

# Introduction to Cryogenics for accelerators

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CAS

Basics of Accelerator Science and Technology at CERN

Chavanne de Bogis, CH

4-8 Nov 2013

# Préambule

## Reference

Great thanks to predecessors for this type of exercise, particularly to Ph. Lebrun and his “legacy” of slides

## Disclaimer

Being an engineer and new in this domain as “teacher”, I will try to share with you some information with emphasis on “applied cases” with a “pragmatic approach” rather than only a theoretical one.

*There are plenty of books, previous CAS courses with lot's of formulas and various equations. I leave it to you to check bibliography if this is what you are looking for!*

# Contents

- Introduction
- Cryogenic fluids
- Heat transfer & thermal insulation
- Cryogenic distribution & cooling schemes
- Refrigeration & liquefaction
- Various complements
- Concluding remarks, references

- **cryogenics**, that branch of physics which deals with the production of very low temperatures and their effects on matter

*Oxford English Dictionary*

2<sup>nd</sup> edition, Oxford University Press (1989)

- **cryogenics**, the science and technology of temperatures below 120 K

*New International Dictionary of Refrigeration*

3<sup>rd</sup> edition, IIF-IIR Paris (1975)

Temperature in Celsius (C): unit defined with 0 C (ice) and 100 C (vapour)

Temperature in Kelvin (K): 1 K = 1 C, but 0 K = -273.15 C (absolut zero)

# Densification, liquéfaction & séparation des gaz

LNG



130 000 m<sup>3</sup> LNG carrier  
with double hull

Air separation by cryogenic  
distillation

Up to 4500 t/day LOX



LIN & LOX

Rocket fuels

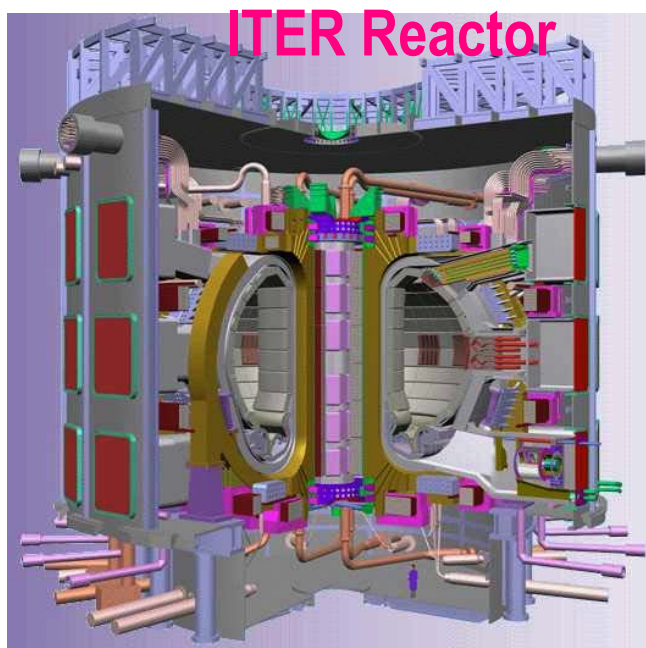


Ariane 5

25 t LHY, 130 t LOX



# Cooling of superconducting devices



# Main reasons to superconducting

*For accelerators in high energy physics*

- Compactness through higher fields

Capital Cost

$$E_{\text{beam}} \approx 0.3 \cdot \mathbf{B} \cdot r$$

[Gev]            [T]   [m]

$$E_{\text{beam}} \approx \mathbf{E} \cdot L$$

[Gev]            [MV/m]   [m]

At design stage, working at highest possible temperature is always considered, but often not selected to maximise beam energy ...

Cryogenic systems takes longer to recover from failures than conventional ones !  
(but there is work on it!)

- Saving operating energy

Operating Cost

Electromagnets:

Resistive:  $P_{\text{input}} \approx E_{\text{beam}}$

Superconducting:  $P_{\text{input}} \approx P_{\text{ref}}$

Acceleration cavities

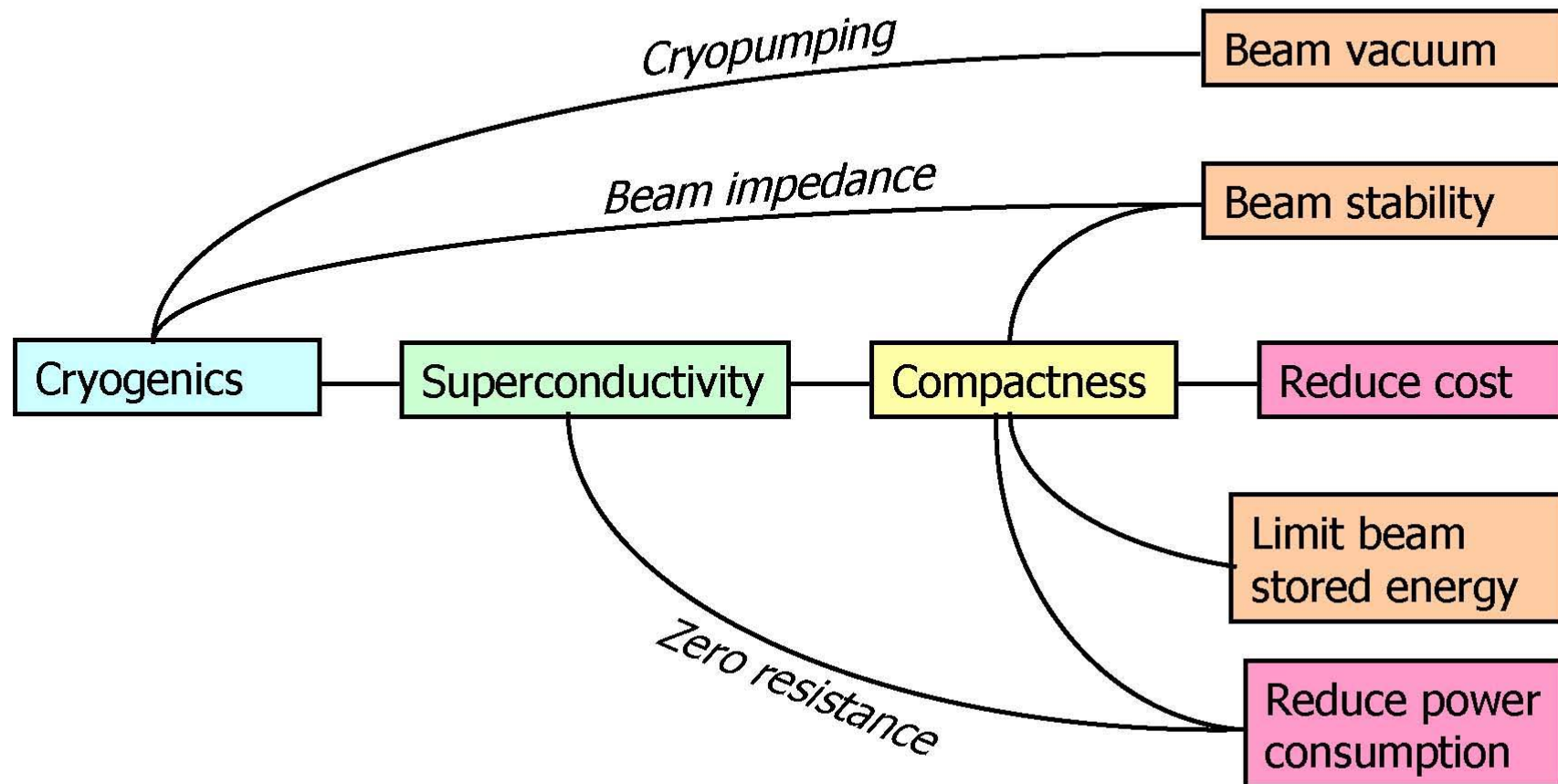
$$P_{\text{input}} \approx R_s \cdot L \cdot \mathbf{E}^2 / w$$

$$R_s \approx R_{\text{BCS}} + R_o$$

$$R_{\text{BCS}} \approx (1/T) \exp(-BT_c/T)$$



## Rationale for superconductivity & cryogenics in particle accelerators

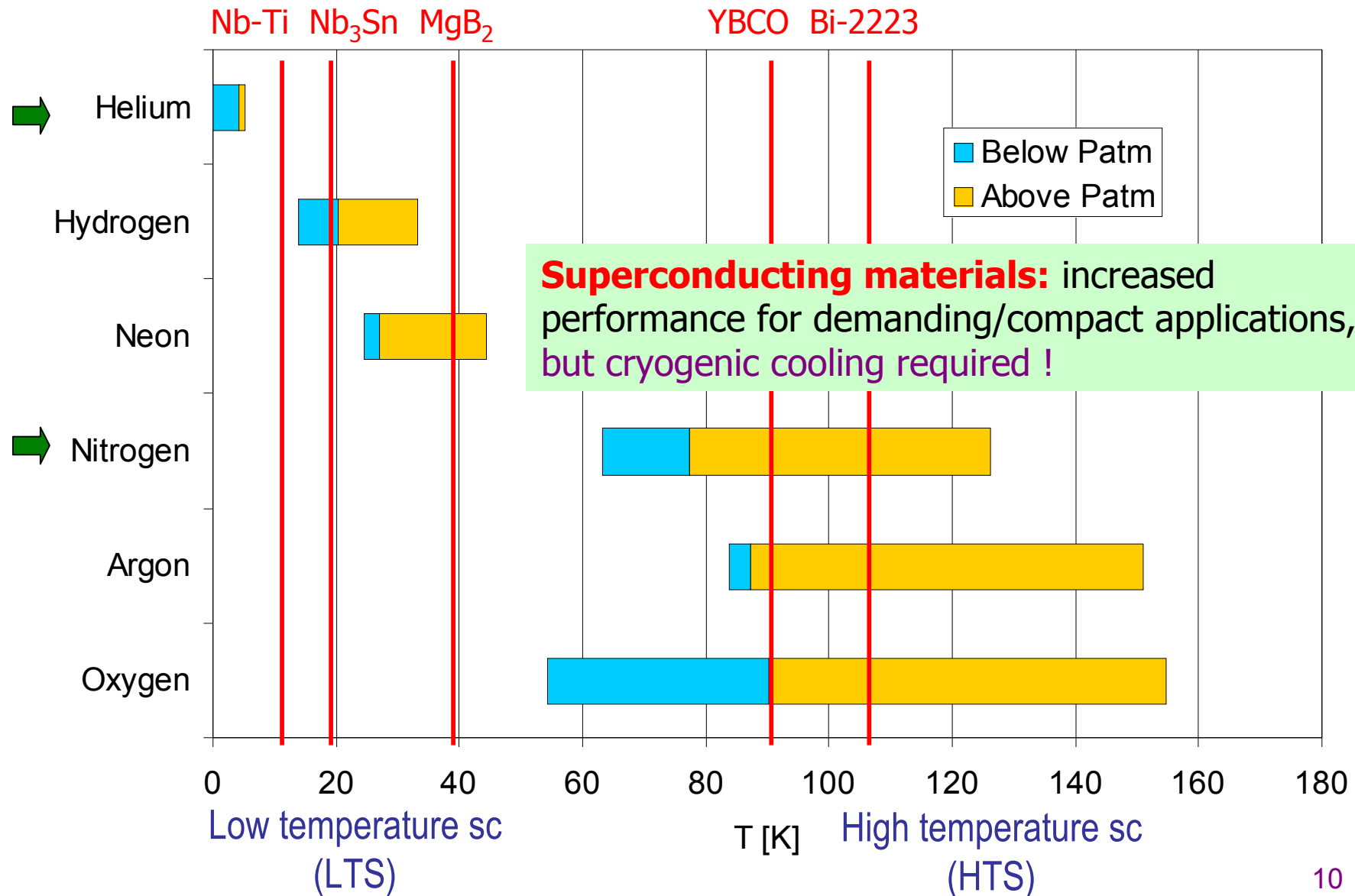




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# Useful range of cryogenics, and potential applications



## Characteristic temperatures of cryogenes

| <b>Cryogen</b> | <b>Triple point [K]</b> | <b>Normal boiling point [K]</b> | <b>Critical point [K]</b> |
|----------------|-------------------------|---------------------------------|---------------------------|
| Methane        | 90.7                    | 111.6                           | 190.5                     |
| Oxygen         | 54.4                    | 90.2                            | 154.6                     |
| Argon          | 83.8                    | 87.3                            | 150.9                     |
| Nitrogen       | 63.1                    | 77.3                            | 126.2                     |
| Neon           | 24.6                    | 27.1                            | 44.4                      |
| Hydrogen       | 13.8                    | 20.4                            | 33.2                      |
| Helium         | 2.2 (*)                 | 4.2                             | 5.2                       |

(\*):  $\lambda$  point

## Properties of cryogenics compared to water

| Property                 |                      | He   | N <sub>2</sub> | H <sub>2</sub> O |
|--------------------------|----------------------|------|----------------|------------------|
| Normal boiling point     | [K]                  | 4.2  | 77             | 373              |
| Critical temperature     | [K]                  | 5.2  | 126            | 647              |
| Critical pressure        | [bar]                | 2.3  | 34             | 221              |
| Liq./Vap. density (*)    |                      | 7.4  | 175            | 1600             |
| Heat of vaporization (*) | [J.g <sup>-1</sup> ] | 20.4 | 199            | 2260             |
| Liquid viscosity (*)     | [μPa]                | 3.3  | 152            | 278              |

(\*) at normal boiling point

# Vaporization of normal boiling cryogenics under 1 W applied heat load

$$\text{Power} \approx \dot{m} \cdot \text{Latent\_Heat}$$

[W]      [g/s]      [J/g]

| Cryogen  | [mg.s <sup>-1</sup> ] | [l.h <sup>-1</sup> ]<br>(liquid) | [l.min <sup>-1</sup> ]<br>(gas NTP) |
|----------|-----------------------|----------------------------------|-------------------------------------|
| Helium   | 48                    | 1.38                             | 16.4                                |
| Nitrogen | 5                     | 0.02                             | 0.24                                |

# Amount of cryogenics required to cool down 1 kg iron

$$\text{Power} \approx m' \cdot \text{Latent\_Heat}$$

[W]      [g/s]      [J/g]

$$\text{Power} \approx m' \cdot \text{Specific\_Heat} \cdot \Delta T$$

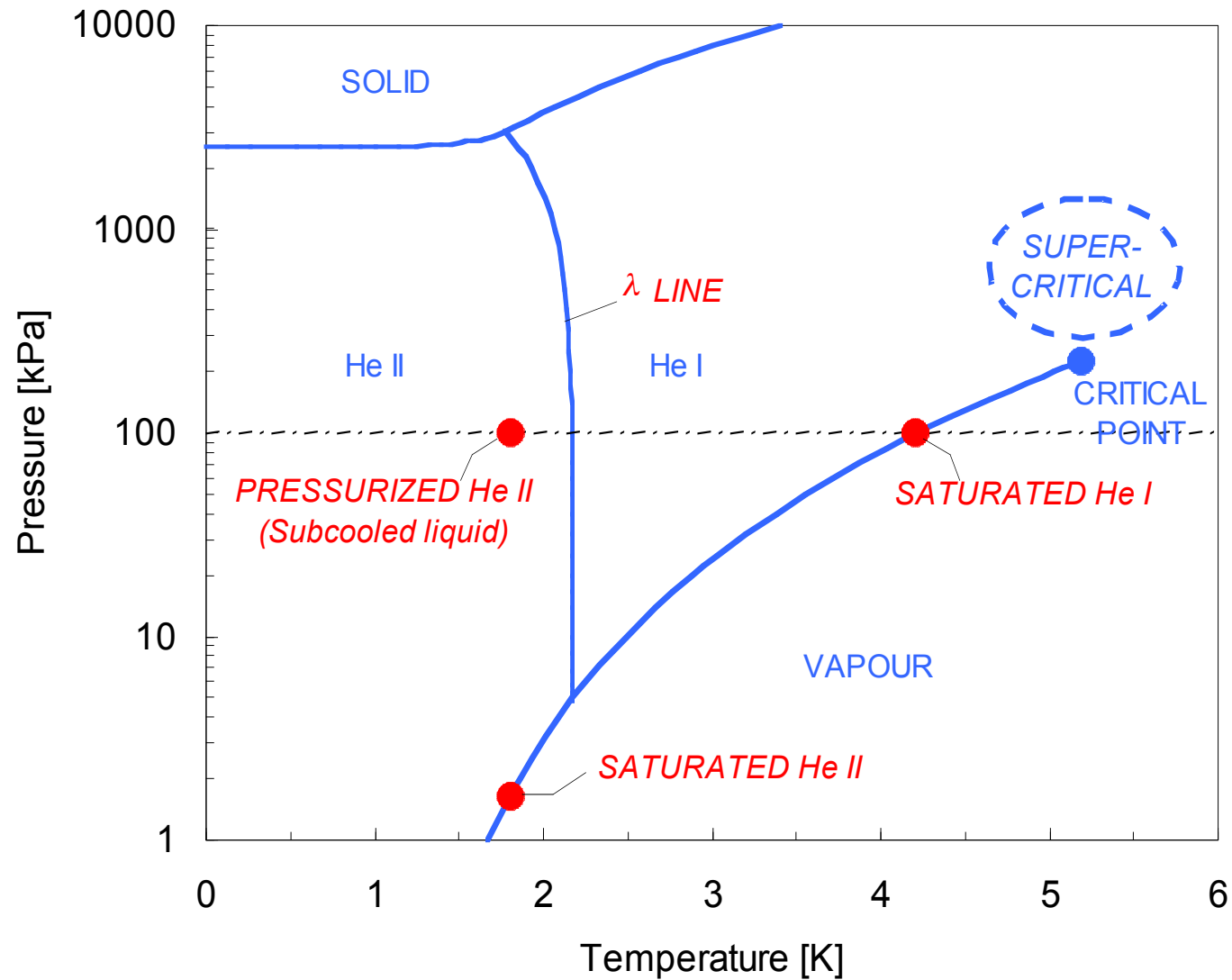
[W]      [g/s]      [J/g.K]      [K]



| Using                 | Latent heat only | Latent heat and enthalpy of gas |
|-----------------------|------------------|---------------------------------|
| LHe from 290 to 4.2 K | 29.5 litre       | 0.75 liter                      |
| LHe from 77 to 4.2 K  | 1.46 litre       | 0.12 litre                      |
| LN2 from 290 to 77 K  | 0.45 litre       | 0.29 litre                      |



