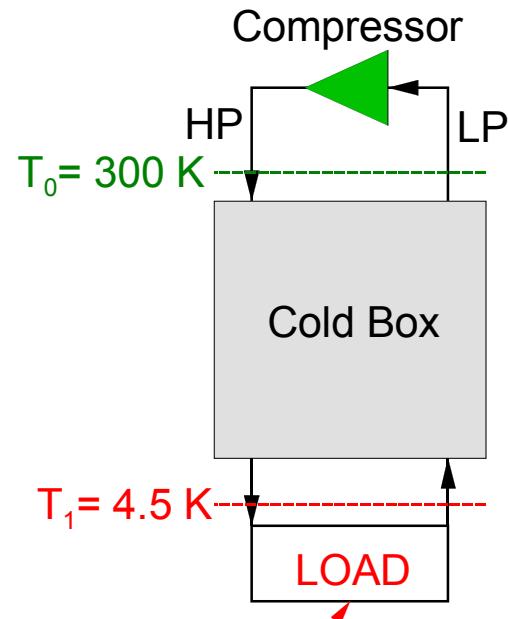
T-S diagram for helium (non-ideal part)



Two-stage Claude cycle

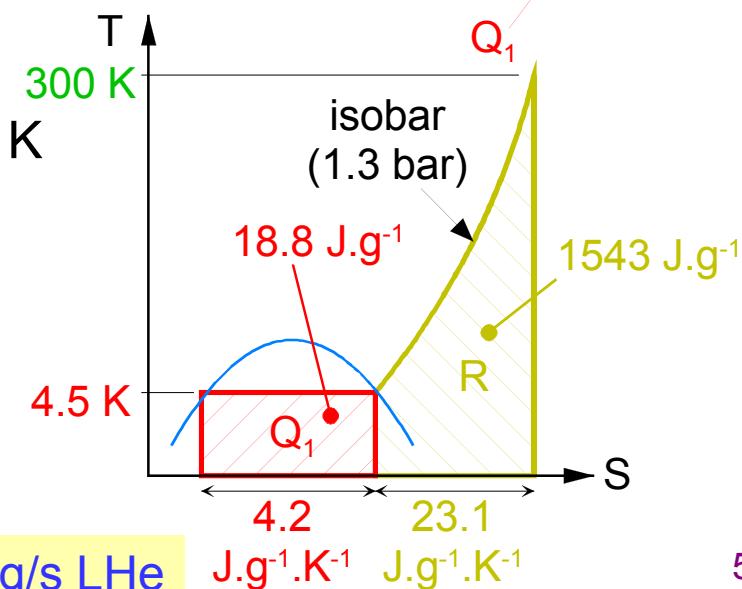
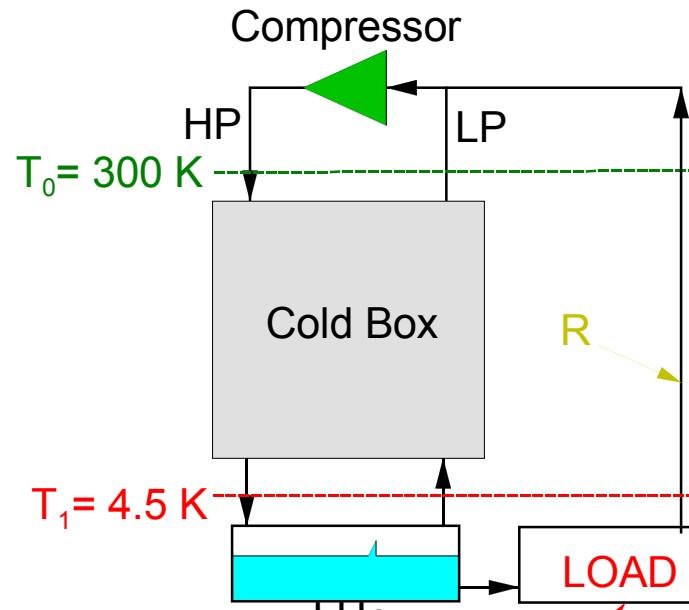


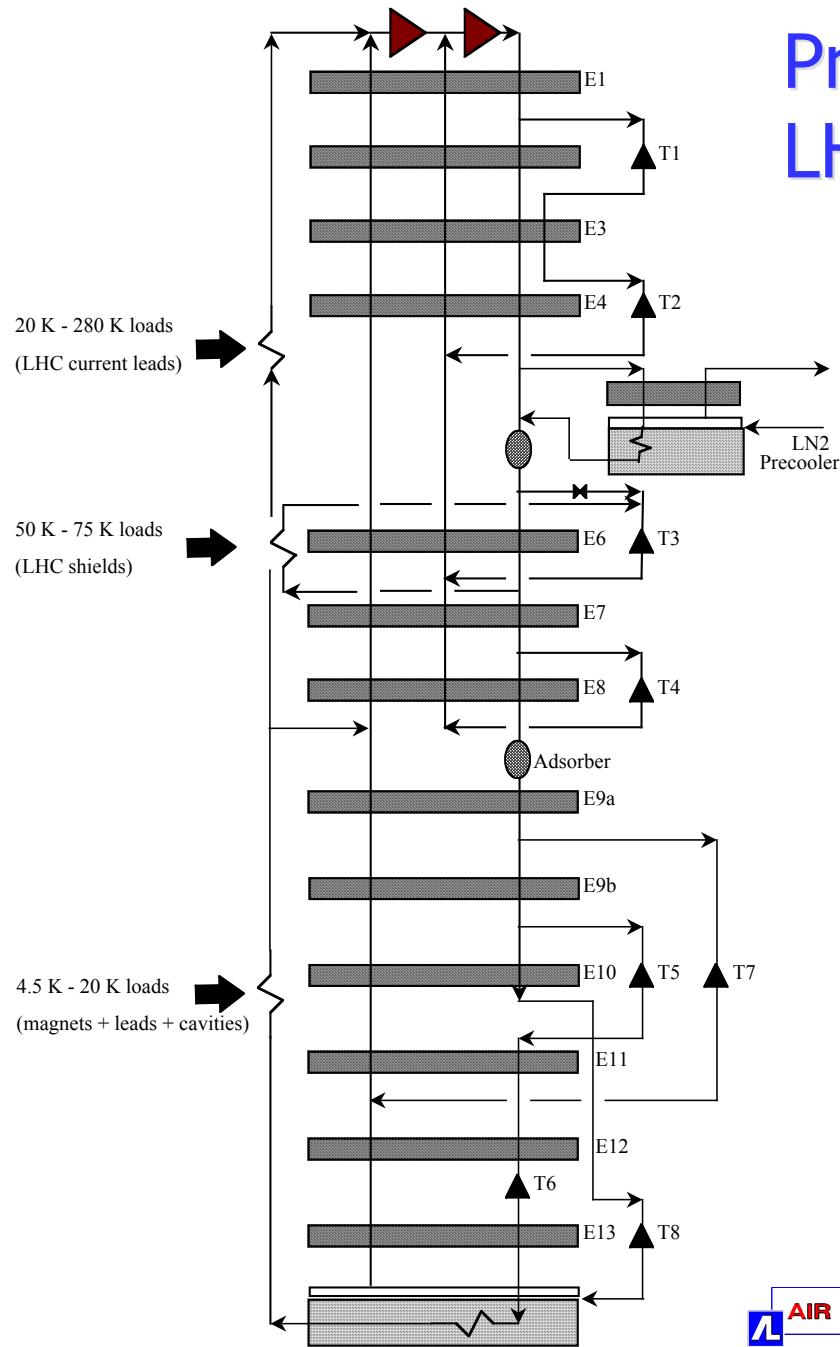
Refrigerator



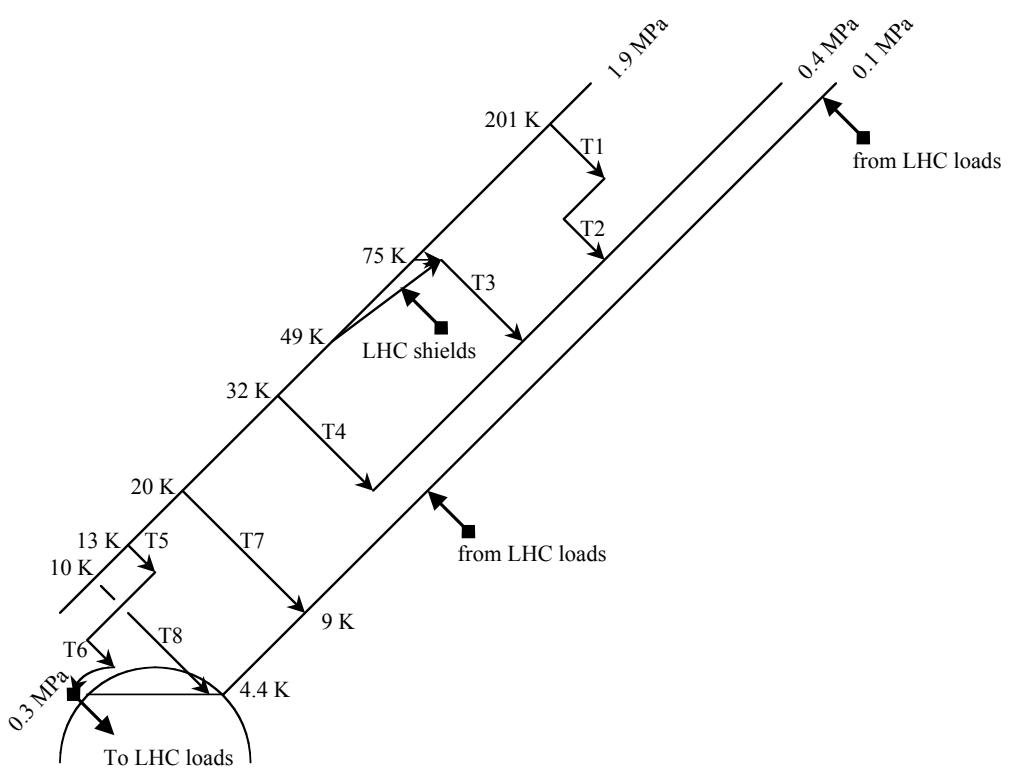
$$125 \text{ W} @ 4.5 \text{ K} \approx 1 \text{ g/s LHe}$$