# Phase diagram of helium



## Helium as a cooling fluid

| Phase domain   | Advantages  | Drawbacks                                  |
|----------------|---|--|
| Saturated He I | Fixed temperature<br>High heat transfer               | Two-phase flow<br>Boiling crisis           |
| Supercritical  | Monophase<br>Negative J-T effect                      | Non-isothermal<br>Density wave instability |
| He II          | Low temperature<br>High conductivity<br>Low viscosity | Second-law cost<br>Subatmospheric          |

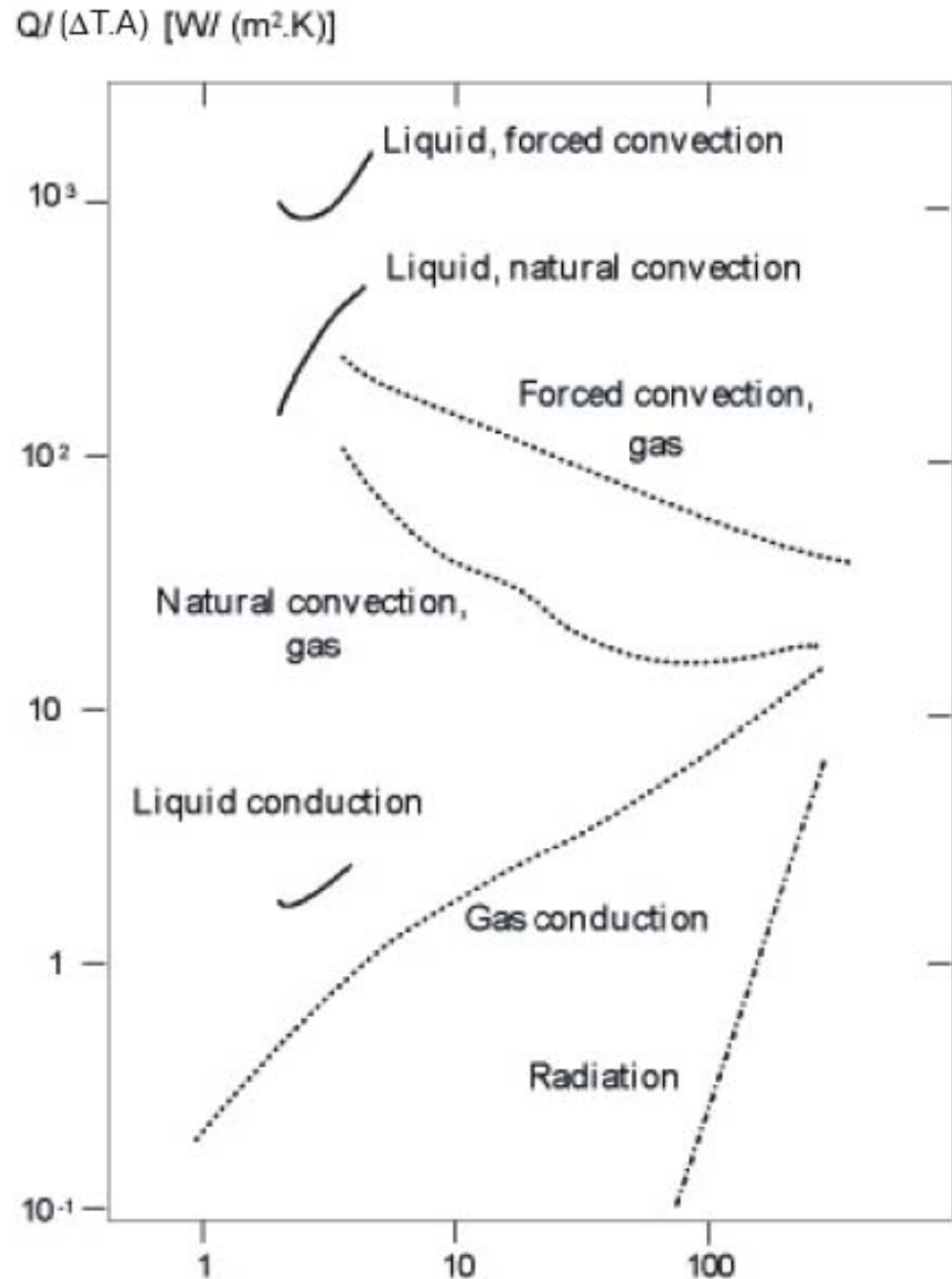
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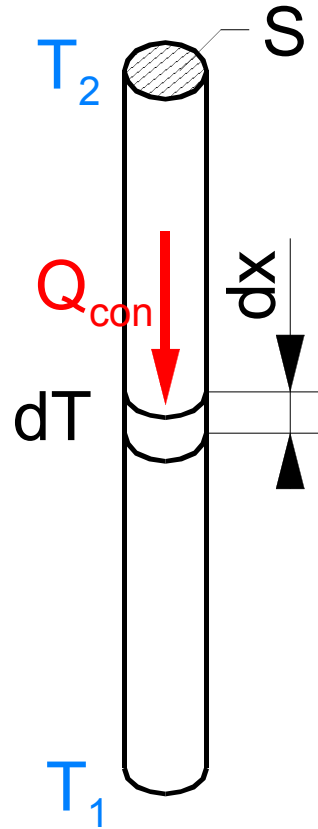
# Typical heat transfer coefficients at cryogenic temperatures

3 mechanisms involved:

- Conduction
- Radiation
- Convection



# Heat conduction in solids



Fourier's law:

$$Q_{\text{con}} = k(T) \cdot S \cdot \frac{dT}{dx}$$

$k(T)$ : thermal conductivity [W/m.K]

Integral form:

$$Q_{\text{con}} = \frac{S}{L} \cdot \int_{T_1}^{T_2} k(T) \cdot dT$$

$\int k(T) \cdot dT$  : thermal conductivity integral [W/m]

Thermal conductivity integrals for standard construction materials are tabulated

Risks associated with "optimisation":

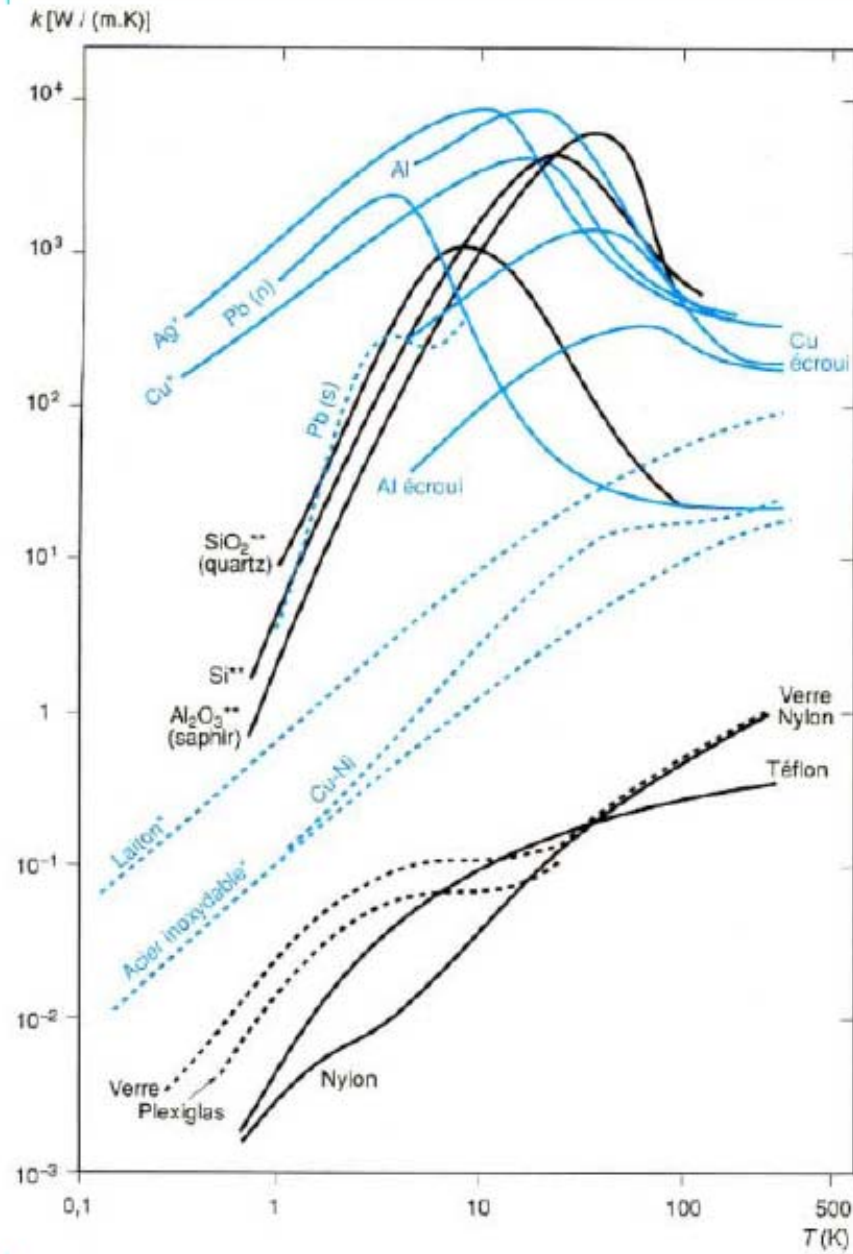
- small section  $S$ : towards limit for material resistance
- long length  $L$ : towards limits for mechanical stability
- insulators (large)  $K$ : difficulties with transfer of forces

# Thermal conductivity integrals, selection of materials [W/m]

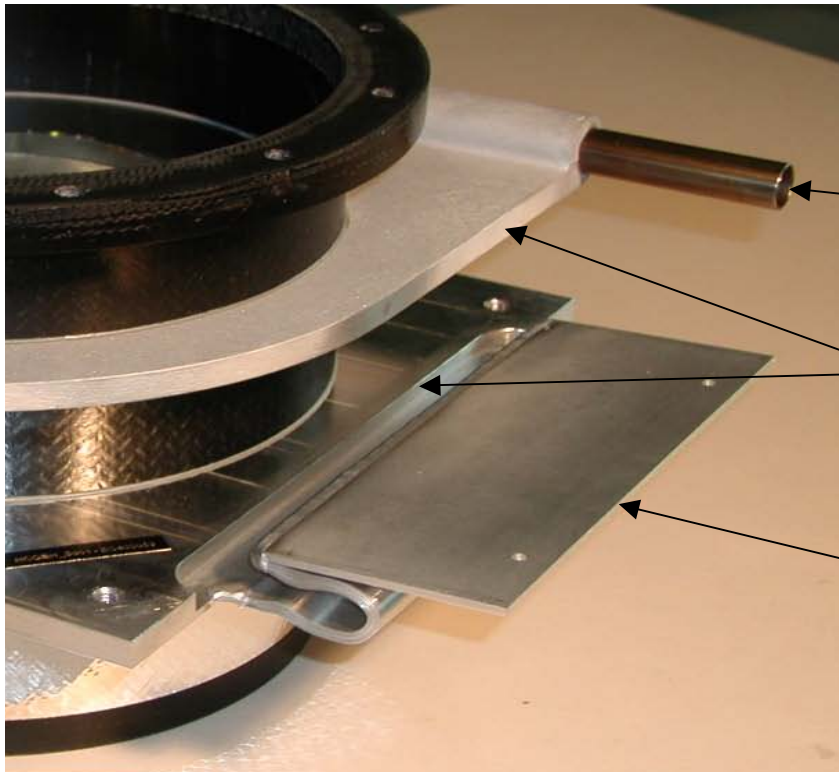
| From vanishingly low temperature up to | 20 K  | 80 K  | 290 K  |
|--|-------|-------|--------|
| OFHC copper                            | 11000 | 60600 | 152000 |
| DHP copper                             | 395   | 5890  | 46100  |
| 1100 aluminium                         | 2740  | 23300 | 72100  |
| 2024 aluminium alloy                   | 160   | 2420  | 22900  |
| AISI 304 stainless steel               | 16.3  | 349   | 3060   |
| G-10 glass-epoxy composite             | 2     | 18    | 153    |



## Thermal conductivity of materials at cryogenic temperatures



# Non-metallic composite support post with heat intercepts for LHC magnets



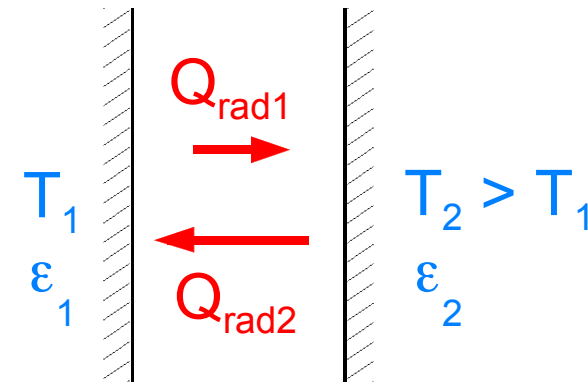
5 K cooling line (SC He)

Aluminium intercept plates  
glued to G-10 column

Aluminium strips to thermal  
shield at 50-75 K

# Thermal radiation

- Wien's law
  - Maximum of black body power spectrum  
 $\lambda_{max} T = 2898 [\mu\text{m.K}]$
- Stefan-Boltzmann's law
  - Black body
  - "Gray"body
  - "Gray" surfaces at  $T_1$  and  $T_2$



$$Q_{rad} = \sigma A T^4$$

$$\sigma = 5.67 \times 10^{-8} \text{ W/m}^2.\text{K}^4$$

(Stefan Boltzmann's constant)

$$Q_{rad} = \epsilon \sigma A T^4$$

$\epsilon$  emissivity of surface

$$Q_{rad} = E \sigma A (T_1^4 - T_2^4)$$

$E$  function of  $\epsilon_1, \epsilon_2$ , geometry

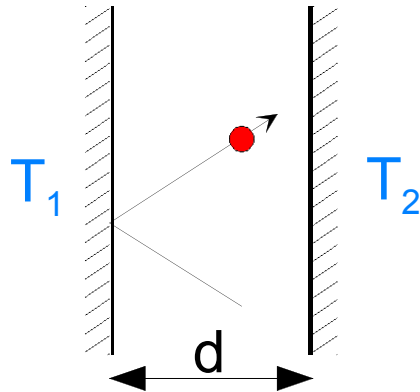
$$E \cdot T^4$$

Best would be to have a reflective (high  $E$ ) "parasol" to intercept  $T_4$  ...

## Emissivity of technical materials at low temperatures

|                                  | <b>Radiation from 290 K<br/>Surface at 77 K</b> | <b>Radiation from 77 K<br/>Surface at 4.2 K</b> |
|----------------------------------|---|---|
| Stainless steel, as found        | 0.34  | 0.12  |
| Stainless steel, mech. polished  | 0.12  | 0.07  |
| Stainless steel, electropolished | 0.10  | 0.07  |
| Stainless steel + Al foil        | 0.05  | 0.01  |
| Aluminium, as found              | 0.12  | 0.07  |
| Aluminium, mech. polished        | 0.10  | 0.06  |
| Aluminium, electropolished       | 0.08  | 0.04  |
| Copper, as found                 | 0.12  | 0.06  |
| Copper, mech. Polished           | 0.06  | 0.02  |

## Residual gas conduction



$\lambda_{molecule}$  : mean free path of gas molecules

Best would be to avoid residual gas ...

- Viscous regime

- At high gas pressure  $\lambda_{molecule} \ll d$
- Classical conduction  $Q_{res} = k(T) A dT/dx$
- Thermal conductivity  $k(T)$  independent of pressure

- Molecular regime

- At low gas pressure  $\lambda_{molecule} \gg d$
- Kennard's law  $Q_{res} = A \alpha(T) \Omega P (T_2 - T_1)$
- Conduction heat transfer proportional to pressure, independent of spacing between surfaces  
 $\Omega$  depends on gas species
- Accommodation coefficient  $\alpha(T)$  depends on gas species,  $T_1$ ,  $T_2$ , and geometry of facing surfaces

# Multi-layer insulation (MLI)



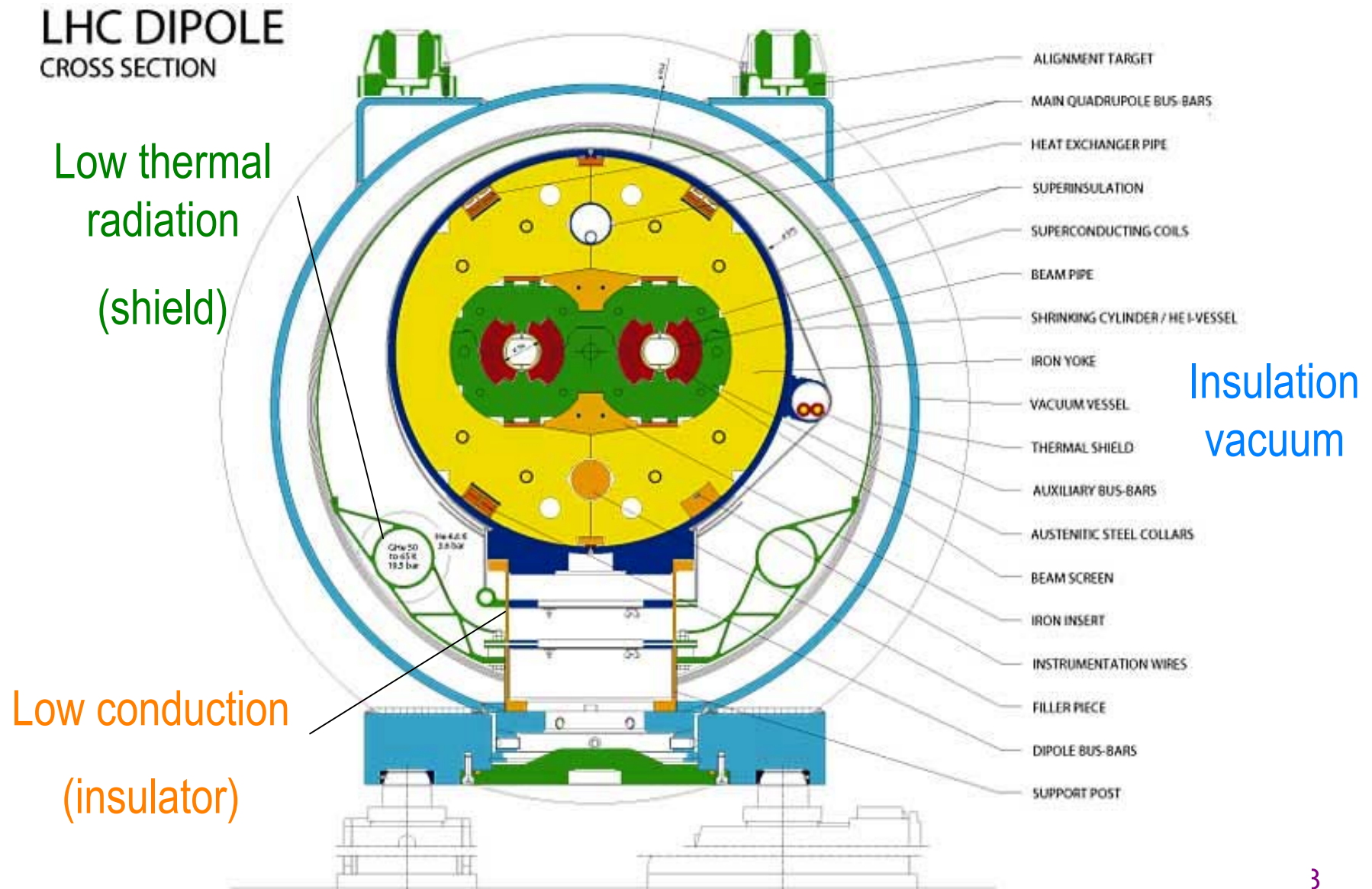
- Complex system involving three heat transfer processes
  - $Q_{MLI} = Q_{rad} + Q_{sol} + Q_{res}$
  - With  $n$  reflective layers of equal emissivity,  $Q_{rad} \sim 1/(n+1)$
  - Due to parasitic contacts between layers,  $Q_{sol}$  increases with layer density
  - $Q_{res}$  due to residual gas trapped between layers, scales as  $1/n$  in molecular regime
  - Non-linear behaviour requires layer-to-layer modeling
- In practice
  - Typical data available from (abundant) literature
  - Measure performance on test samples



## Typical heat fluxes at vanishingly low temperature between flat plates [W/m<sup>2</sup>]

|  |       |                 |
|--|-------|-----------------|
| Black-body radiation from 290 K                  | 401   | Thermal shields |
| Black-body radiation from 80 K                   | 2.3   |                 |
| Gas conduction (100 mPa He) from 290 K           | 19    | Degraded vacuum |
| Gas conduction (1 mPa He) from 290 K             | 0.19  |                 |
| Gas conduction (100 mPa He) from 80 K            | 6.8   |                 |
| Gas conduction (1 mPa He) from 80 K              | 0.07  |                 |
| MLI (30 layers) from 290 K, pressure below 1 mPa | 1-1.5 | Super isolation |
| MLI (10 layers) from 80 K, pressure below 1 mPa  | 0.05  |                 |
| MLI (10 layers) from 80 K, pressure 100 mPa      | 1-2   |                 |

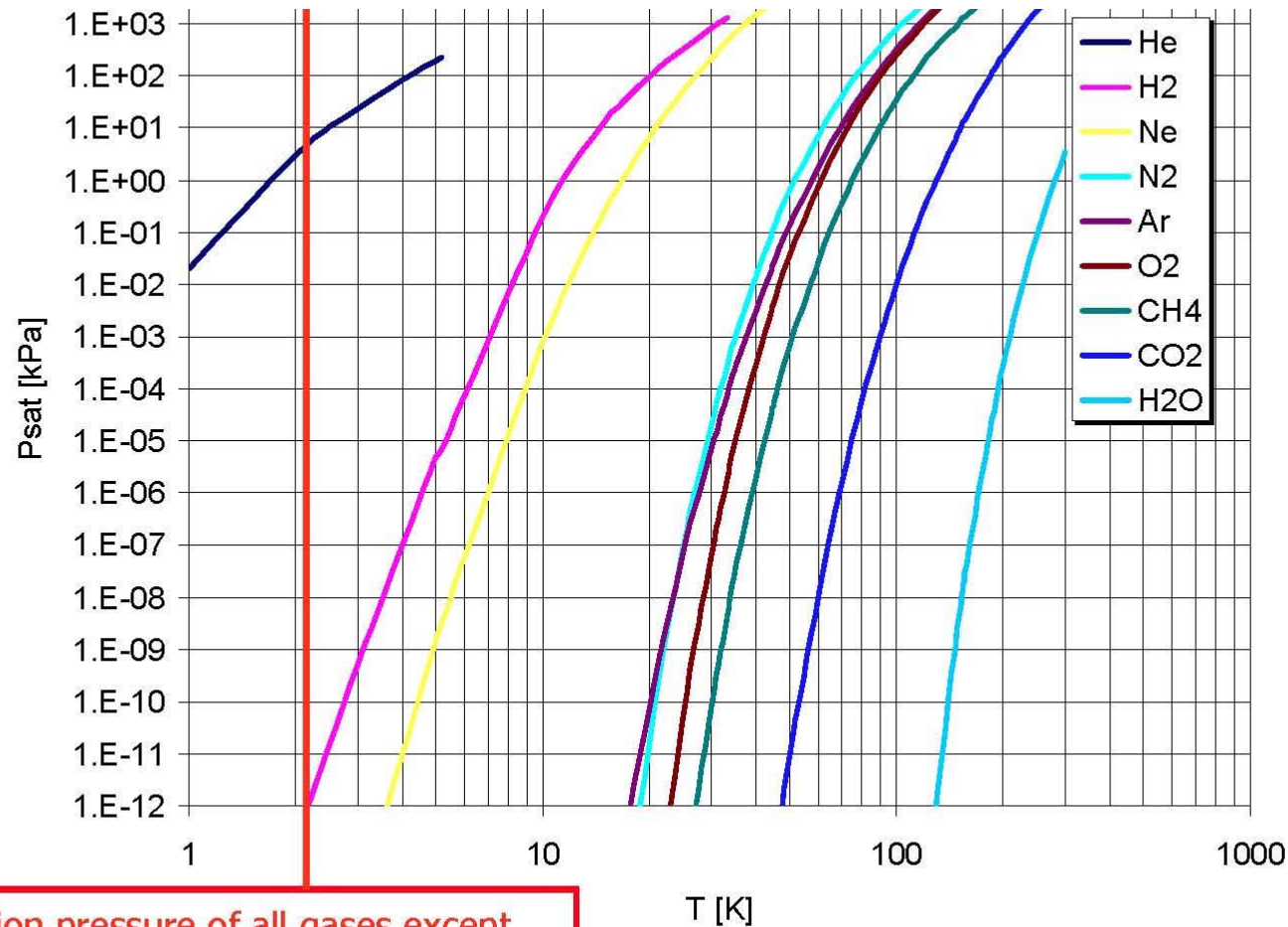
# Cross section of a LHC dipole





## Cryopumping maintains good vacuum

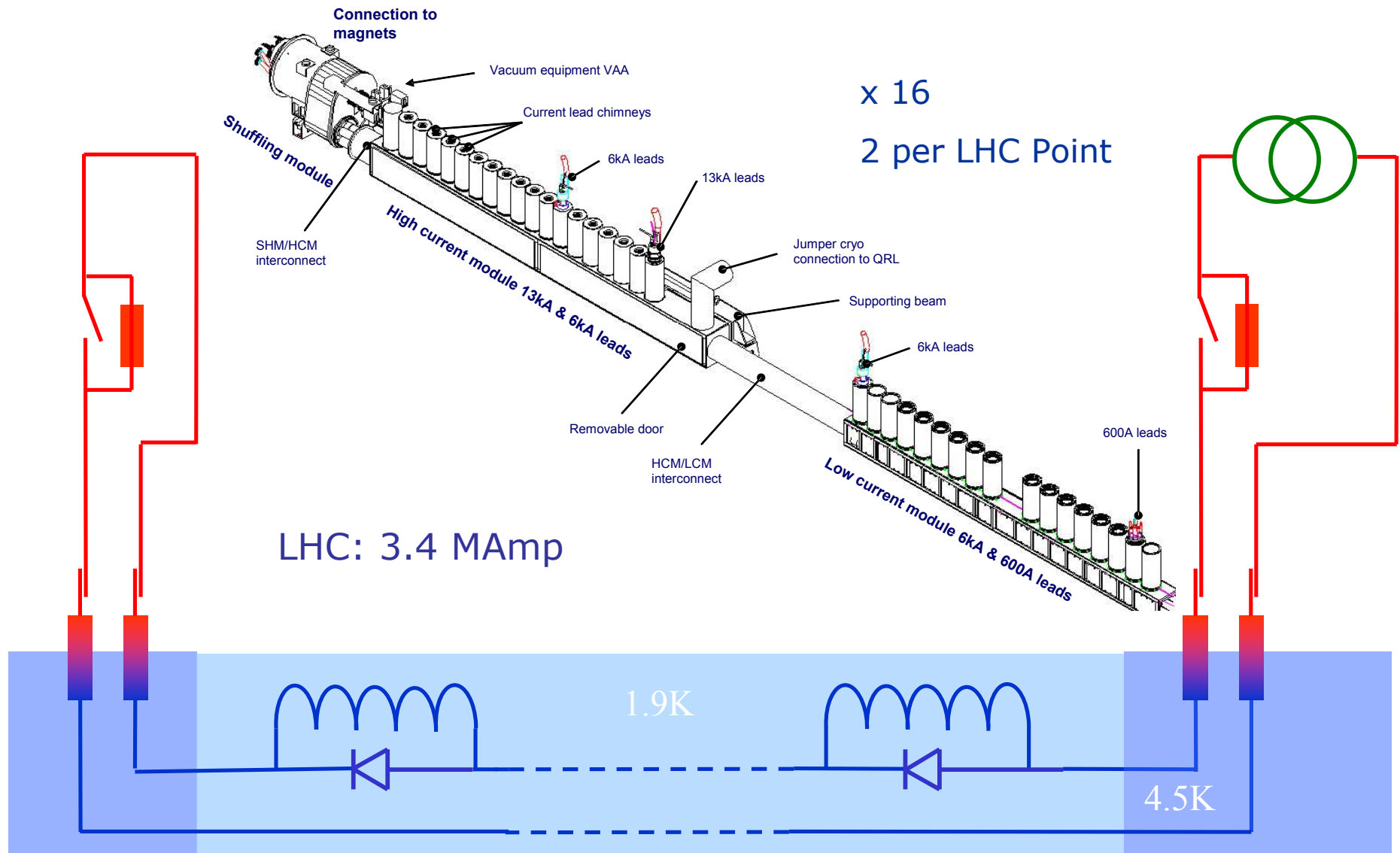
### Vapour pressure at cryogenic temperatures



Saturation pressure of all gases except helium vanish at cryogenic temperature

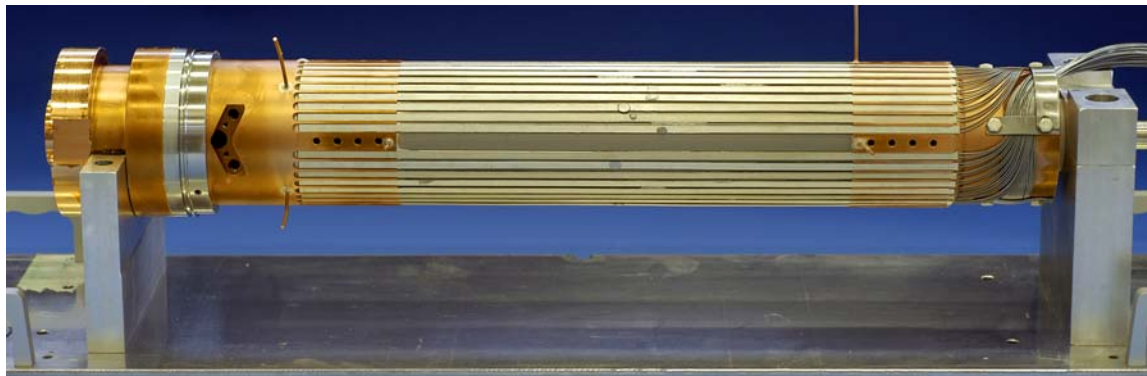
Cryopumping maintains good vacuum

# Electrical Feed Box for current leads



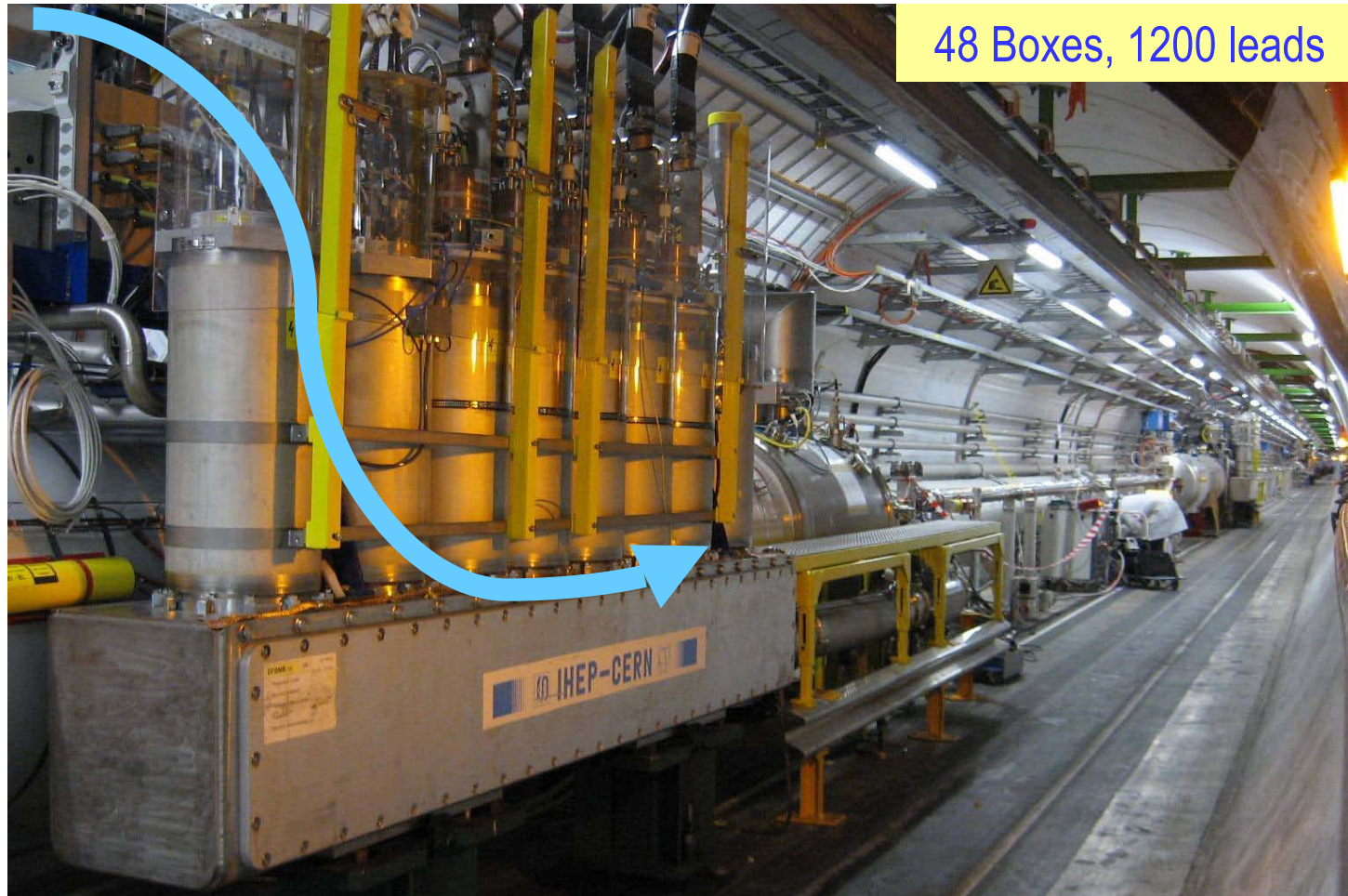
## Beating the WFL law: HTS current leads

- The WFL law essentially states that good electrical conductors are also good thermal conductors
- Current leads need good electrical conductors with low thermal conductivity
- Superconductors are bad thermal conductors with zero resistivity
- Build current lead with superconductor up to temperature as high as possible, i.e. HTS





## Electrical feed boxes for current leads



48 Boxes, 1200 leads

More than 10'000 Amperes per chimney, from room temperature down to 4.5K in about a meter



## HTS vs. normal conducting current leads

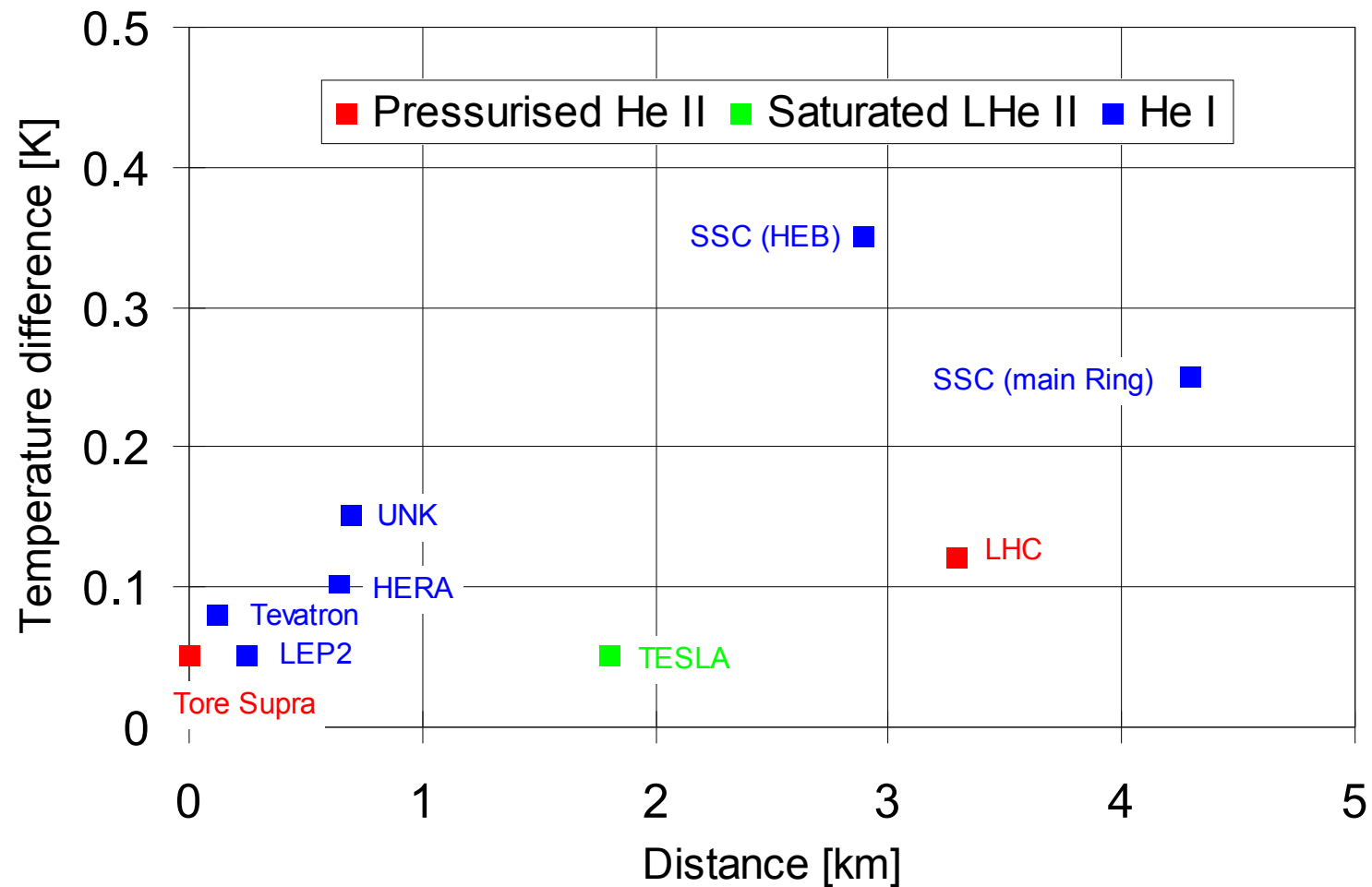
| Type                              | Resistive | HTS (4 to 50 K)<br>Resistive (above) |
|-----------------------------------|-----------|--------------------------------------|
| Heat into LHe [W/kA]              | 1.1       | 0.1                                  |
| Total exergy consumption [W/kA]   | 430       | 150                                  |
| Electrical power from grid [W/kA] | 1430      | 500                                  |

For LHC, using HTS allowed to save the equivalent of 1 large 18kW@4.5K refrigerator !