Liquefier



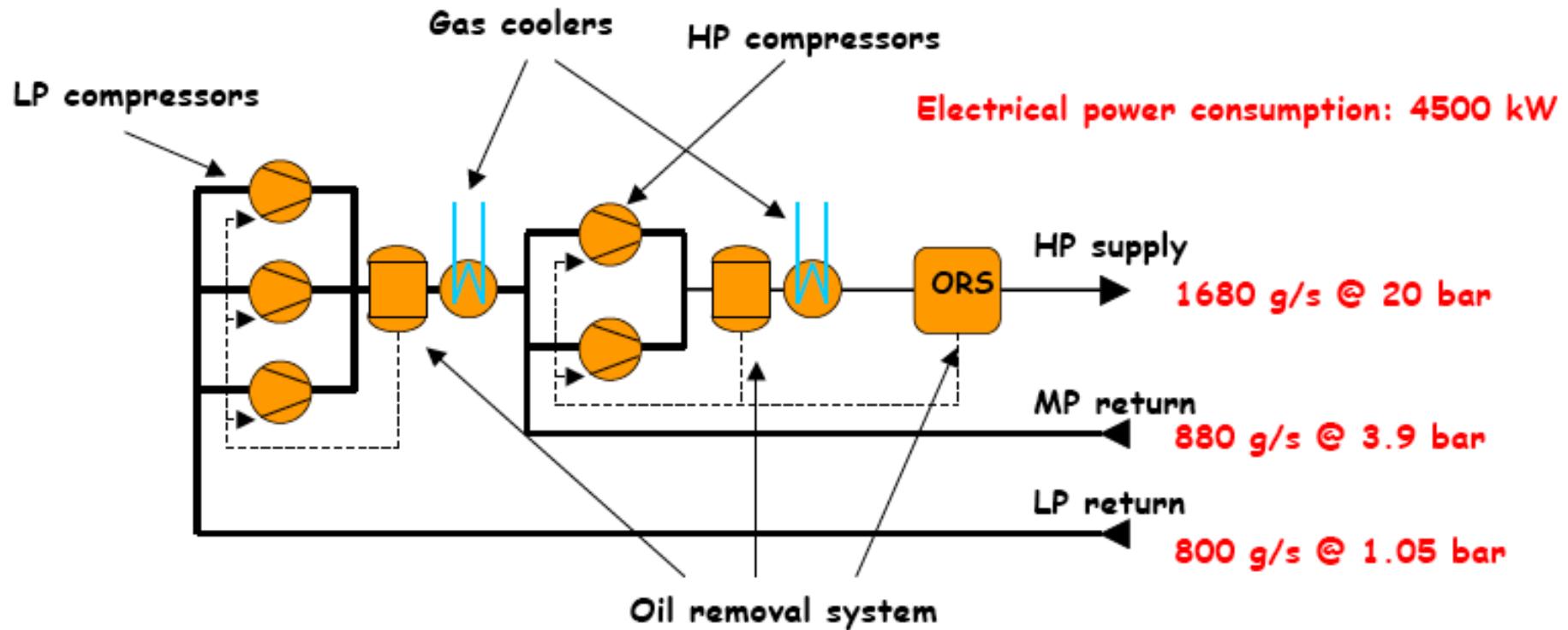


Process cycle & T-S diagram of LHC 18 kW @ 4.5 K cryoplant



Process diagram, LHC compressors 18 kW @ 4.5 K

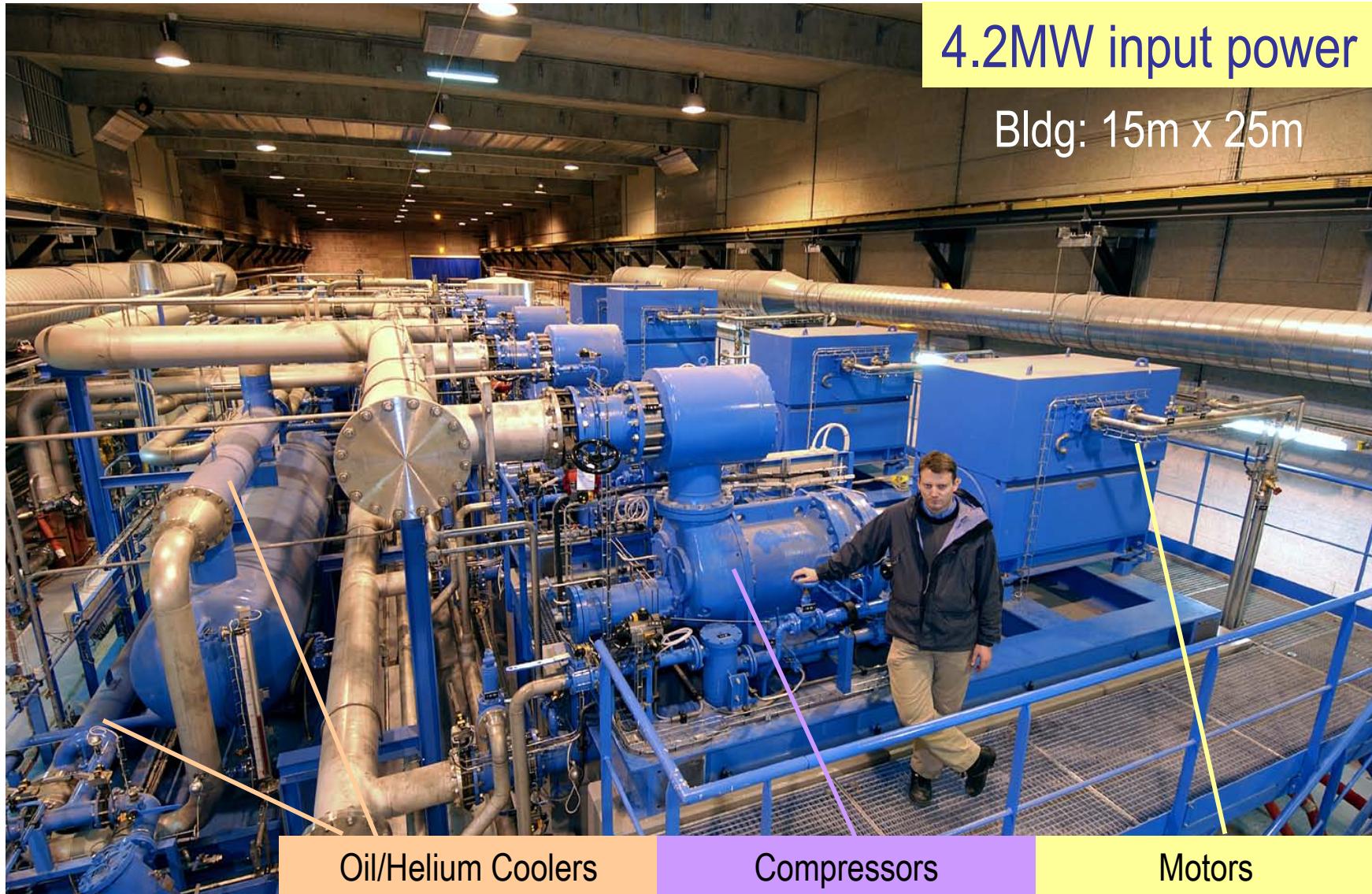
Oil lubricated screw compressors, water cooled, oil separation included



Machine derived from industrial refrigeration (or compressed air)

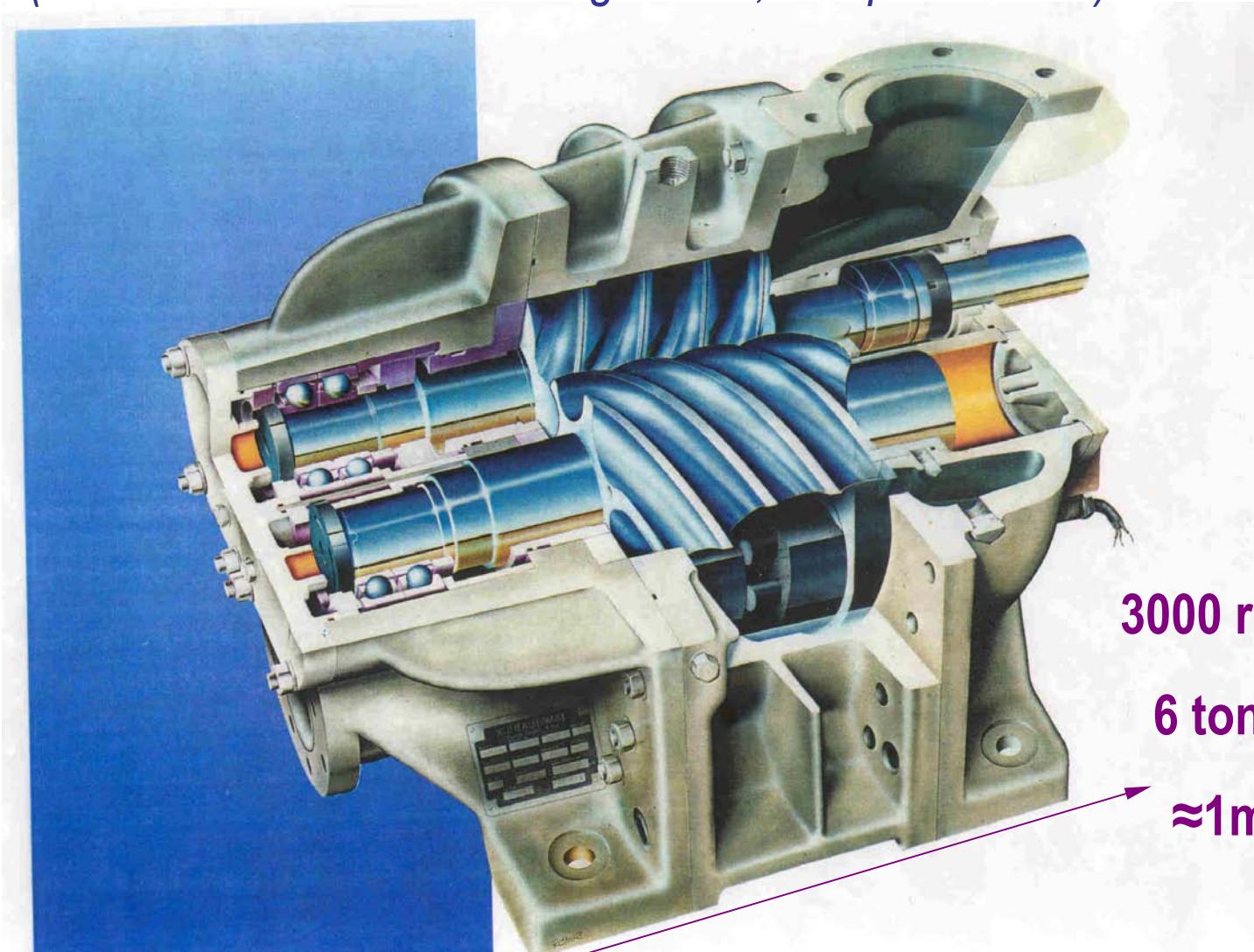
No more piston (high PR, low flow), not yet centrifugal (high flow, low PR)