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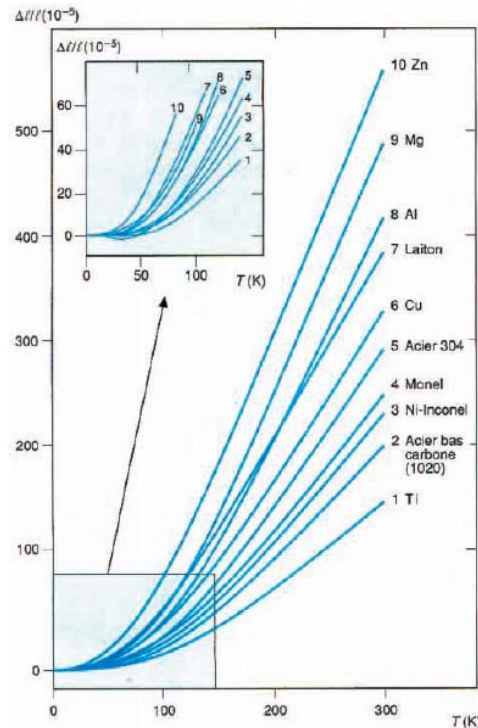
# Transport of refrigeration in large distributed cryogenic systems



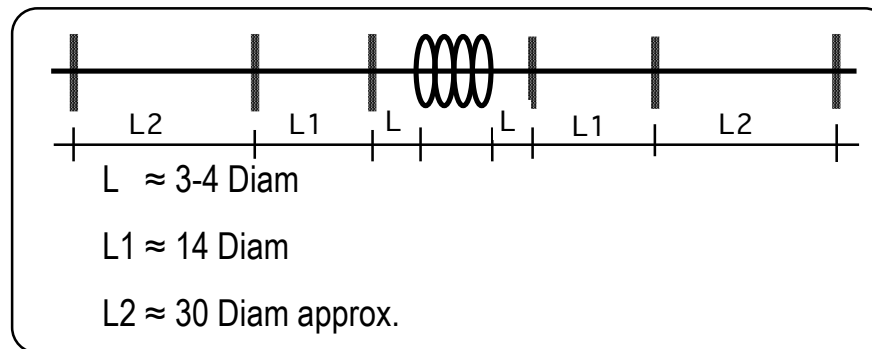
# Cryogenic distribution scheme: design issues

- Monophase vs. two-phase
  - temperature control
  - hydrostatic head & flow instabilities
- Pumps vs. no pumps
  - efficiency & cost
  - reliability & safety
- LN<sub>2</sub>
  - cooldown and/or normal operation
  - capital & operating costs of additional fluid
  - safety in underground areas (ODH)
- Lumped vs. distributed cryoplants
- Separate cryoline vs. integrated piping
- Number of active components (valves, actuators)
- Redundancy of configuration

# Thermal contraction for cryo lines



3 mm / m of thermal contraction  
=> Compensation required !

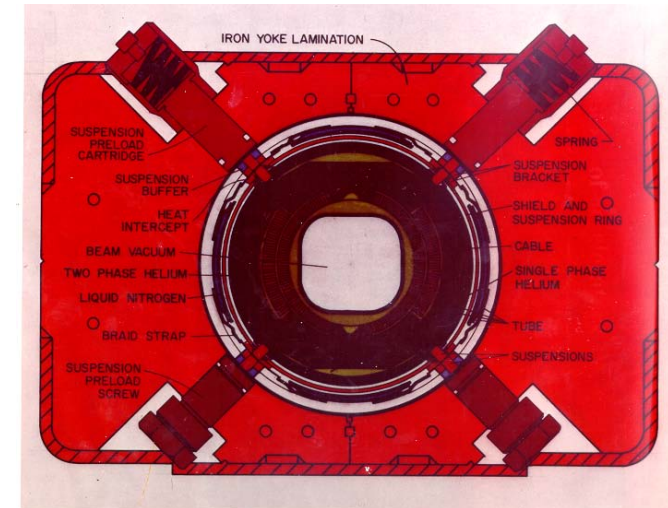
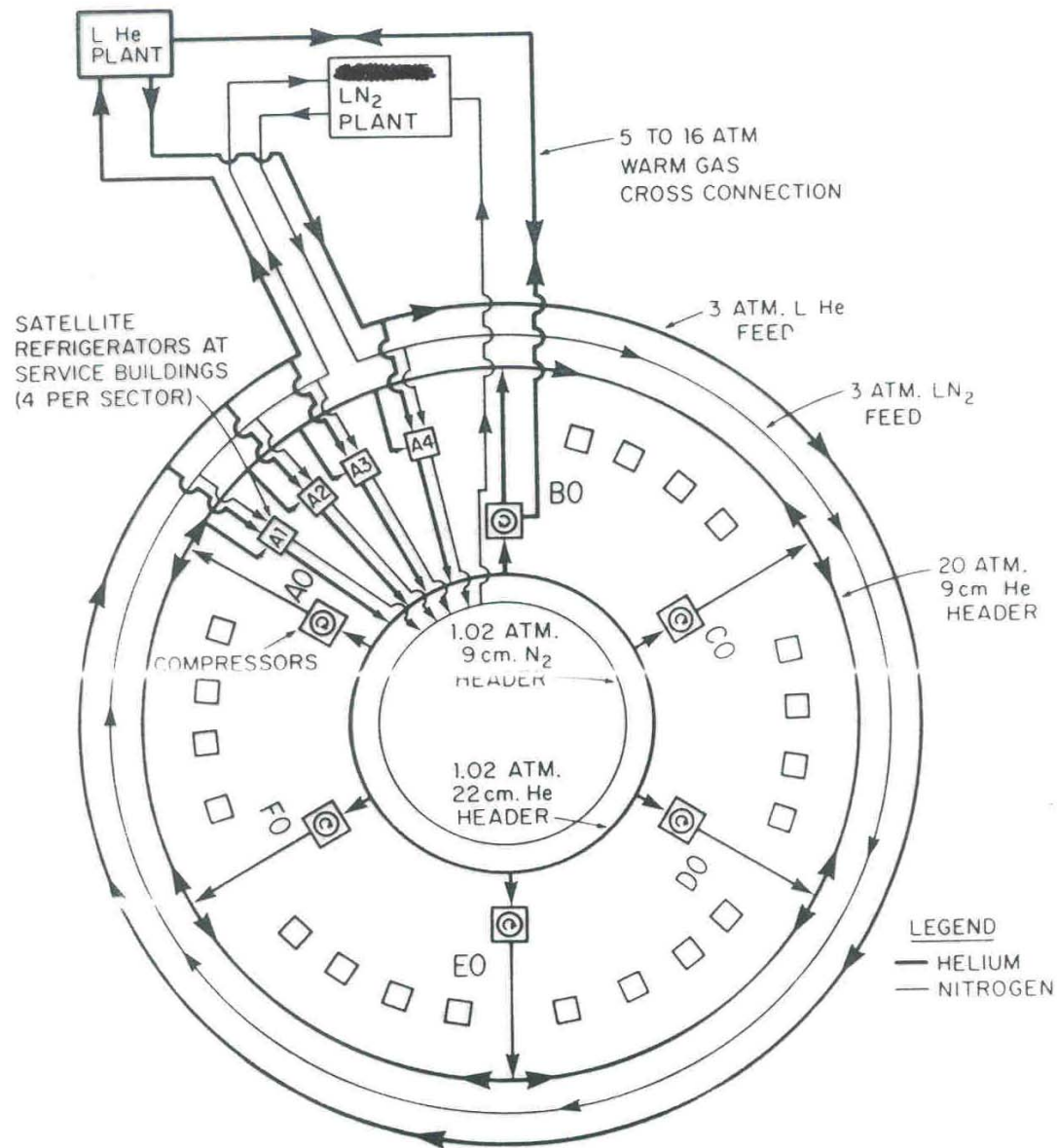


Thermal compensation



This is **THE** delicate part in the design of a cryogenic line, as thermal performance can only be considered once the line withstand mechanical forces !!!

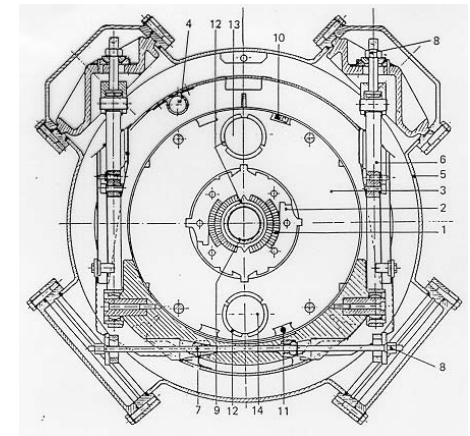
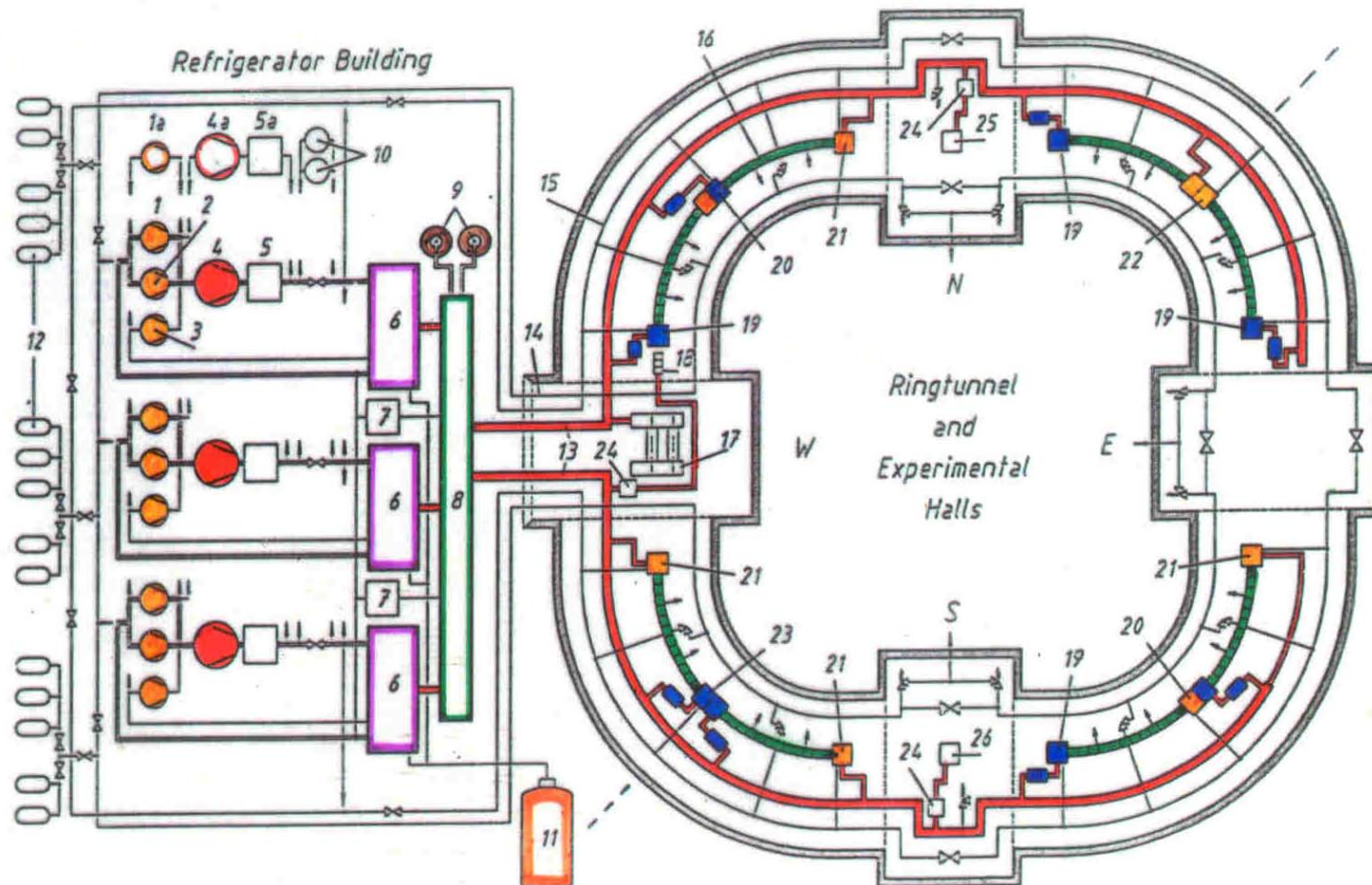
# Tevatron distribution scheme



Central helium liquefier,  
separate ring cryoline  
and satellite  
refrigerators



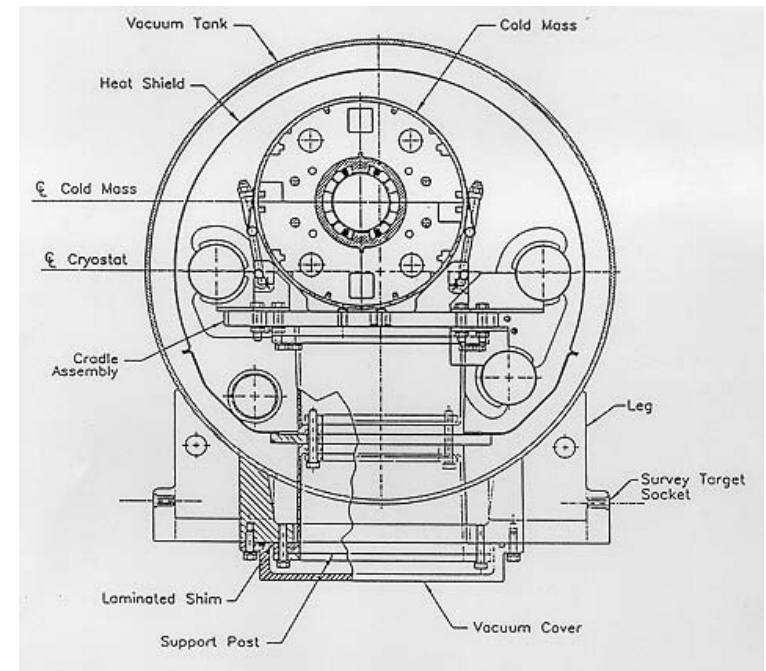
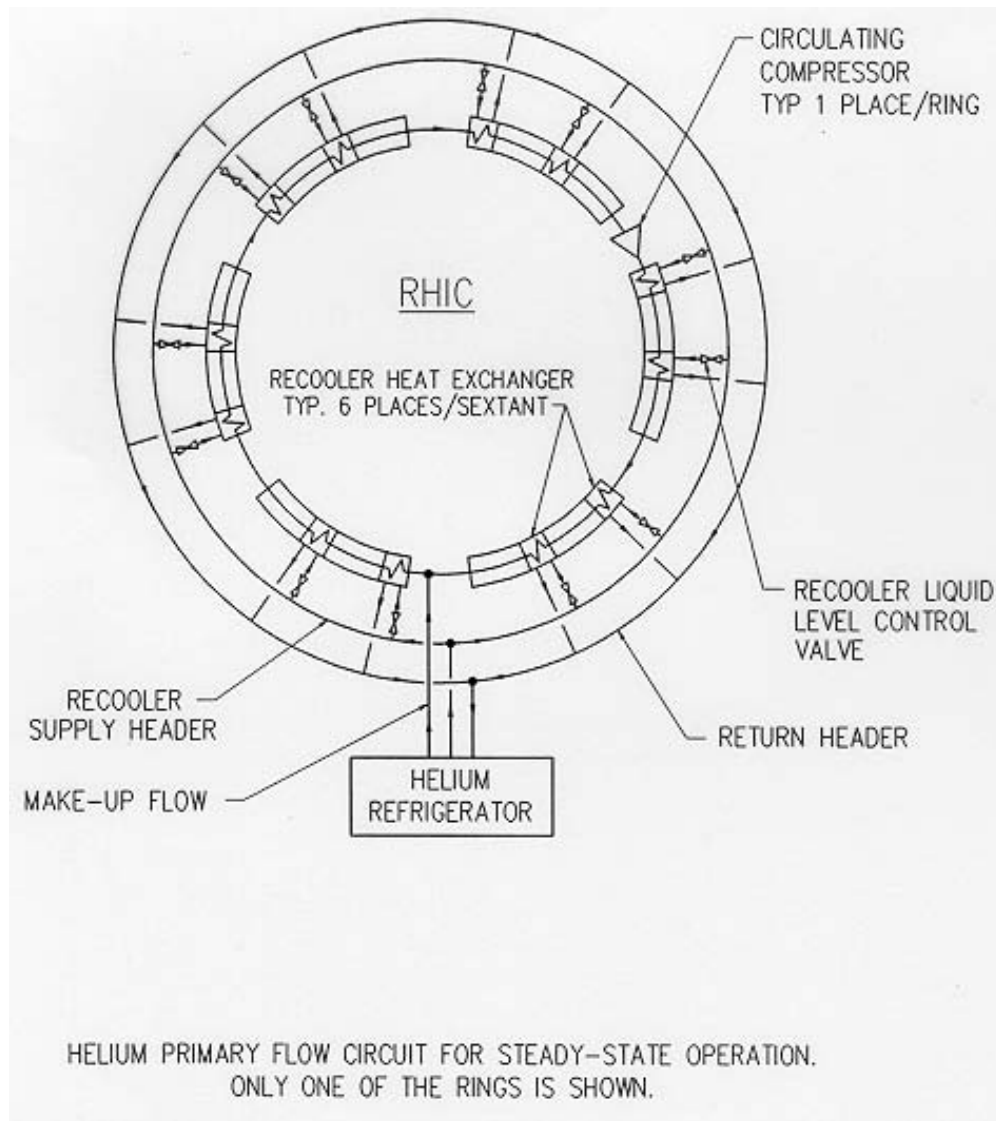
# HERA distribution scheme



Central  
cryoplane and  
separate ring  
cryoline

|                       |                            |                           |                         |
|-----------------------|----------------------------|---------------------------|-------------------------|
| Refrigeration 4.3 K   | 6775 W                     | total mass flow           | 0.871 kg/s              |
| Refrigeration 40/80 K | 20000 W                    | Primary power             | 2845 kW                 |
| Current lead flow     | $20.5 \times 10^{-3}$ kg/s | Specif. power consumption | 281 W (300 K)/W (4.3 K) |

# RHIC distribution scheme

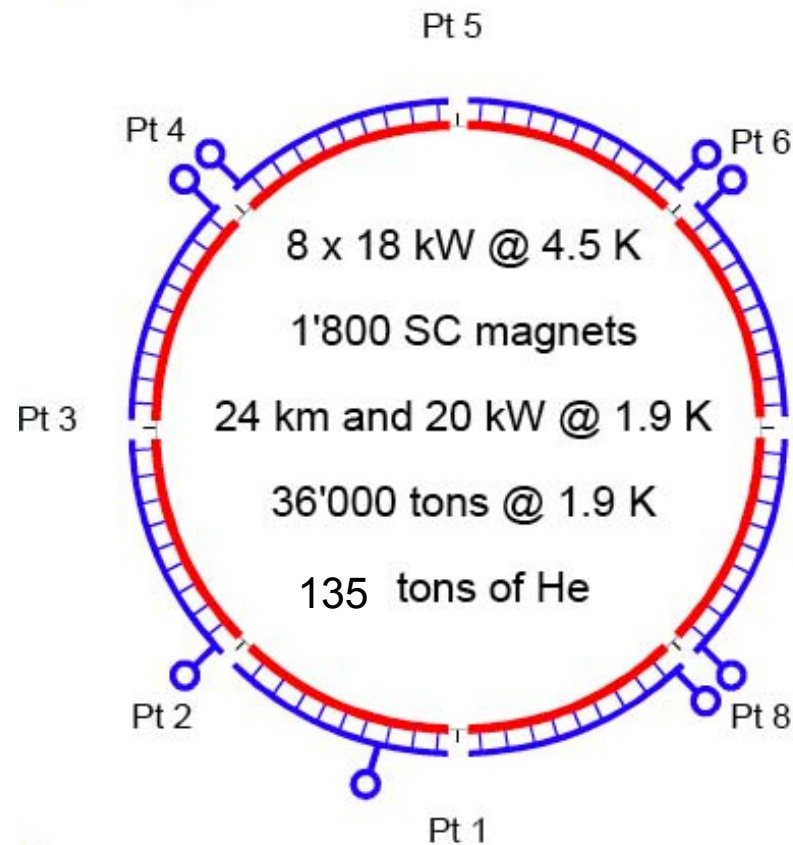


Central cryoplant and piping integrated in magnet cryostat



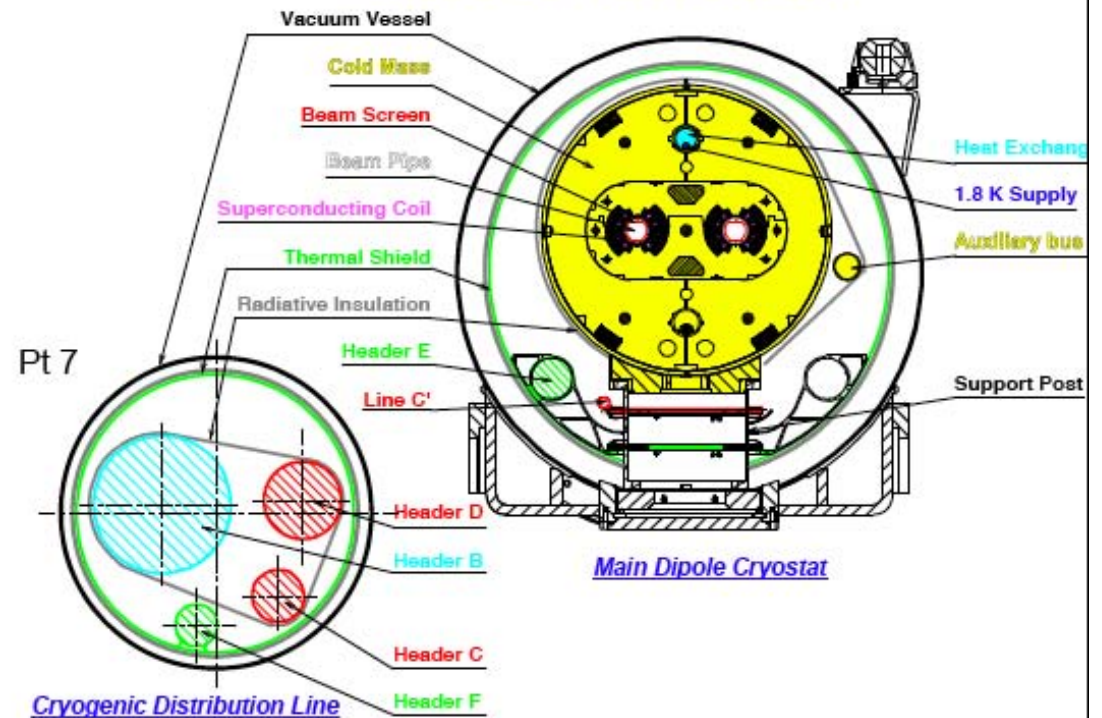


# LHC distribution scheme



○ Cryogenic plant

## Typical LHC Cross-section



Cryoplants at five points,  
separate ring cryoline,  
107 m long strings

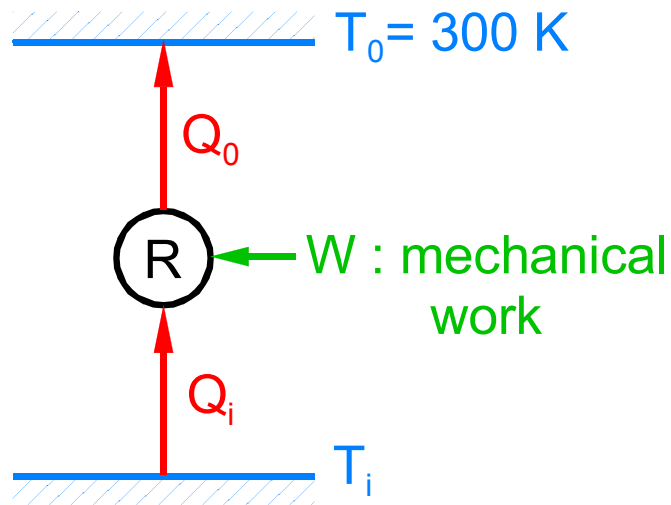
# Superconducting Linac (Tesla\_based)



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# Thermodynamics of cryogenic refrigeration



First principle [Joule]

$$Q_0 = Q_i + W$$

Second principle [Clausius]

$$\frac{Q_0}{T_0} \geq \frac{Q_i}{T_i}$$

(= for reversible process)

Hence,  $W \geq T_0 \cdot \frac{Q_i}{T_i} - Q_i$  which can be written in three different ways:

①  $W \geq T_0 \cdot \Delta S_i - Q_i$  introducing **entropy S** as

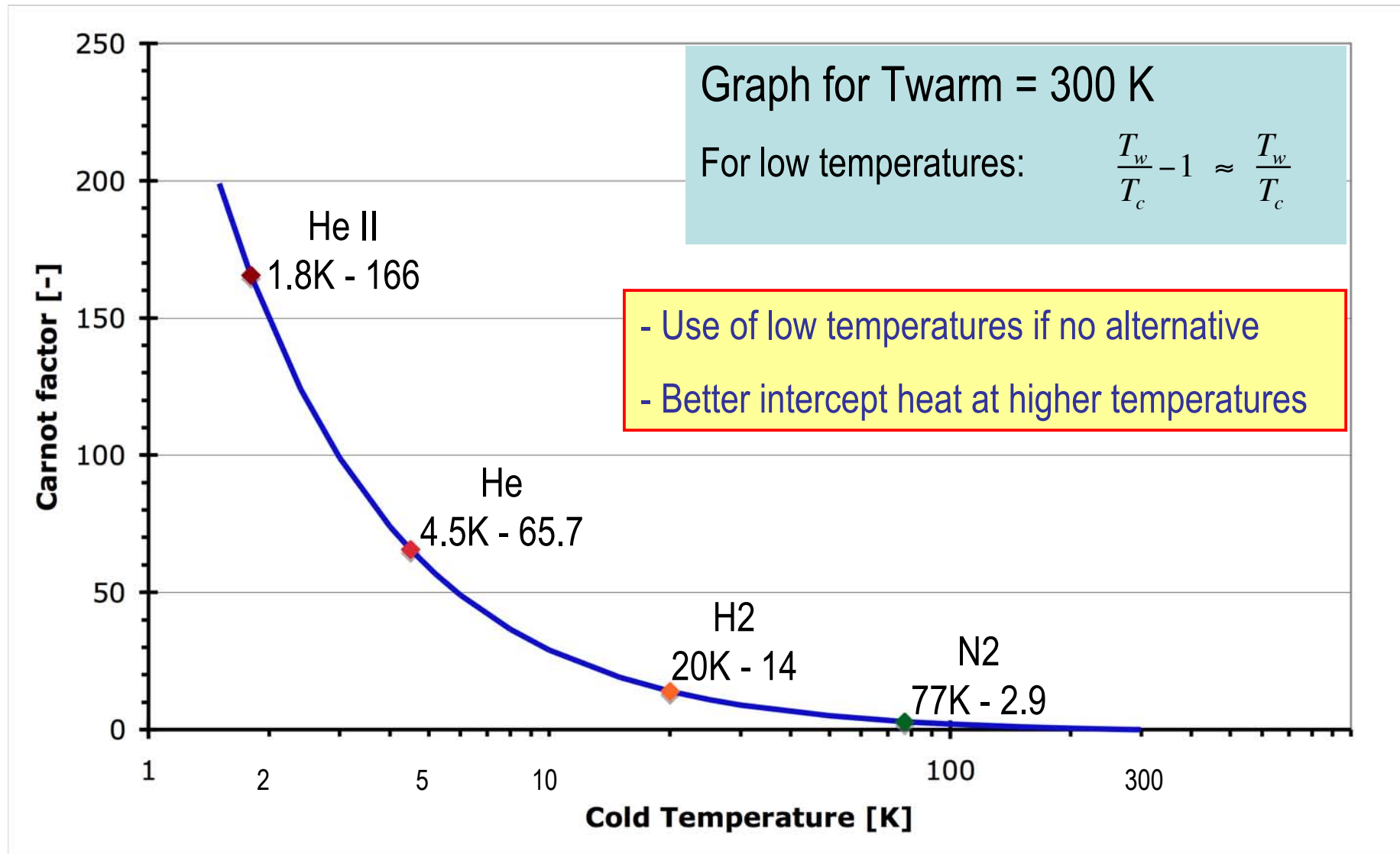
$$\Delta S_i = \frac{Q_i}{T_i}$$

②  $W \geq Q_i \left[ \left( \frac{T_0}{T_i} - 1 \right) \right]$  ← Carnot factor

③  $W \geq \Delta E_i$  introducing **exergy E** as

$$\Delta E_i = Q_i \cdot \left( \frac{T_0}{T_i} - 1 \right)$$

# The Carnot Factor



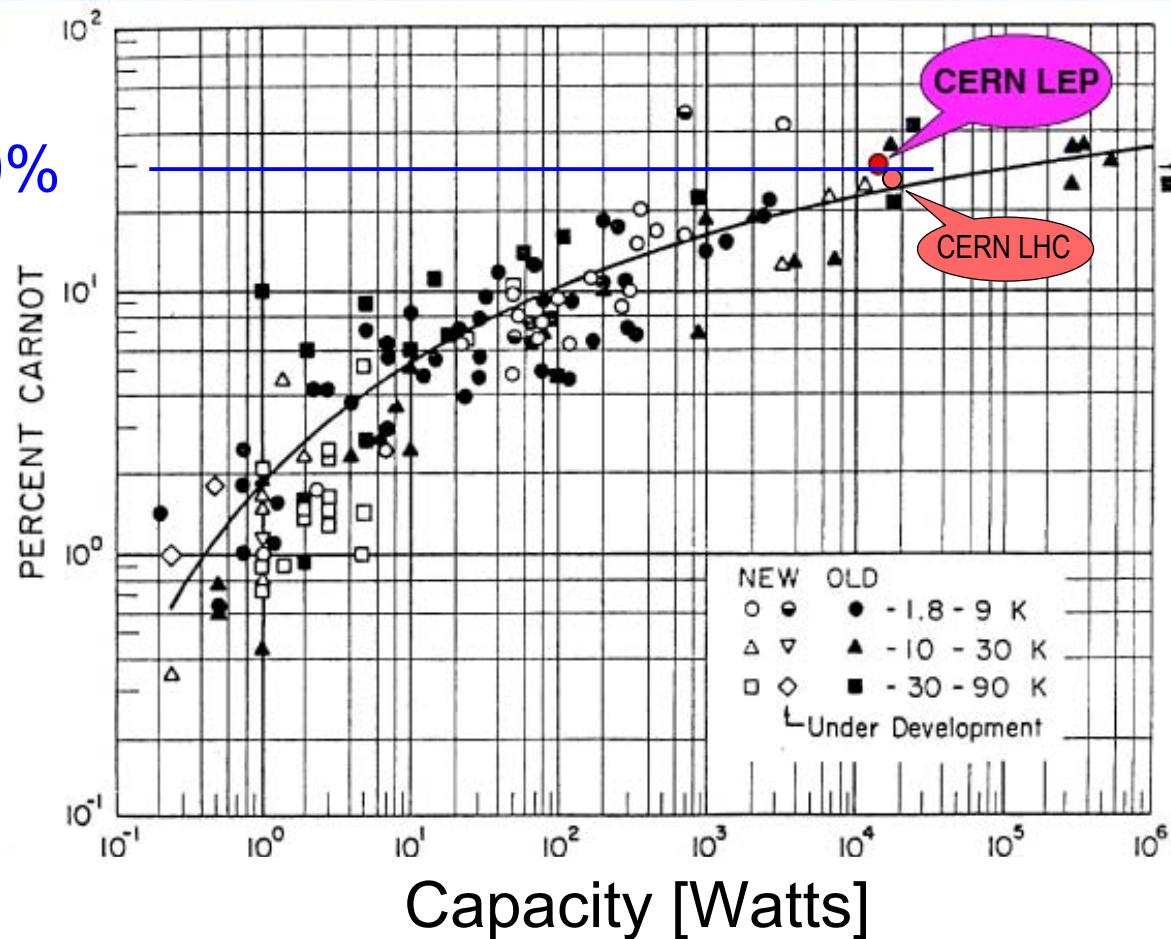


# Helium refrigerators

Power Input  $\approx$  Power@cold x Carnot / %w.r.tCarnot

## LE DIAGRAMME DE STROBRIDGE

30%



The efficiency w.r.t Carnot does not depend on the temperature, but rather on the size

The largest possible, the best !