Compressor station of LHC 18 kW@ 4.5 K helium refrigerator

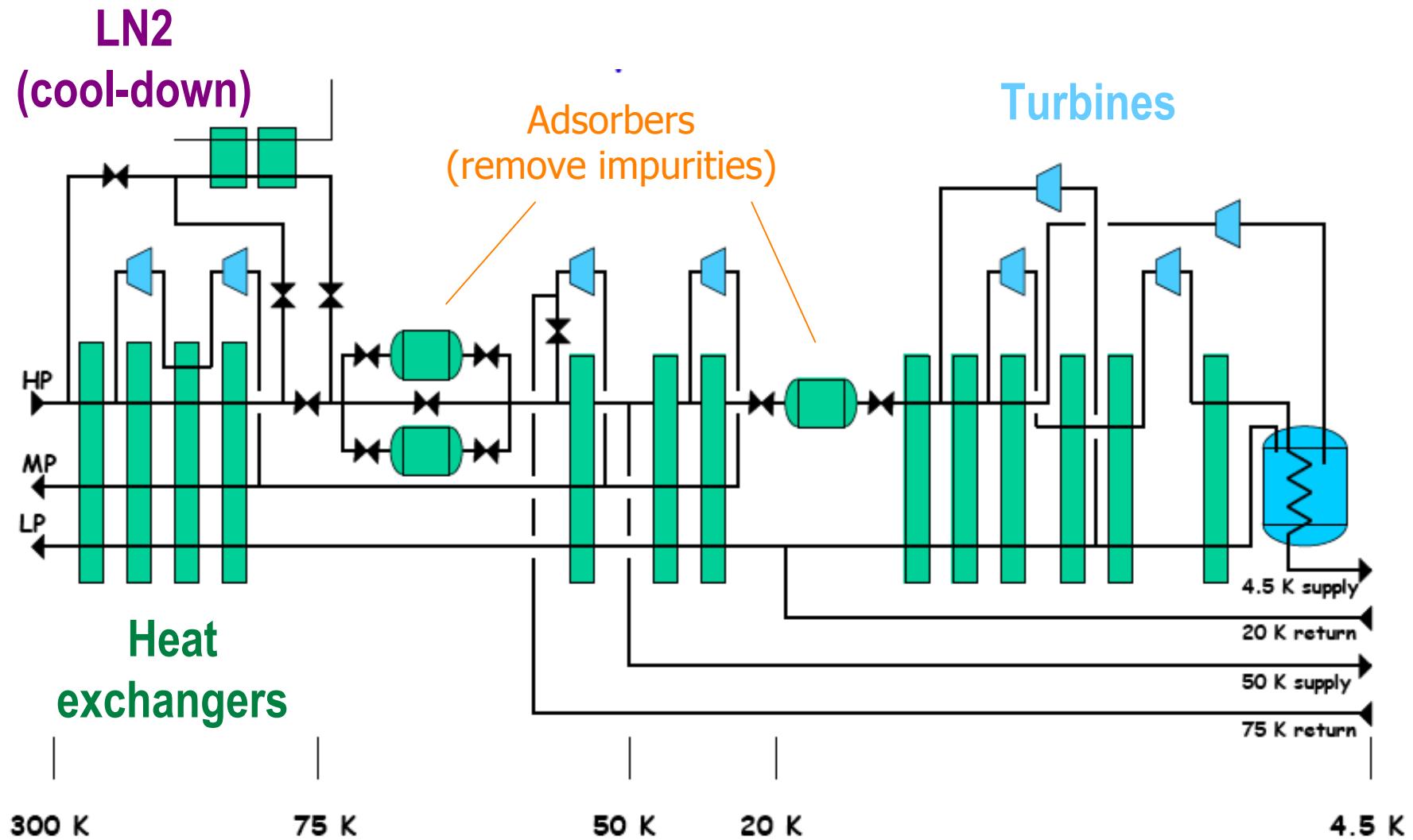


Oil-injected screw compressor

(derived from Industrial refrigeration, compressed air)



Process diagram, LHC refrigerator 18 kW @ 4.5 K



LHC 18 kW @ 4.5 K helium cryoplants

33 kW @ 50 K to 75 K, 23 kW @ 4.6 K to 20 K, 41 g/s liquefaction



Air Liquide



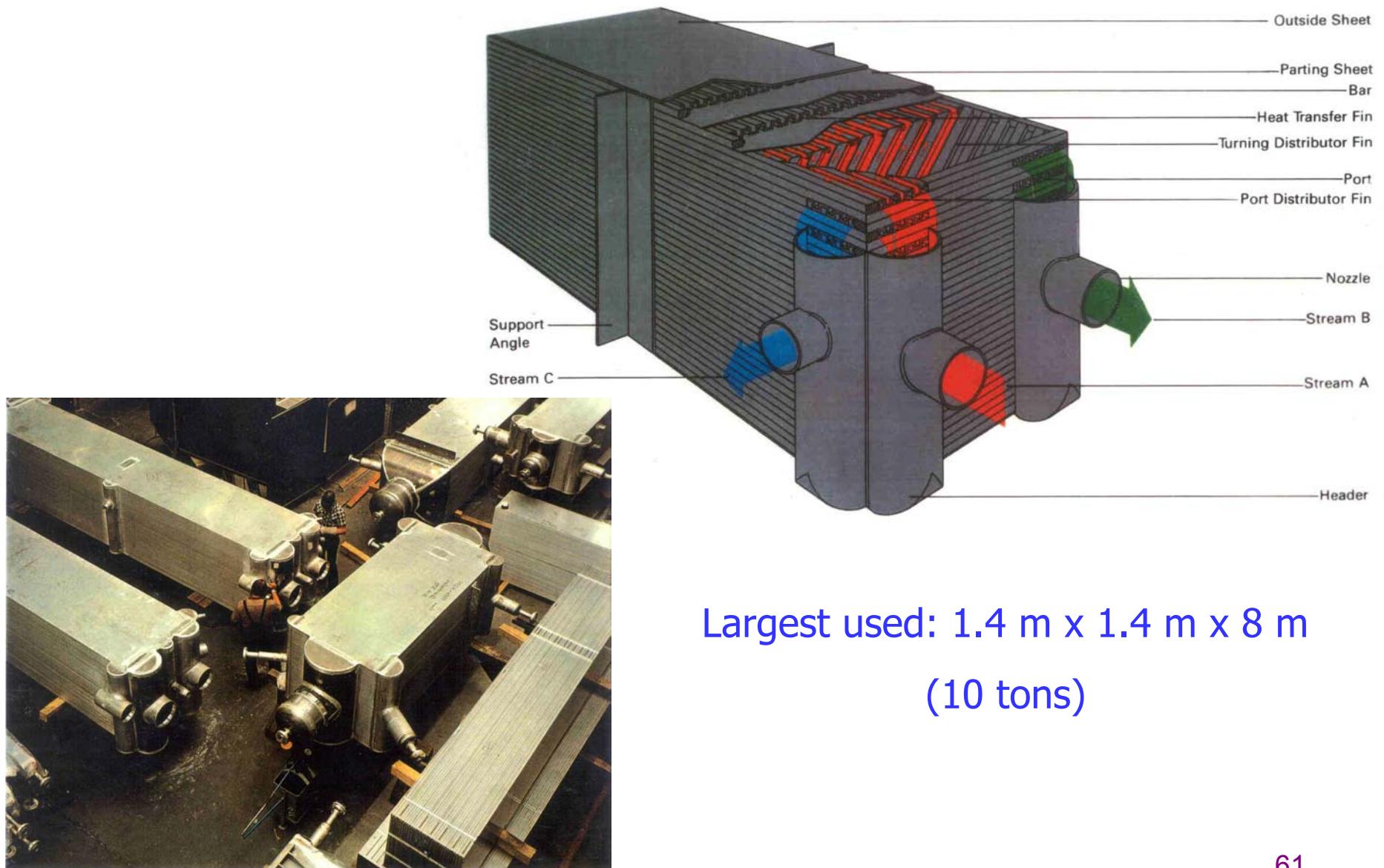
Linde



Diameter: 4 m
Length: 20 m
Weight: 100 tons
600 Input/Output signals



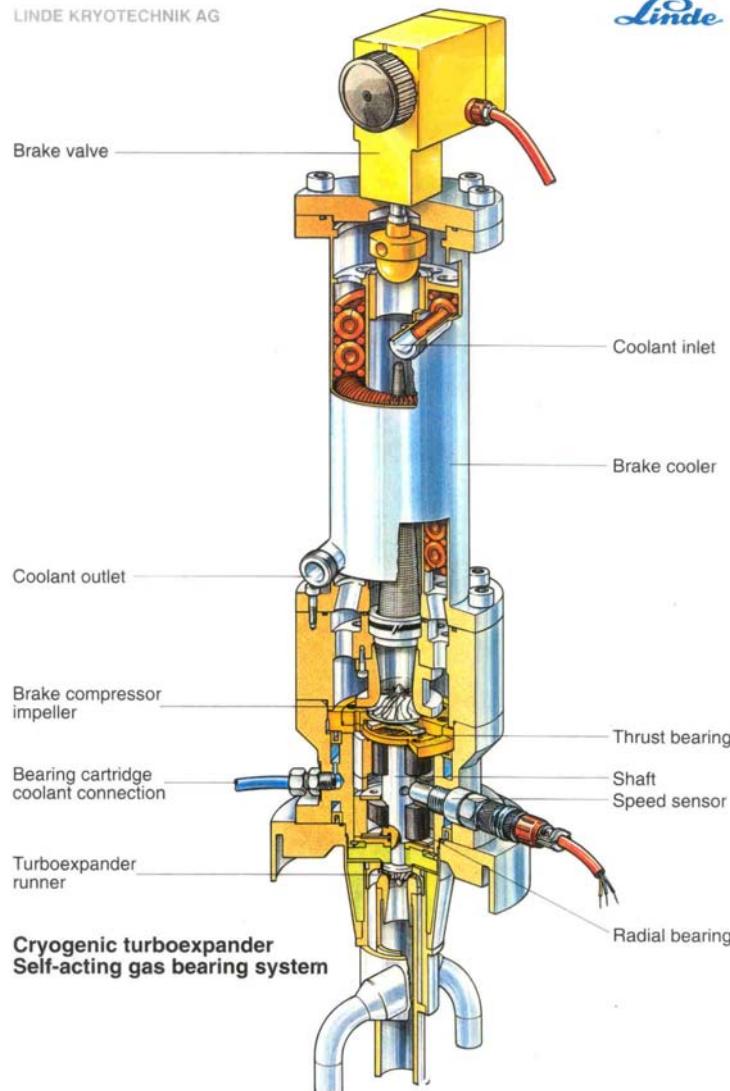
Brazed aluminium plate heat exchanger



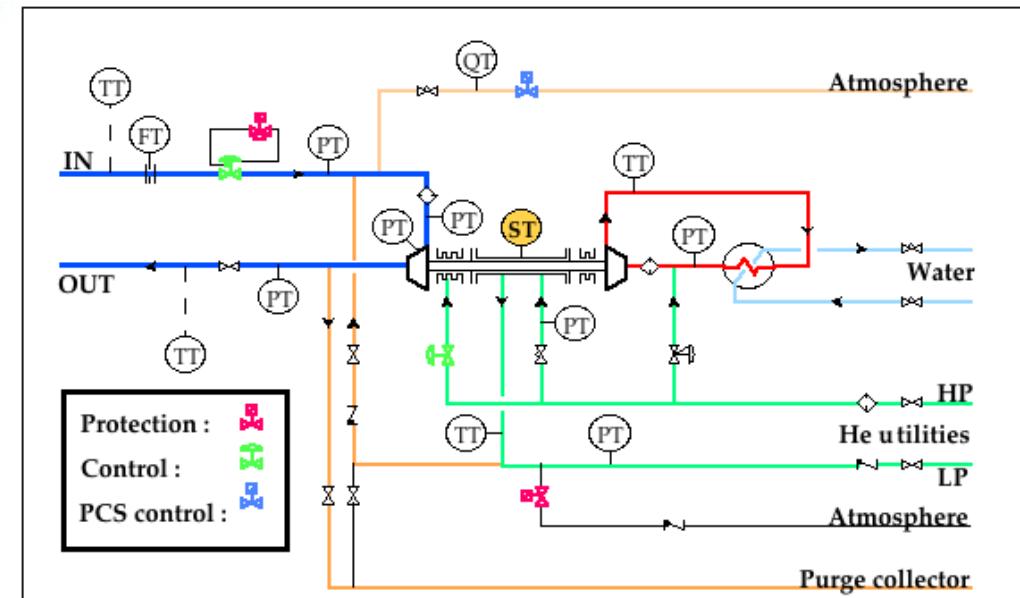
Cryogenic turbo-expander

Specific technology "contact free" gas bearings operated at 120'000 rpm

LINDE KRYOTECHNIK AG

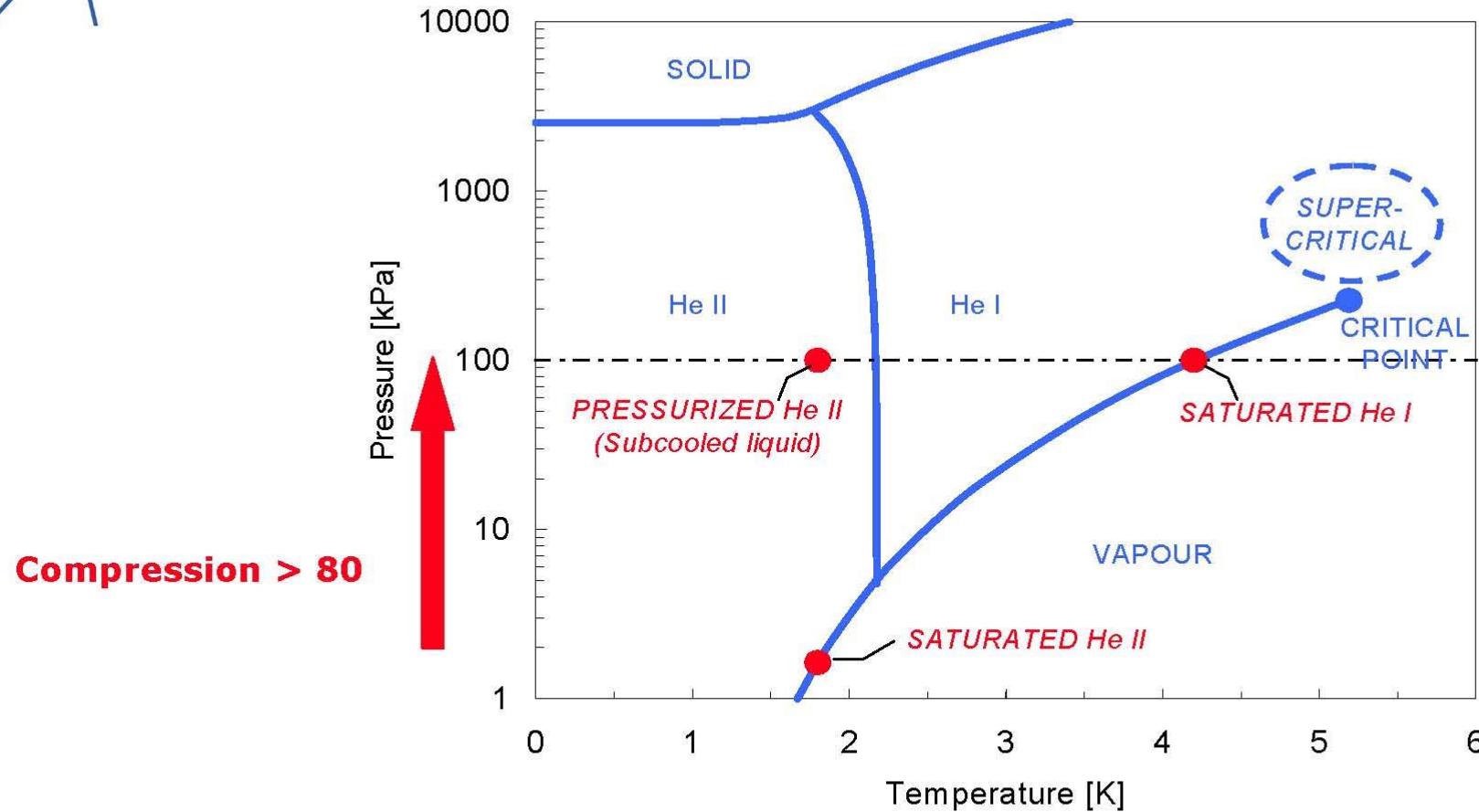


Linde



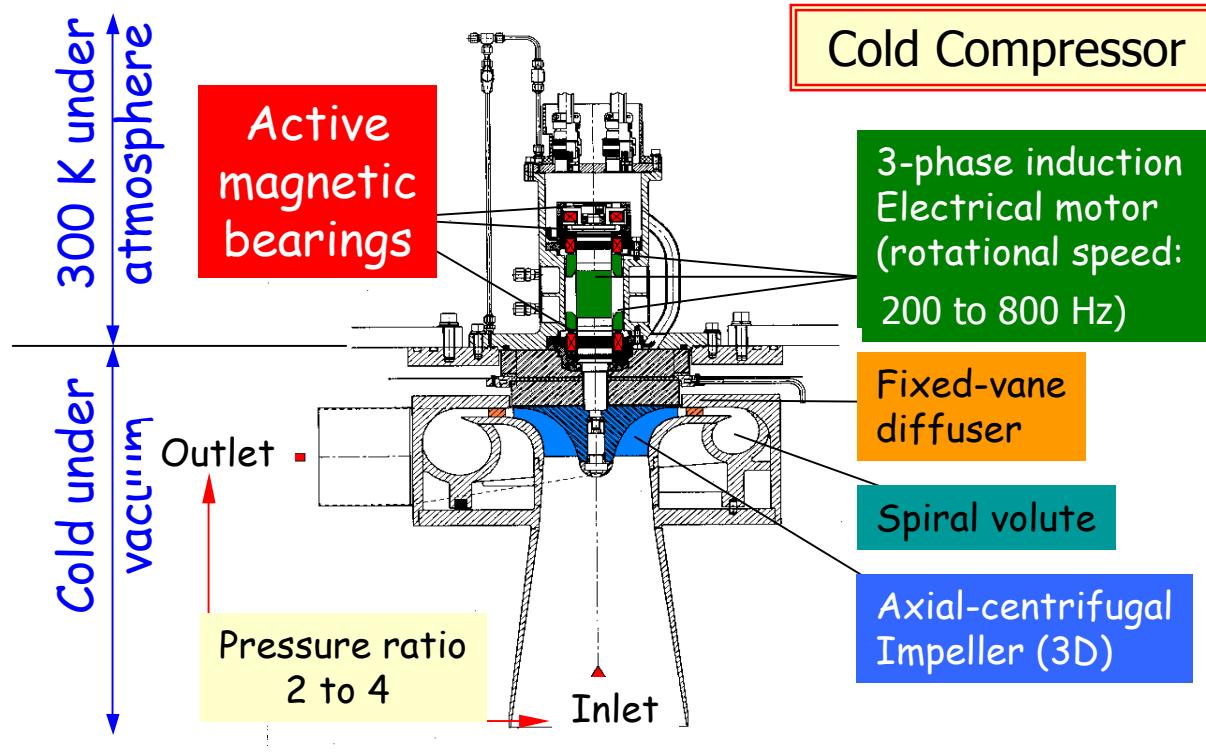
Wheel diameter: 5-15 cm
Shaft length: 20 cm
Rotation: 60'000 to 150'000 rpm

Challenges of power refrigeration at 1.8 K

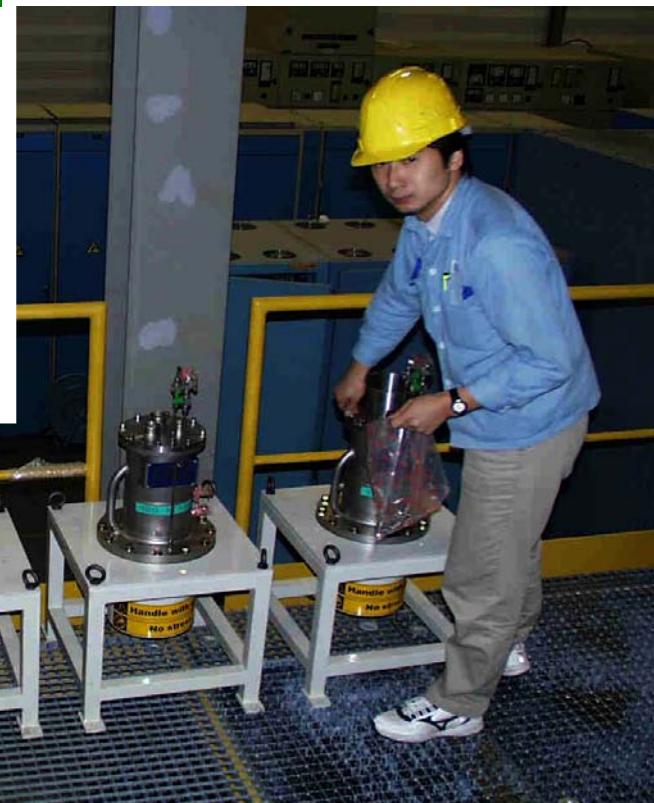


- Compression of large mass flow-rate of He vapor across high pressure ratio
 \Rightarrow intake He at maximum density, i.e. cold
- Need contact-less, vane-less machine \Rightarrow hydrodynamic compressor
- Compression heat rejected at low temperature \Rightarrow thermodynamic efficiency

1.8K Units with cold compressors (x8)



Specific technology to allow large capacity below 2K



Contents

- Introduction
- Cryogenic fluids
- Heat transfer & thermal insulation
- Cryogenic distribution & cooling schemes
- Refrigeration & liquefaction
- Various complements
- Concluding remarks, references

Bulk Liquid & Gaseous cryogen storage solutions



Deliveries in Liquid form:

- 60 trucks LN2 to cool a LHC sector to 80K (14 days-1'200t)
- 20 trucks for external storage of helium (4 months - 90tons)

250m³ Gaseous He
(20B - 850kg He)



How to deal with impurities

- Any liquid or gas other than helium would solidify during the cooling process. This could block the helium flow or degrade moveable components (valves, turbines)
- Typical treatment applied for: Water, air (N₂ and O₂), H₂ (adsorption on porous medium like activated charcoal, molecular thief)

Recommendation:

- ⇒ evacuation of air once circuits are leak-tight (pur helium)
- ⇒ on-line treatment of what could remain or arrive during operation, with target of fraction of ppm(v)

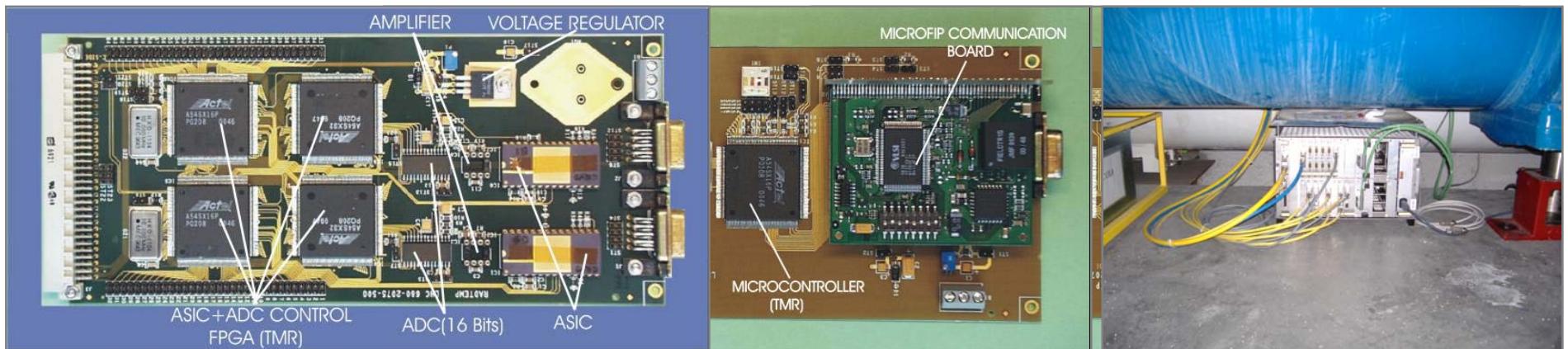
Thermometry

Industrial instrumentation whenever possible, specific developments when necessary

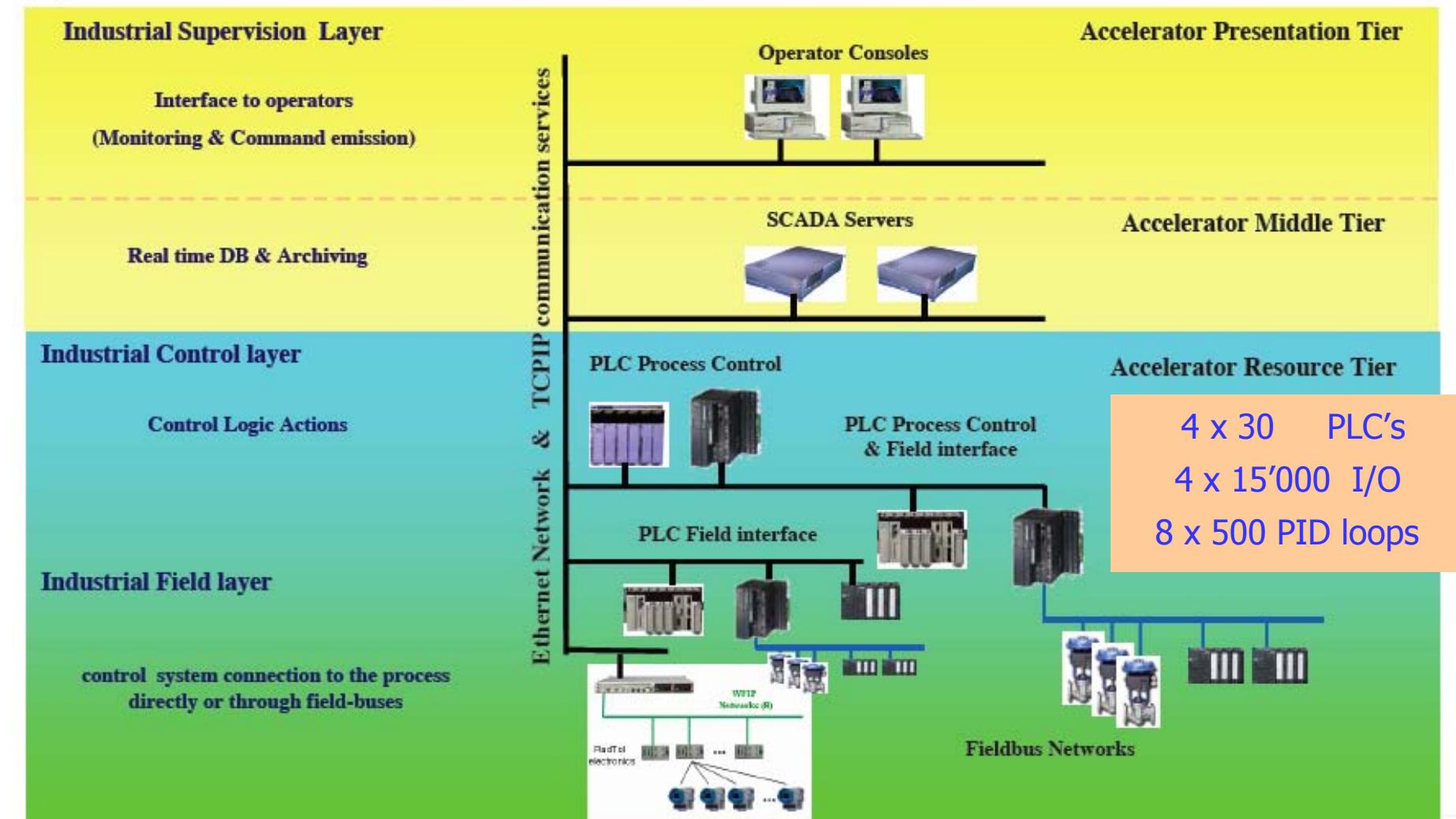
6'000 units, +/- 10 mK @ 2K in LHC radiation conditions



From 'sensor' to 'thermometer' with signal processing



Industrial Control Architecture



Cryo operator in Cern Central Control room



Safety notes

- Major risks associated with cryogenic fluids at low temperatures:
 - **Asphyxia:** Oxygen is replaced by a pure
 - **Cold burns:** in case of contact with cold surfaces
 - **Explosion:** pressure rise in case of warm-up at constant volume
(1l Liq \approx 700 l gas)
 - **Embrittlement:** Thermal contractions, potential fragile at cold
- Be informed about valid standards, like for pressure vessels, safety devices, transport of cryogens, ...