## Minimum refrigeration work

Consider the extraction of 1 W at 4.5 K, rejected at 300 K  
The minimum refrigeration work (equation 2) is:

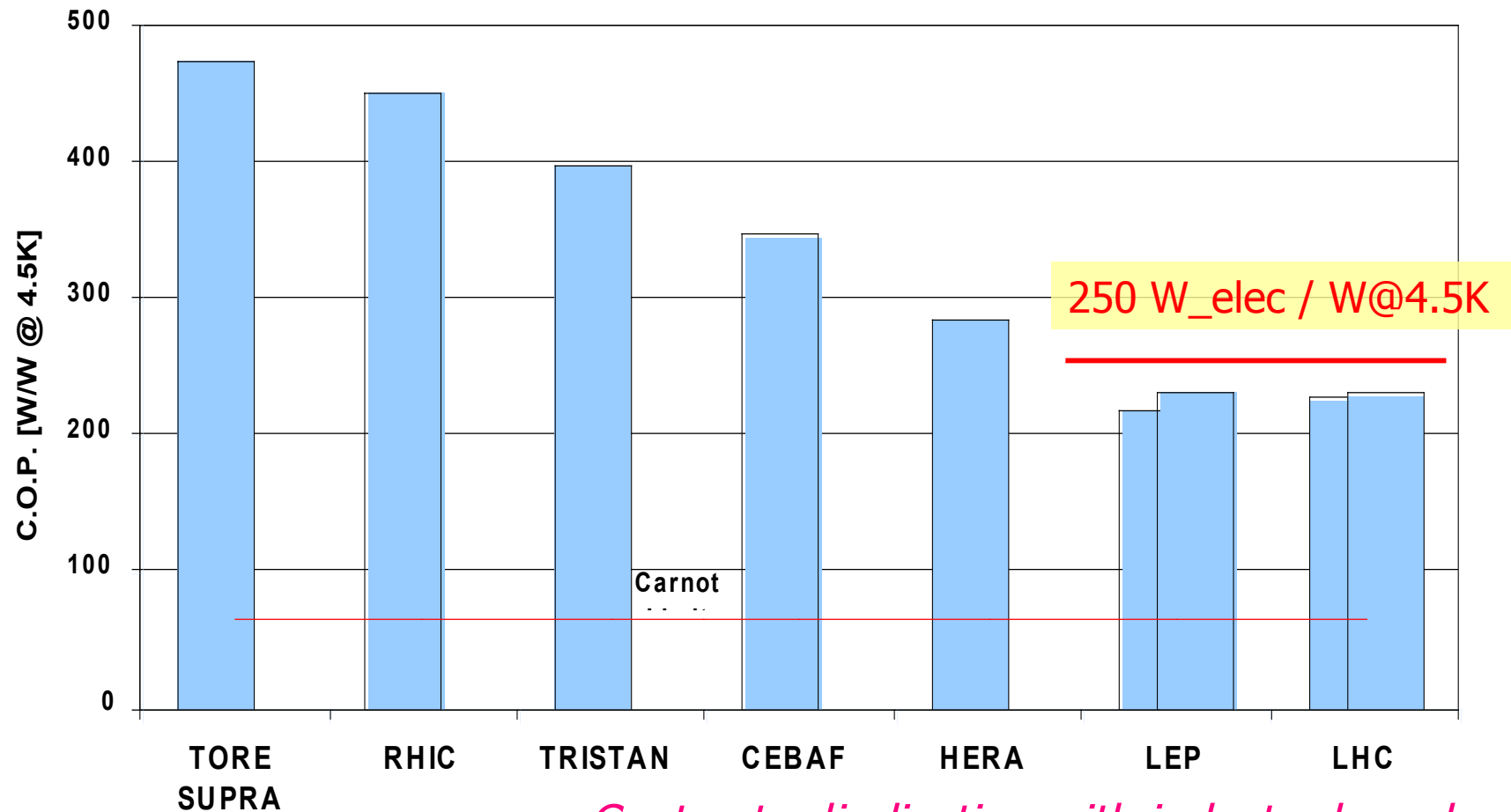
$$W_{\min} = Q_i \cdot \left( \frac{T_0}{T_i} - 1 \right) = 1 \cdot \left( \frac{300}{4.5} - 1 \right) = 65.7 \text{ W}$$

In practice, the most efficient helium refrigerators have an efficiency of about 30% w.r. to the Carnot limit.

$$\Rightarrow W_{\text{real}} = \frac{W_{\min}}{\zeta} = \frac{65.7}{0.3} = 220 \text{ W}$$

# C.O.P. of large cryogenic helium refrigerators

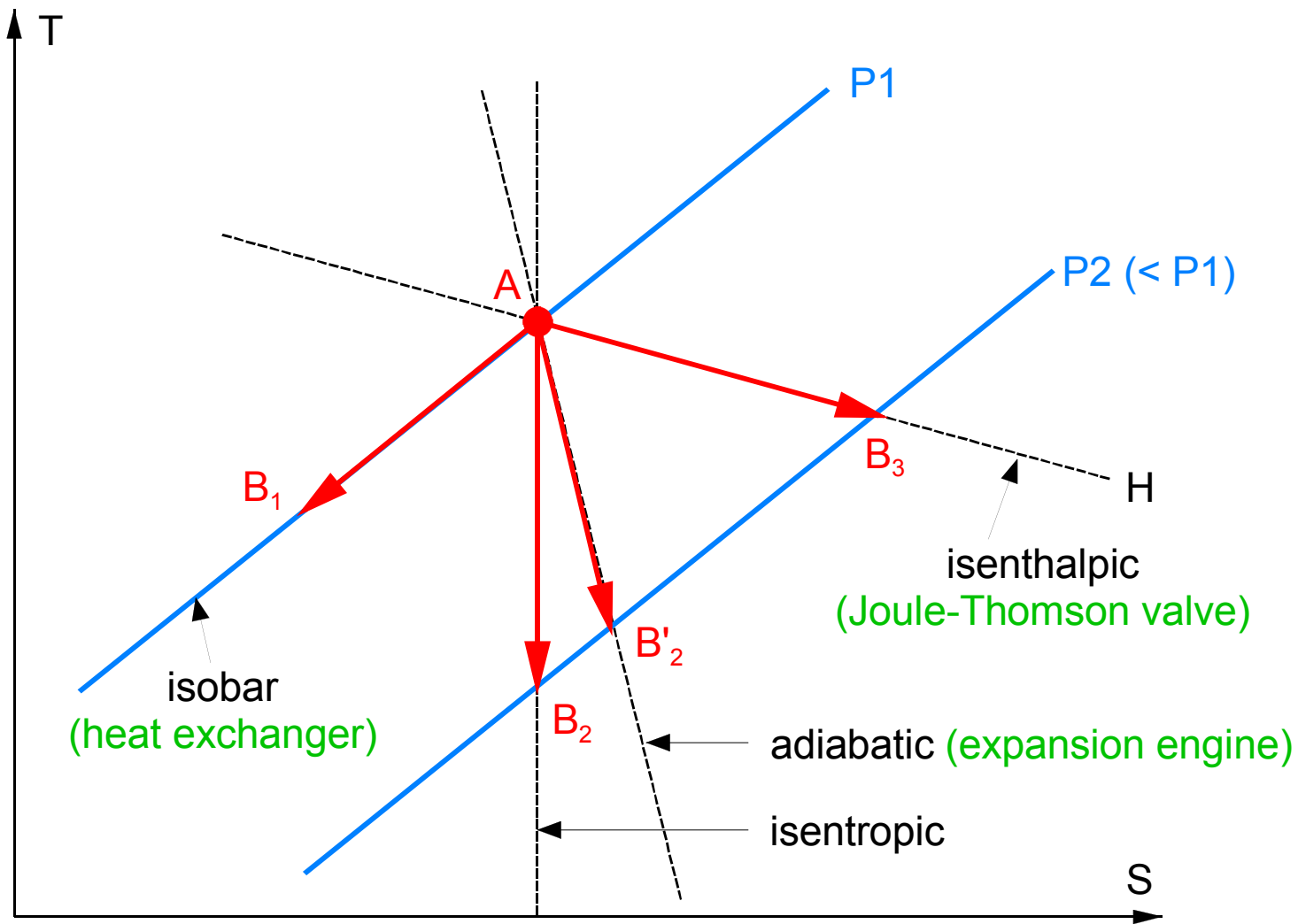
*Time (left to right) is not the only factor for improvement*



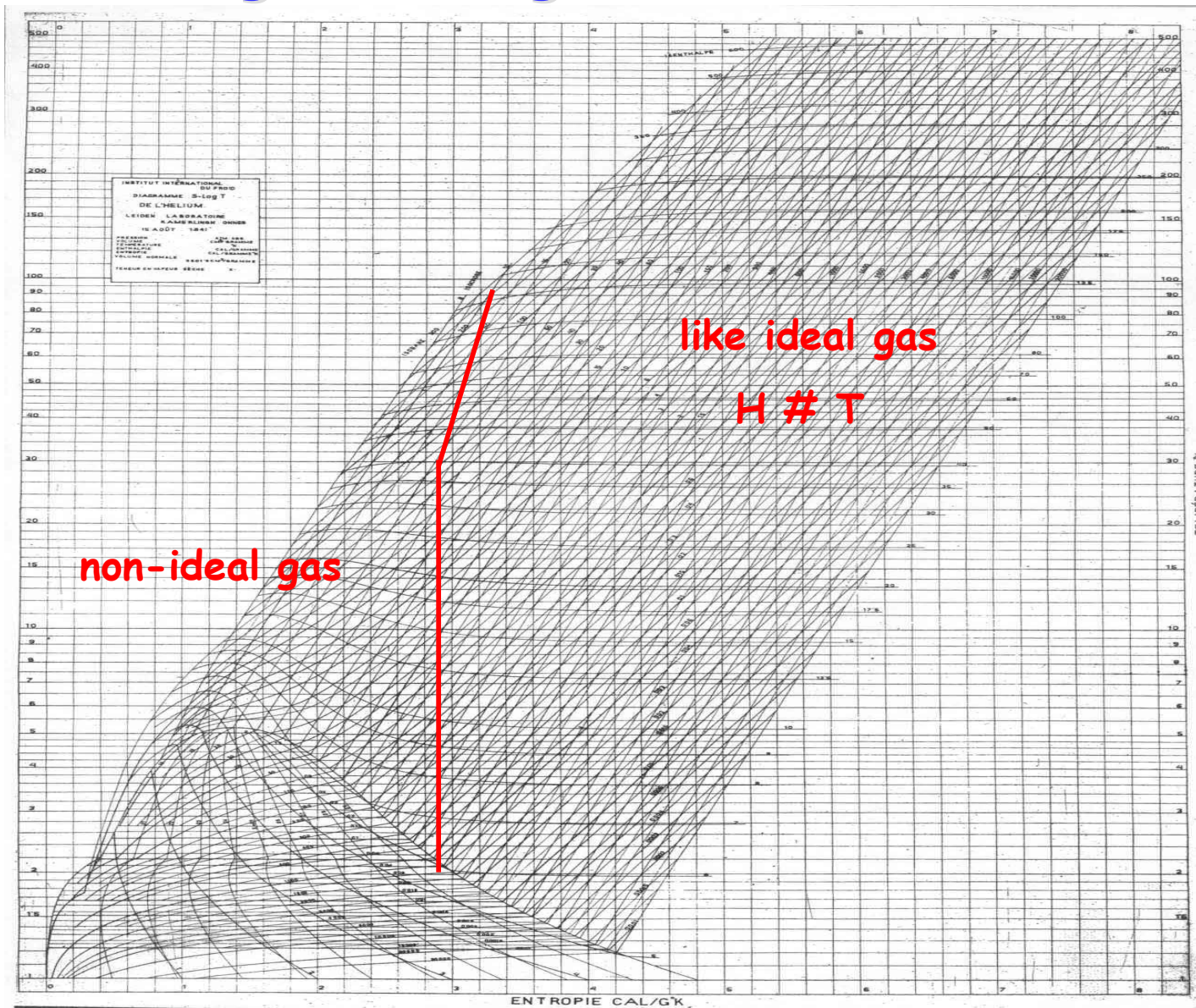
*Contract adjudication with industry based on  
Capital+Operation(10yrs) costs*



# Elementary cooling processes on T-S diagram



# Log T-s Diagram for Helium

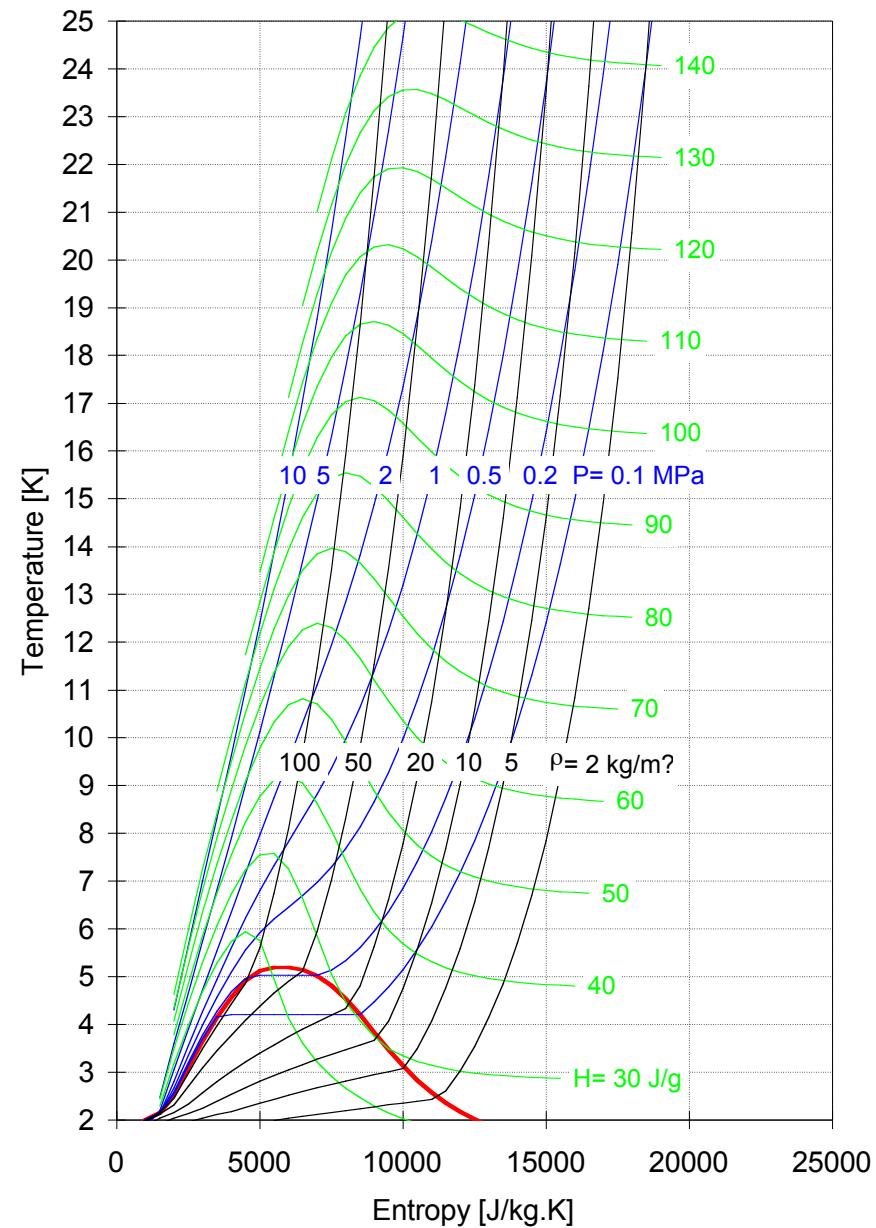


## Maximum Joule-Thomson inversion temperatures

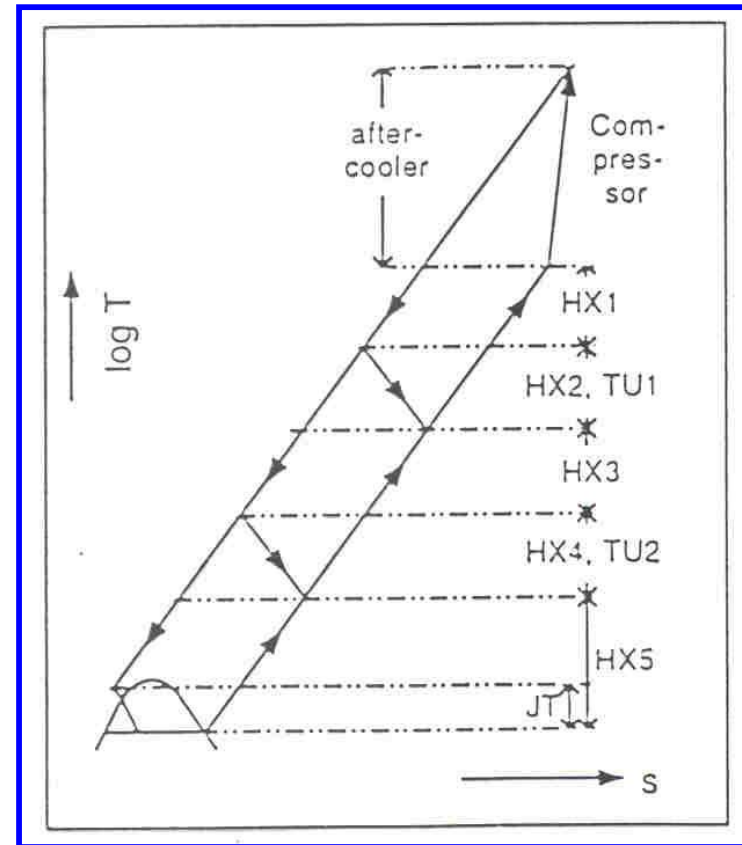
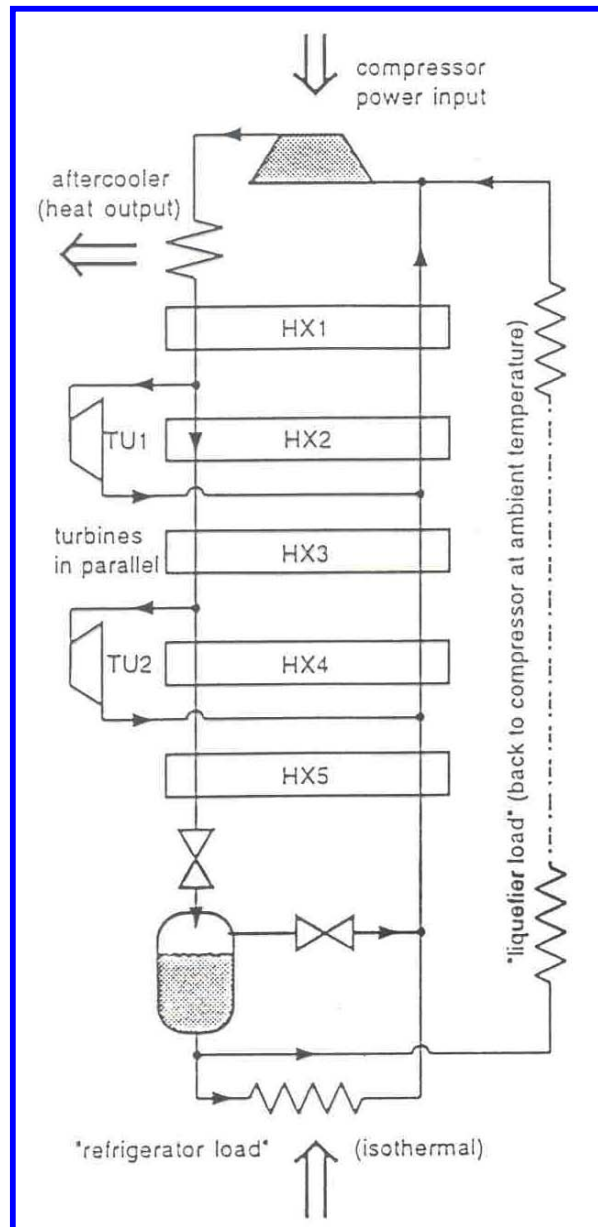
| Cryogen  | Maximum inversion temperature [K] |
|----------|-----------------------------------|
| Helium   | 43                                |
| Hydrogen | 202                               |
| Neon     | 260                               |
| Air      | 603                               |
| Nitrogen | 623                               |
| Oxygen   | 761                               |

While air can be cooled down and liquefied by JT expansion from room temperature, helium and hydrogen need precooling down to below inversion temperature by heat exchange or work-extracting expansion (e.g. in turbines)

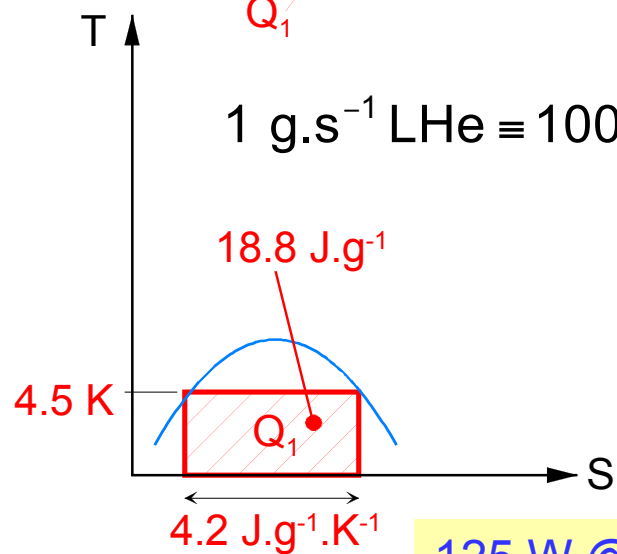
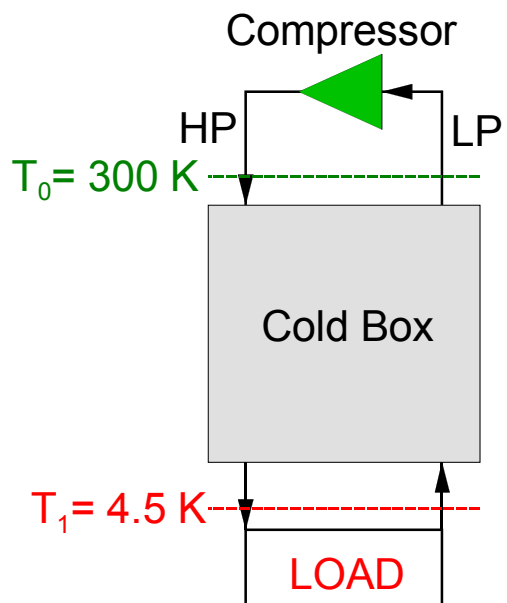
# T-S diagram for helium (non-ideal part)



# Two-stage Claude cycle

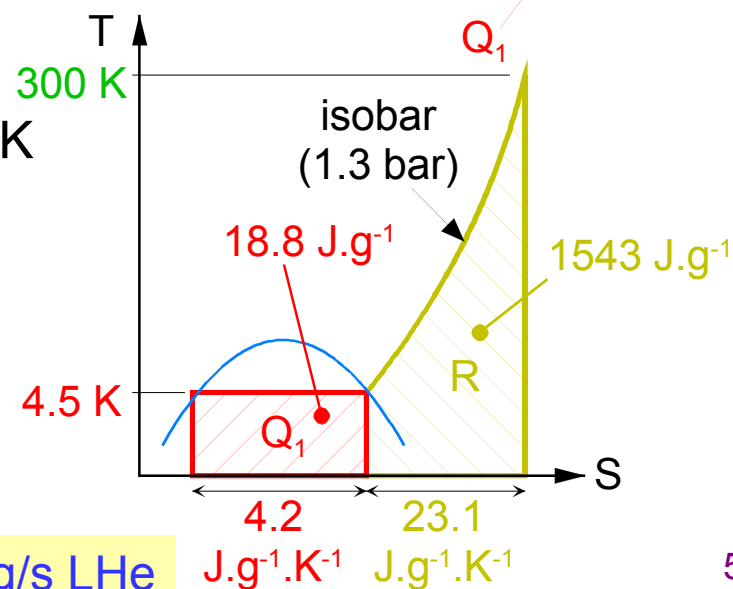
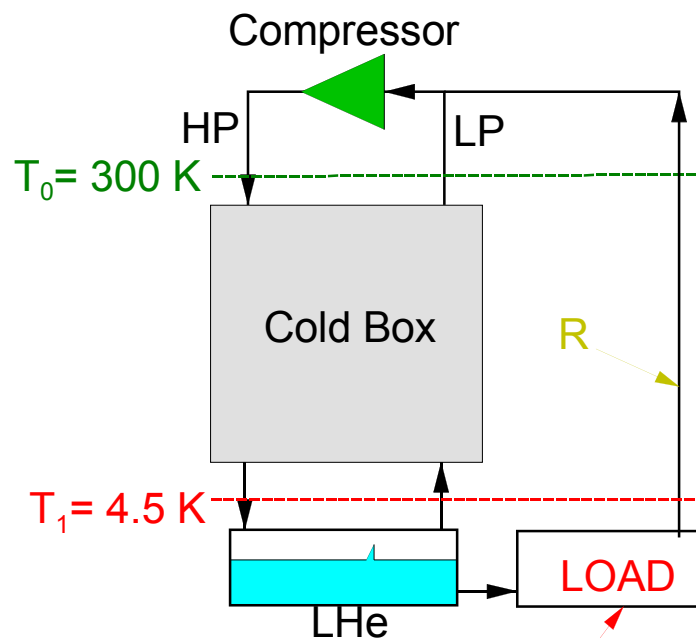


# Refrigerator

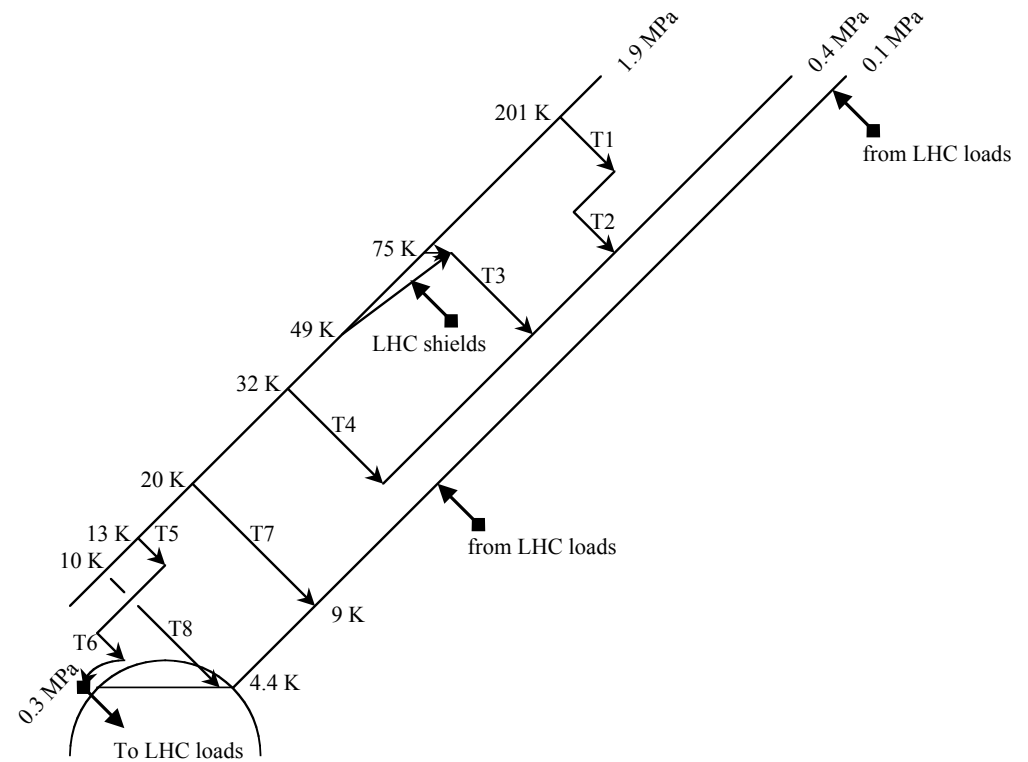
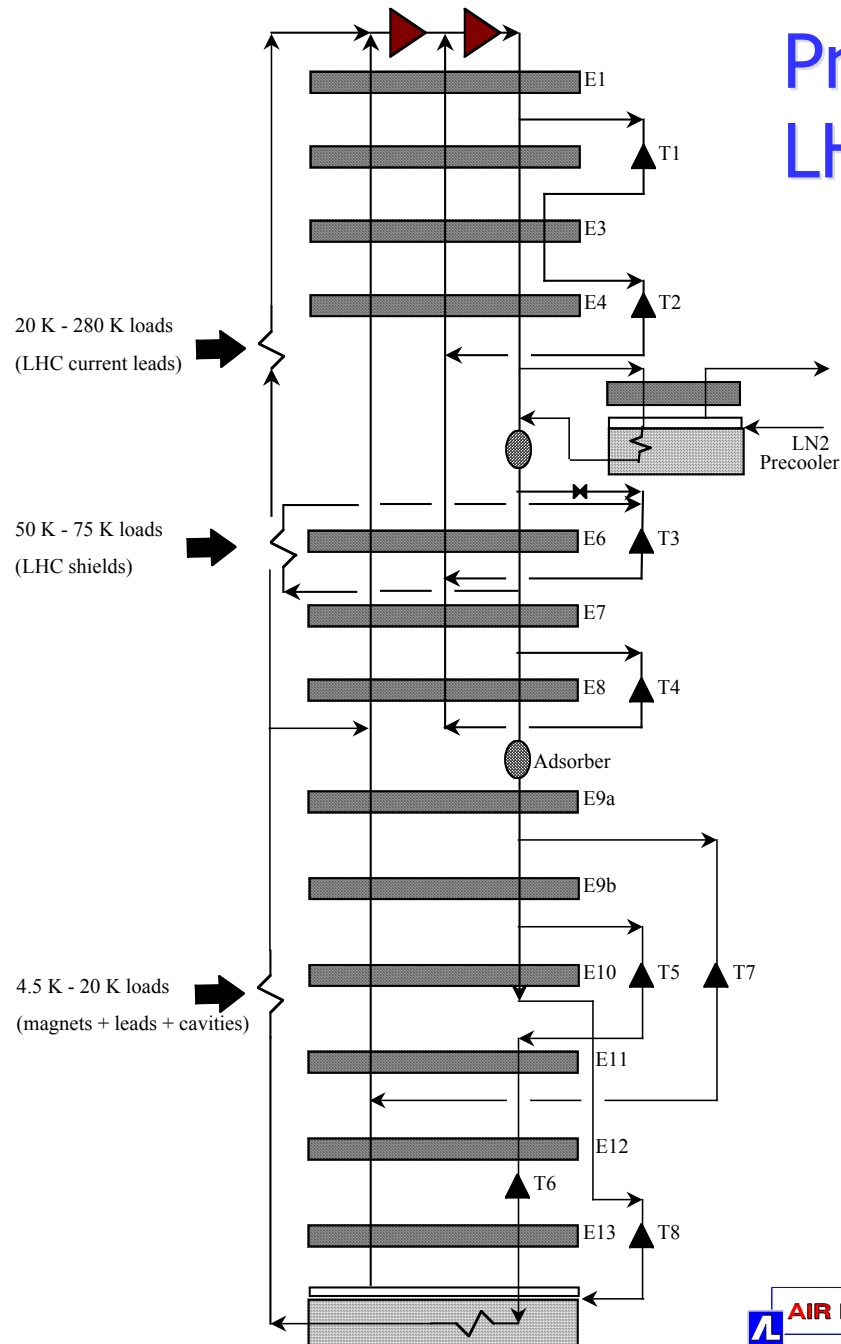


125 W @ 4.5K  $\approx$  1 g/s LHe

# Liquefier



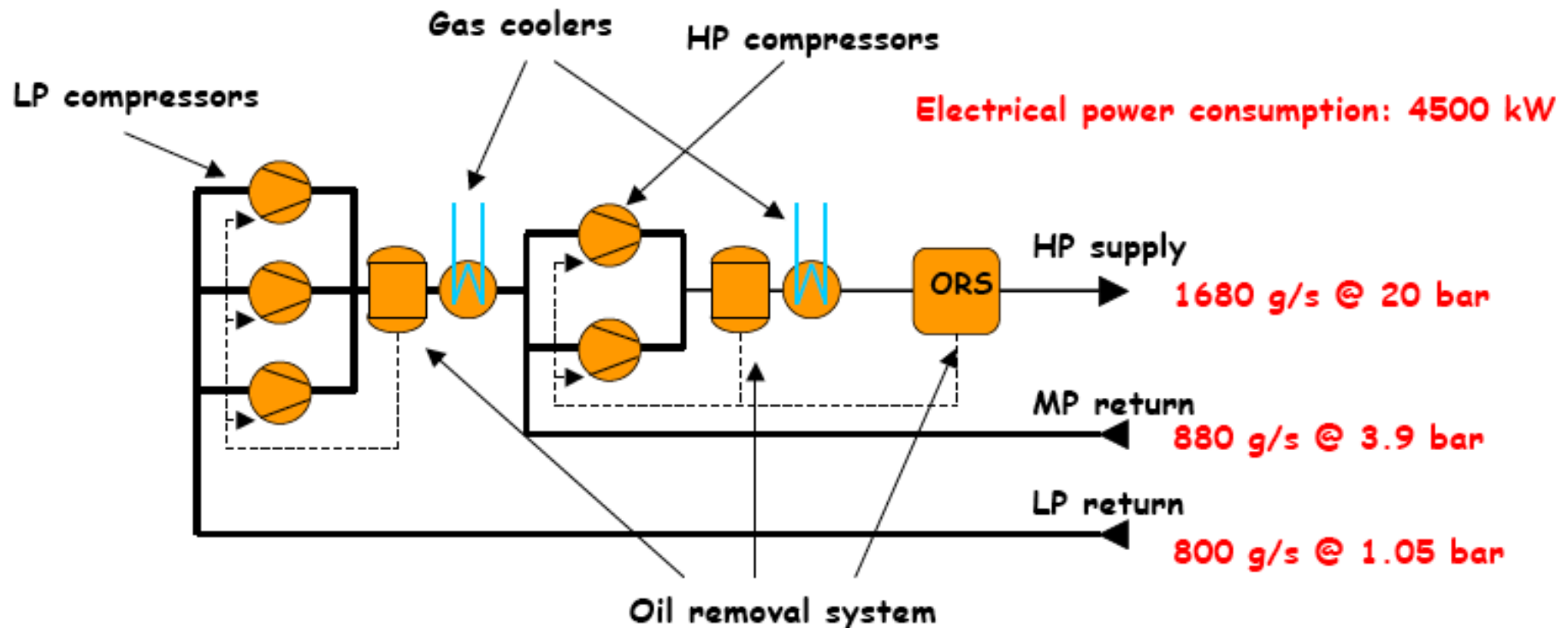
# Process cycle & T-S diagram of LHC 18 kW @ 4.5 K cryoplant





# Process diagram, LHC compressors 18 kW @ 4.5 K

Oil lubricated screw compressors, water cooled, oil separation included



Machine derived from industrial refrigeration (or compressed air)

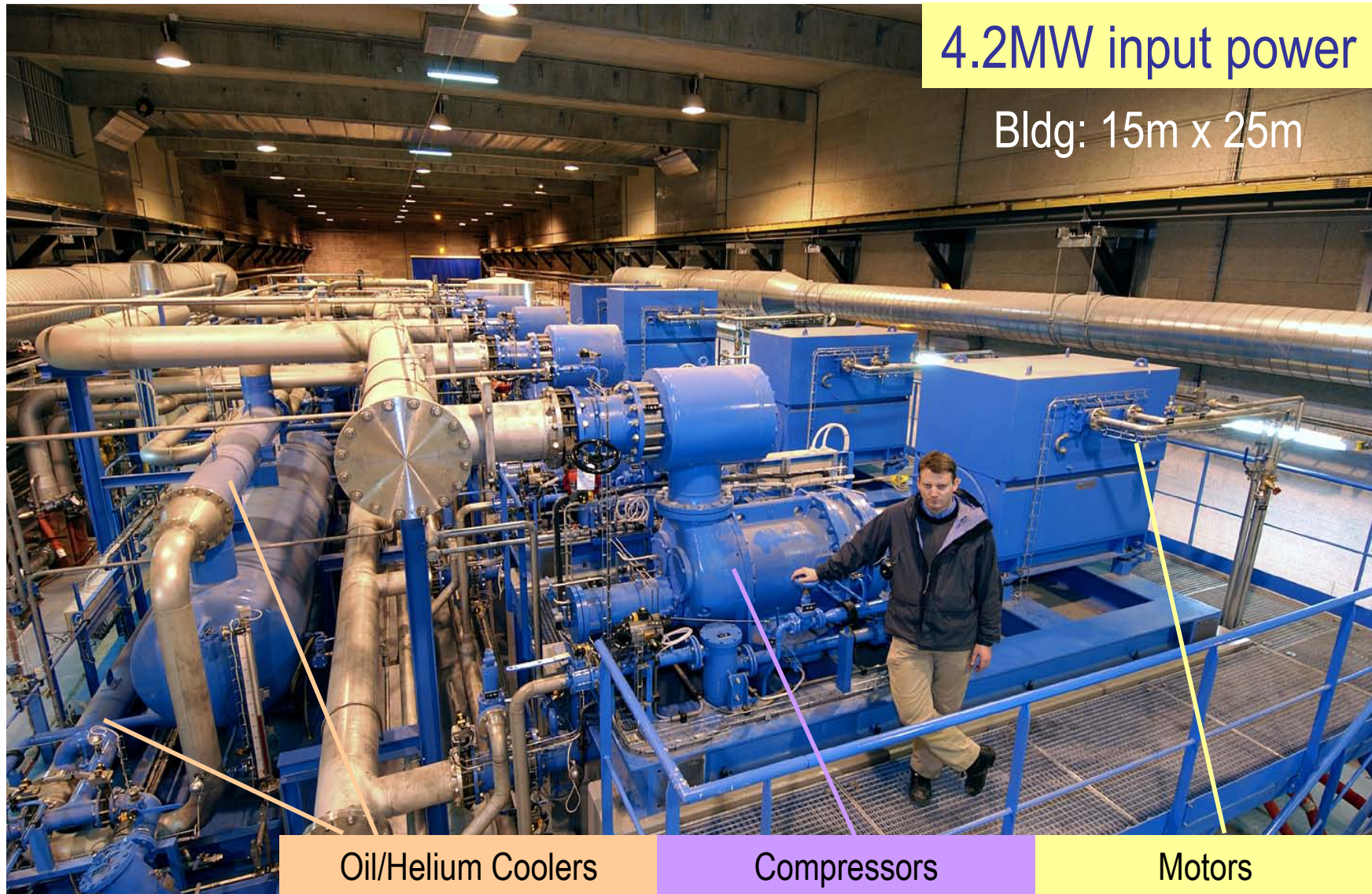
No more piston (high PR, low flow), not yet centrifugal (high flow, low PR)