Concluding remarks

- Cryogenics serving superconducting systems is now part of all major accelerators and future projects.
- While advanced applications tend to favor “below 2K”, many almost industrial applications are based on “4.5K” and RnD (or demonstrators) continues for “high temperature” applications
- If cryogenic engineering follows well defined rules and standards, there are variants depending on boundary conditions, continents, time of a project...
I could only recommend that demonstrated experience be evaluated and adapted to specific requirements you may have !

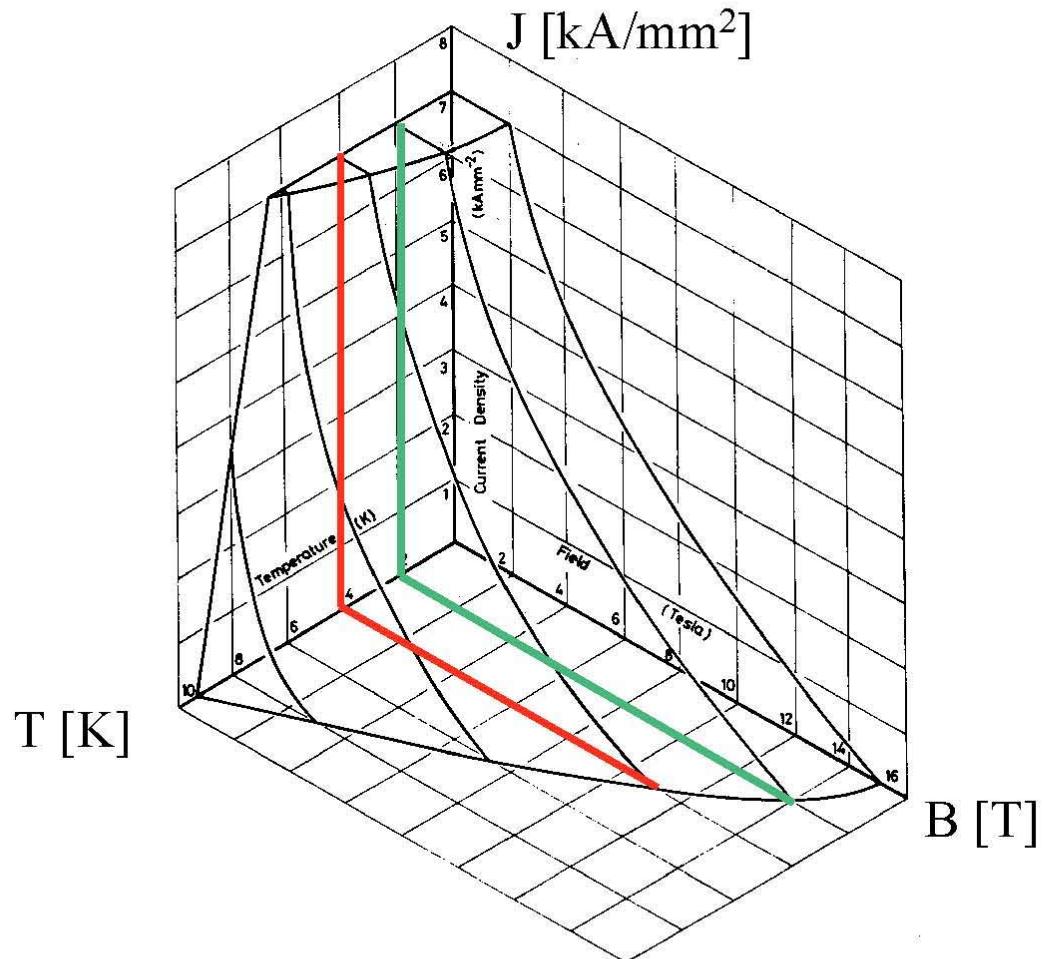
Thanks for your attention,
and hoping you would (now) be more aware with cryogenics !!!

Some references

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- R.B. Scott, *Cryogenic engineering*, Van Nostrand, Princeton (1959)
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- K.D. Timmerhaus & T.M. Flynn, *Cryogenic process engineering*, Plenum Press, New York (1989)
- Proceedings of *CAS School on Superconductivity and Cryogenics for Particle Accelerators and Detectors*, Erice (2002) (+2013)
 - U. Wagner, *Refrigeration*
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 - Ph. Lebrun, *Design of a cryostat for superconducting accelerator magnet*
 - Ph. Lebrun & L. Tavian, *The technology of superfluid helium*
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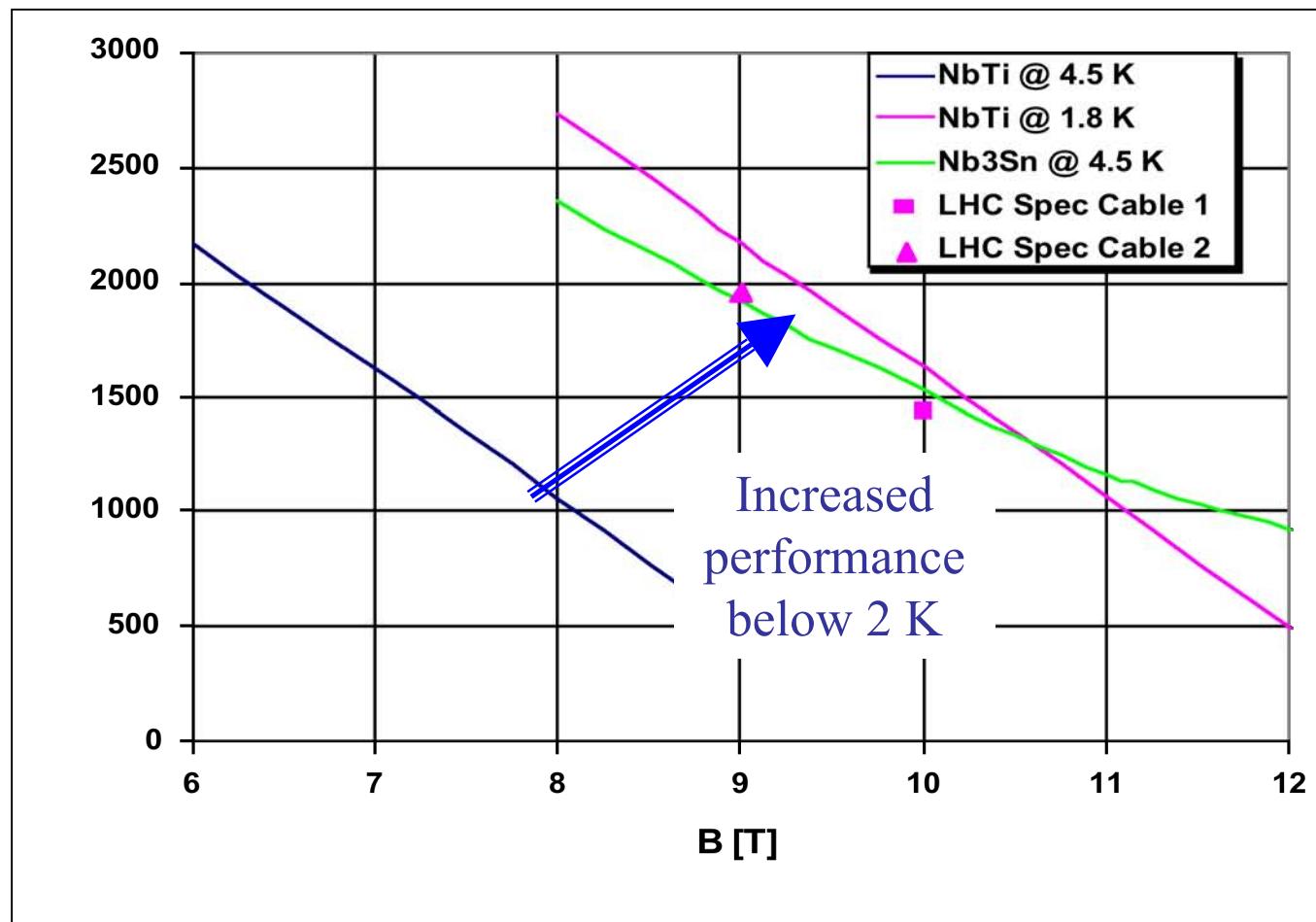
Bonus slides

Operating temperature & performance of superconductors



- Superconductivity only exists in a limited domain of temperature, magnetic field and current density
- Electrotechnical applications require transport current and magnetic field
- Operating temperature of the device must therefore be significantly lower than the critical temperature of the superconductor

Superconducting magnets

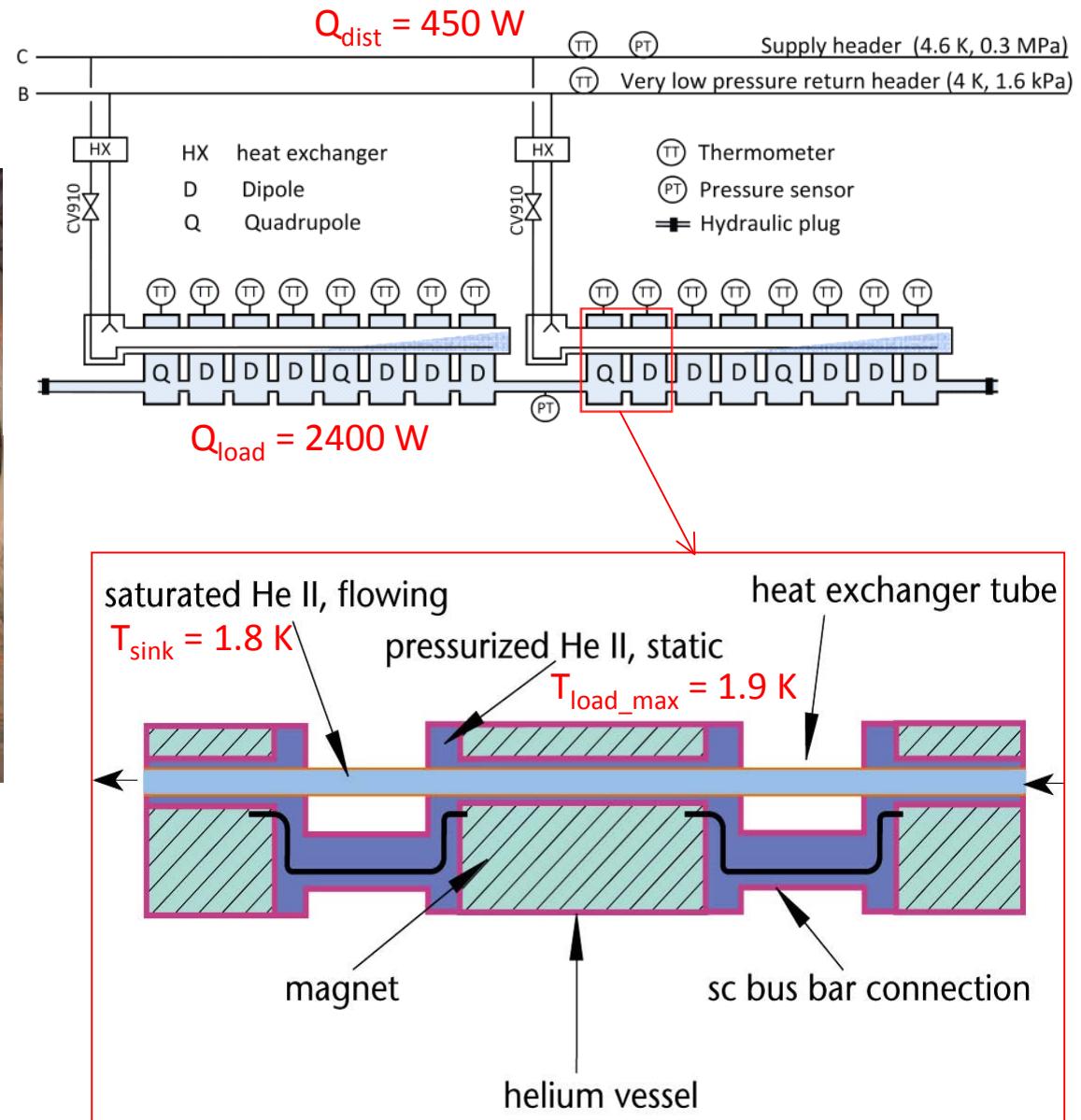


LHC sector cooling scheme

Pressurized/saturated He II



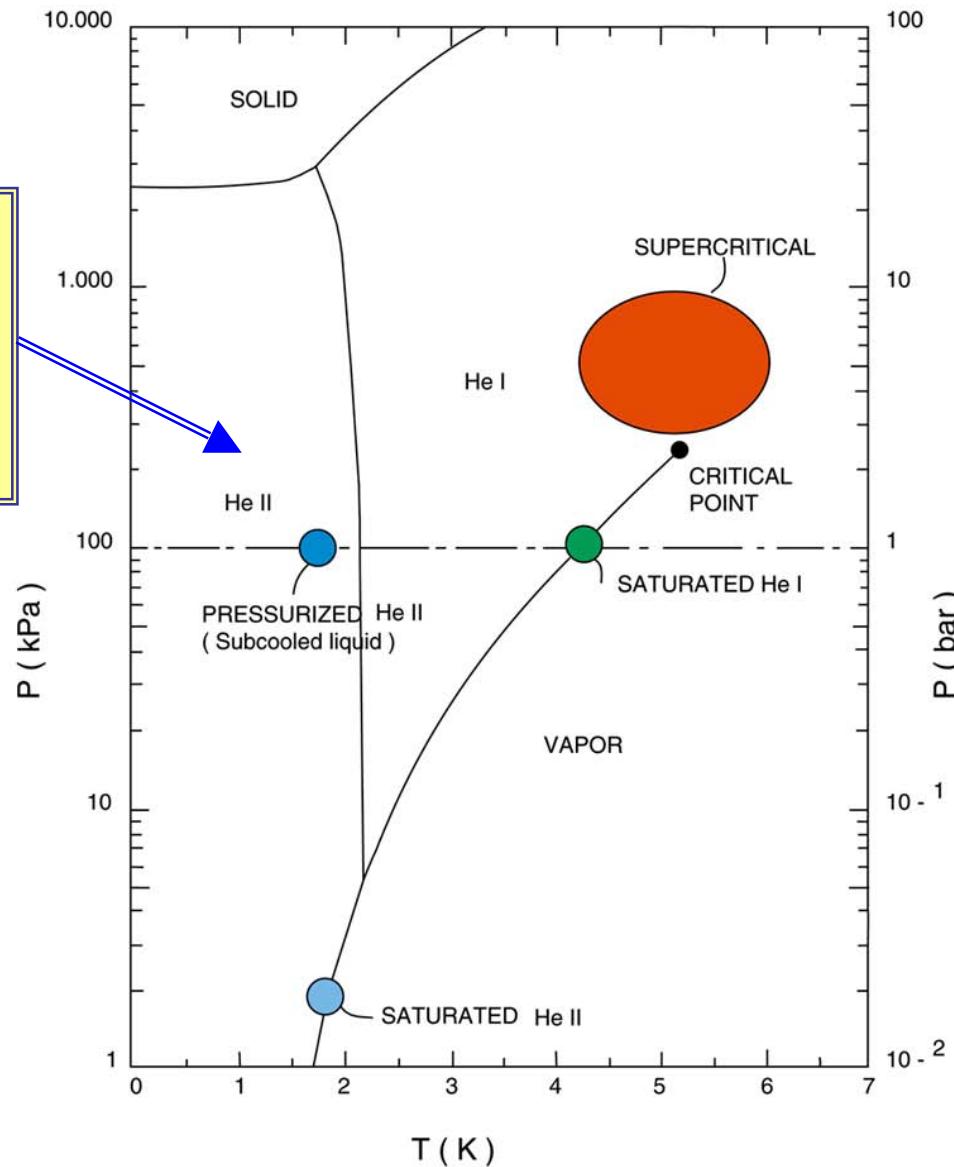
37'500 tons at 1.9 K



Helium phase diagram

Superfluid Helium:

- Lower viscosity
- Larger heat transfer capacity



Basic thermodynamics at low temperature

- Minimum refrigeration work W_{\min} to extract heat Q at temperature T and reject it at ambient temperature T_a

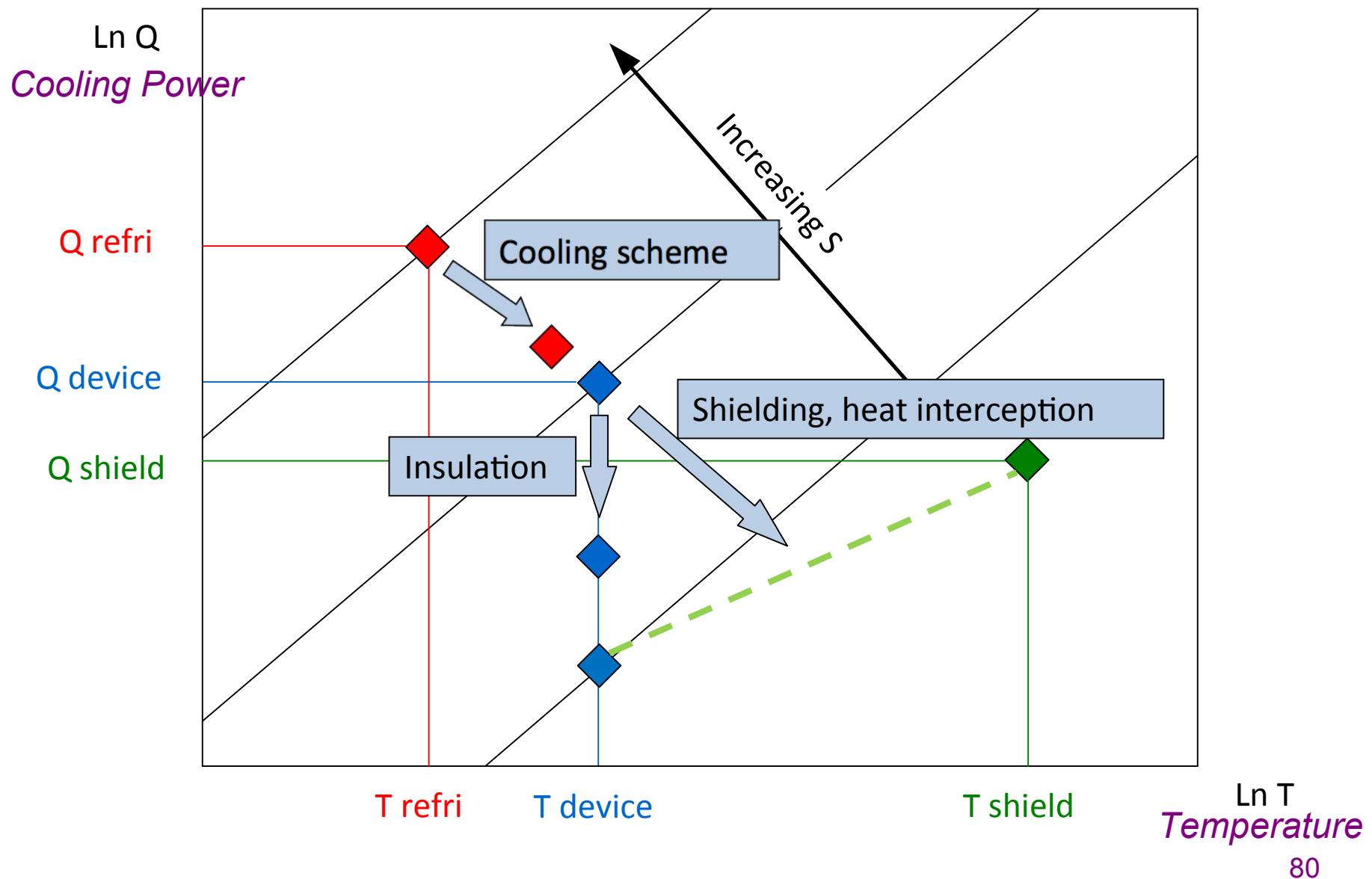
$$W_{\min} = Q (T_a/T - 1) = T_a \Delta S - Q$$

- At cryogenic temperature $T \ll T_a$

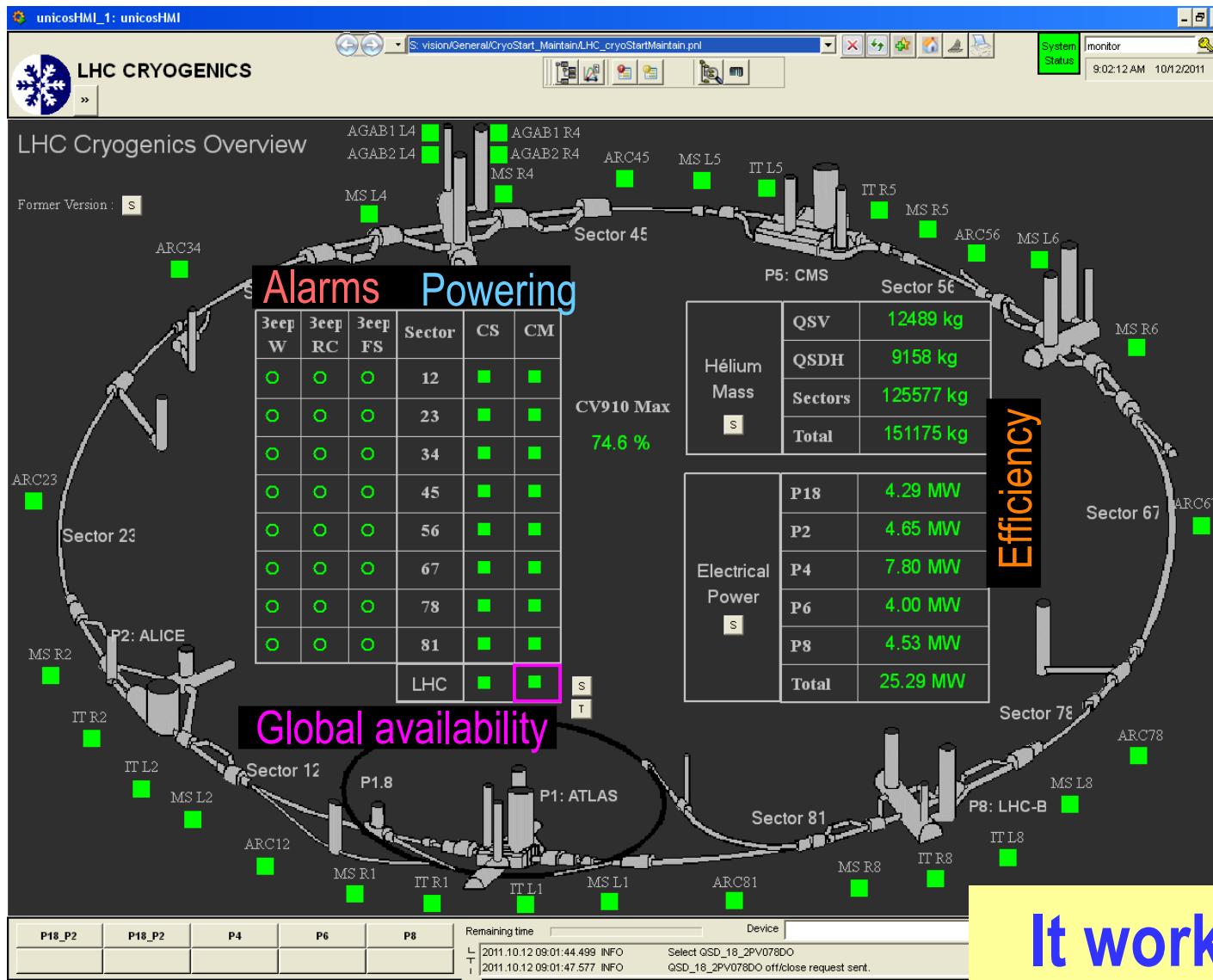
$$W_{\min} \simeq Q T_a/T \simeq T_a \Delta S$$

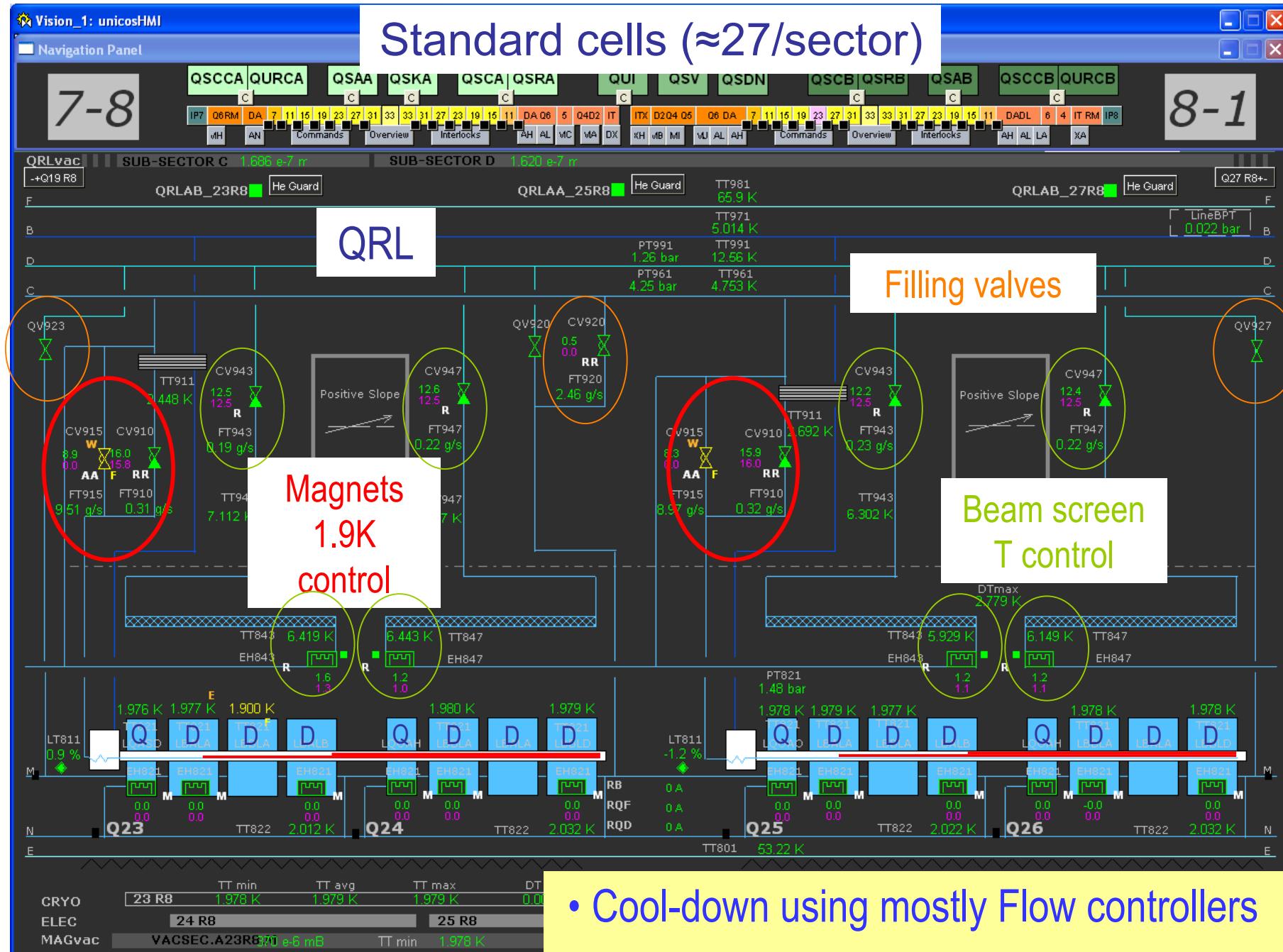
→ entropy is a good measure of the cost of cryogenic refrigeration
→ strategies minimizing ΔS improve cryogenic design

Cryogenic design strategies



Operation, indicators





- Cool-down using mostly Flow controllers
- P, T, L controllers at operating conditions