# Compressor station of LHC 18 kW@ 4.5 K helium refrigerator

4.2MW input power

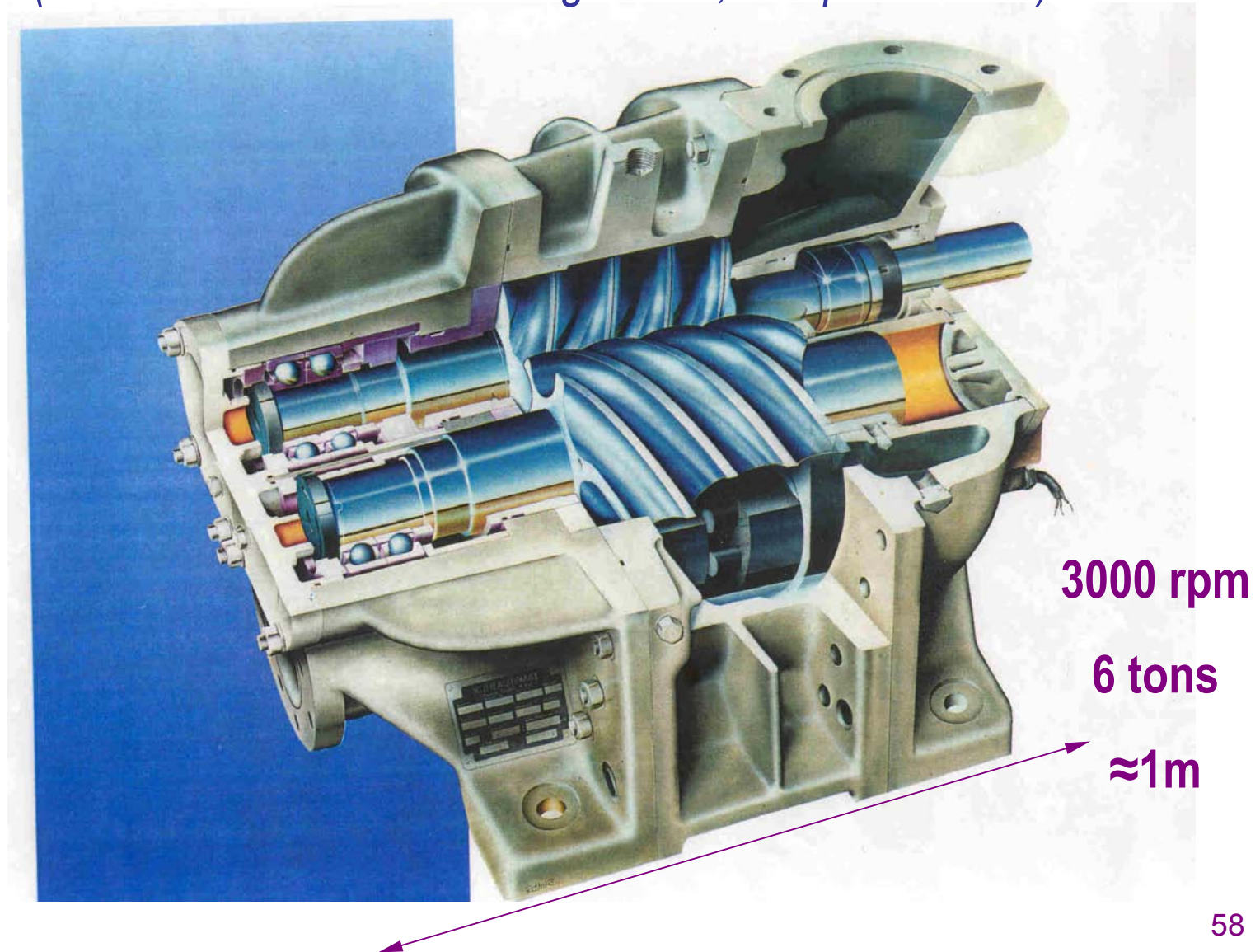
Bldg: 15m x 25m



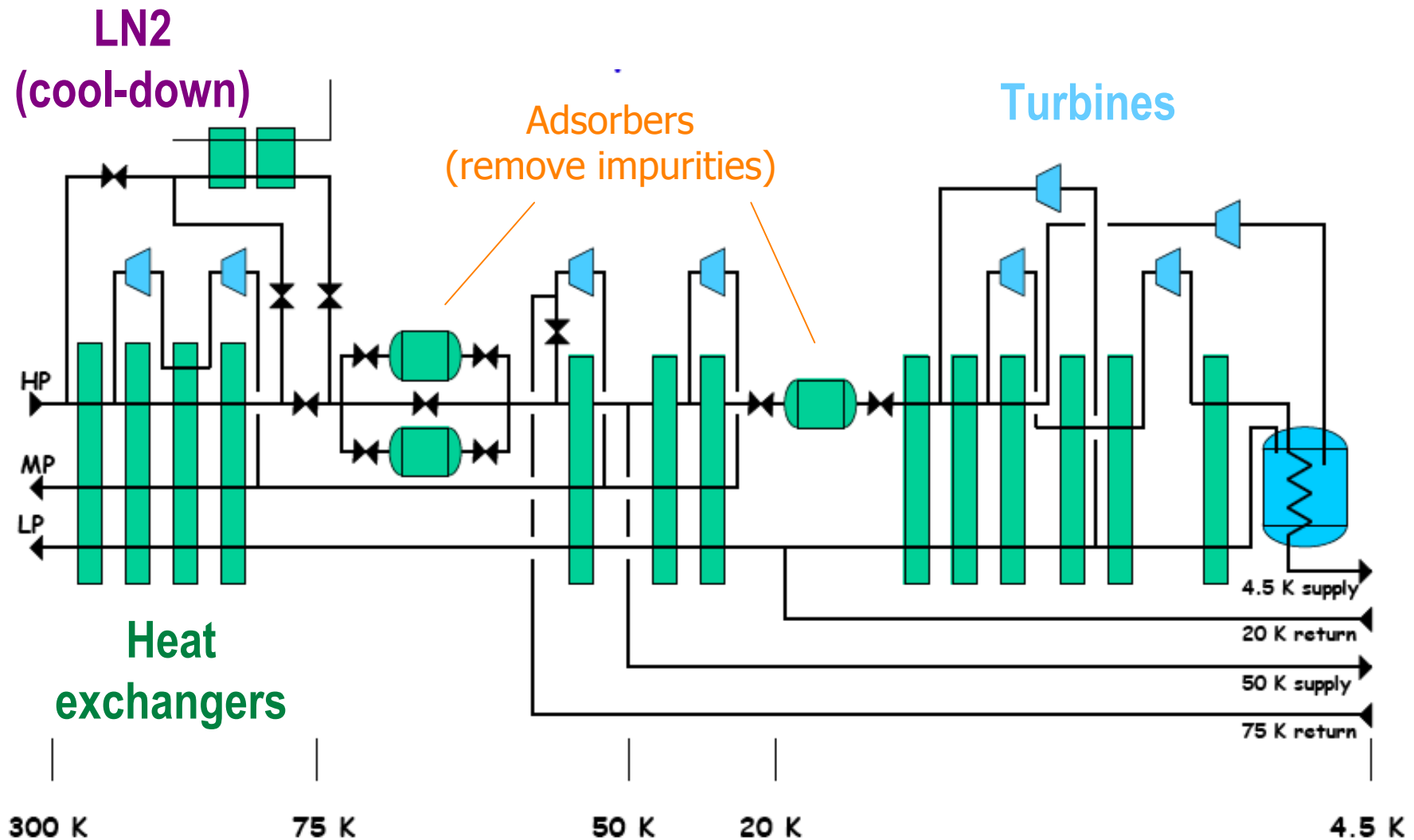


# Oil-injected screw compressor

*(derived from Industrial refrigeration, compressed air)*



# Process diagram, LHC refrigerator 18 kW @ 4.5 K





# LHC 18 kW @ 4.5 K helium cryoplants

33 kW @ 50 K to 75 K, 23 kW @ 4.6 K to 20 K, 41 g/s liquefaction

Diameter: 4 m  
Length: 20 m  
Weight: 100 tons  
600 Input/Output signals



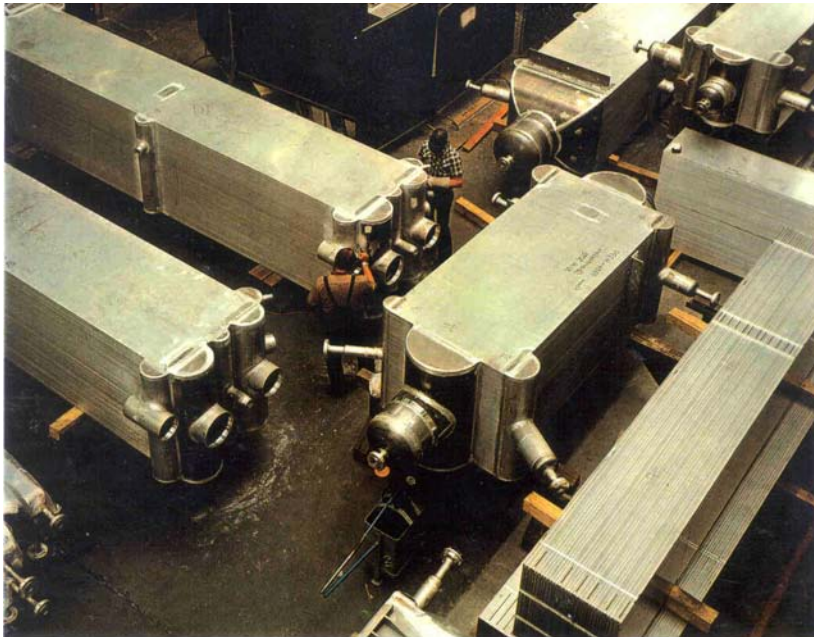
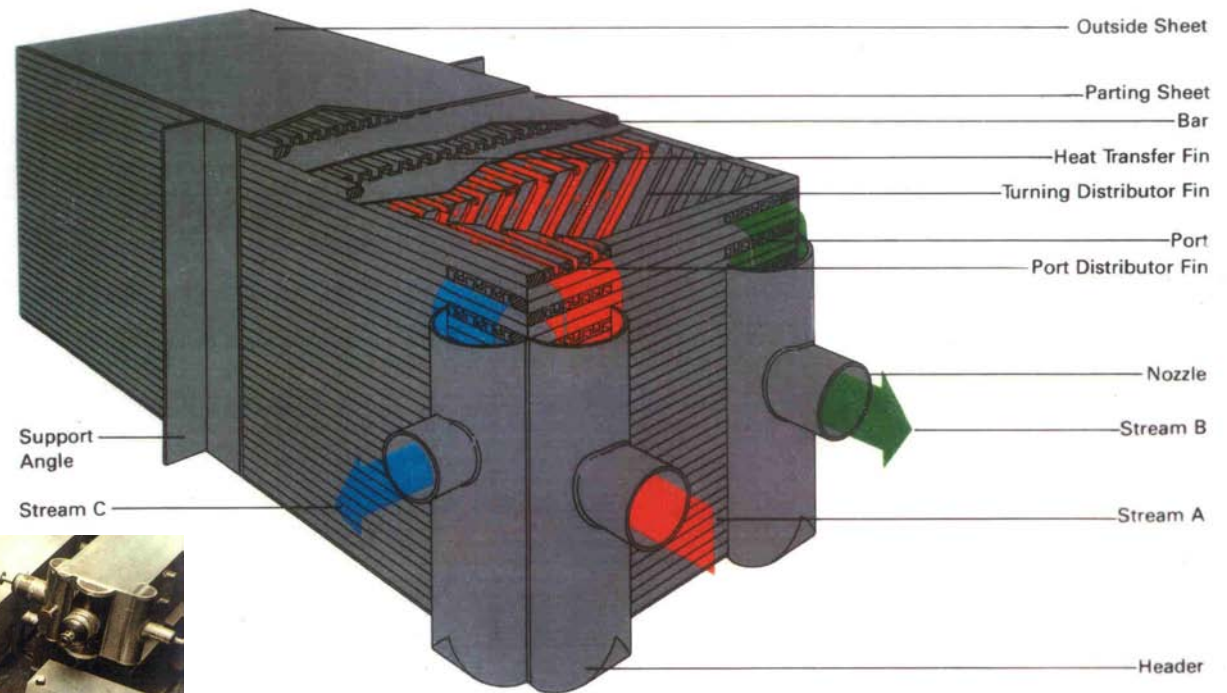
Air Liquide



Linde



# Brazed aluminium plate heat exchanger



Largest used: 1.4 m x 1.4 m x 8 m  
(10 tons)

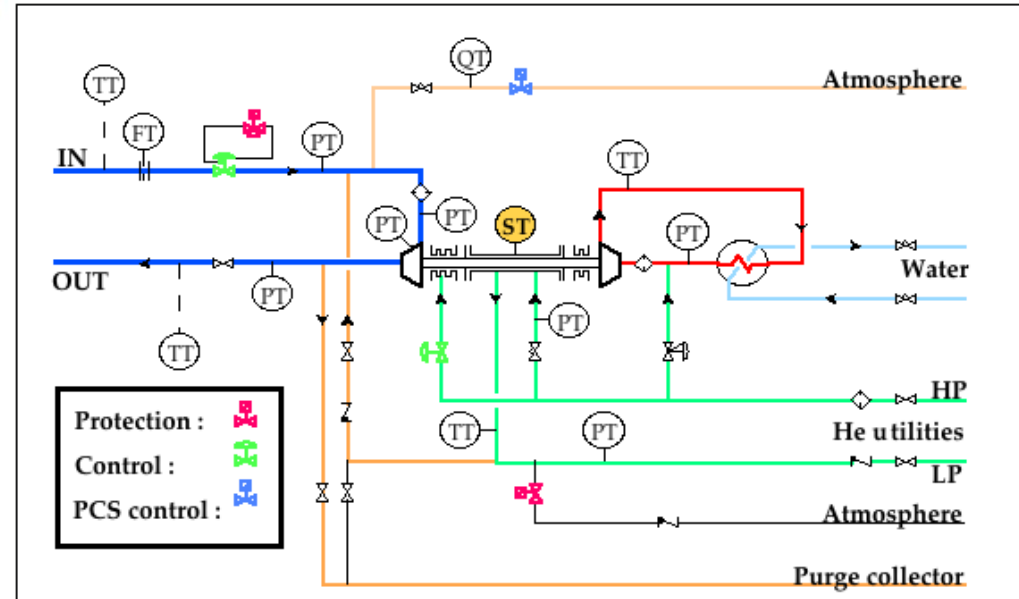
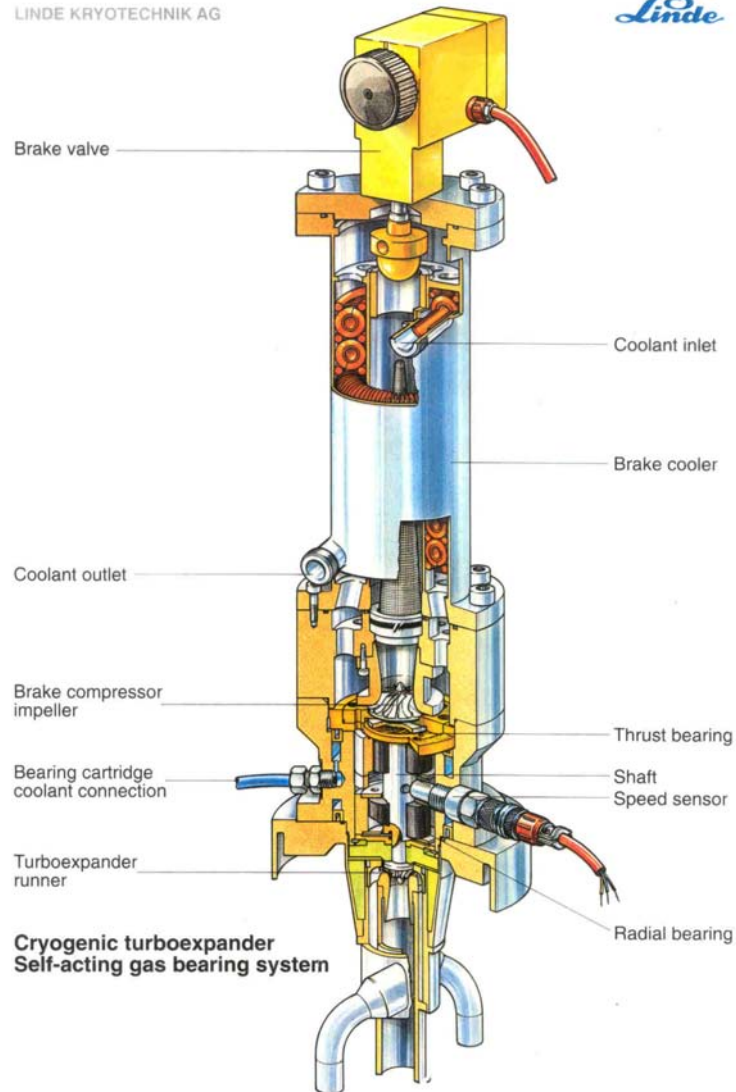


# Cryogenic turbo-expander

*Specific technology "contact free" gas bearings operated at 120'000 rpm*

LINDE KRYOTECHNIK AG

Linde



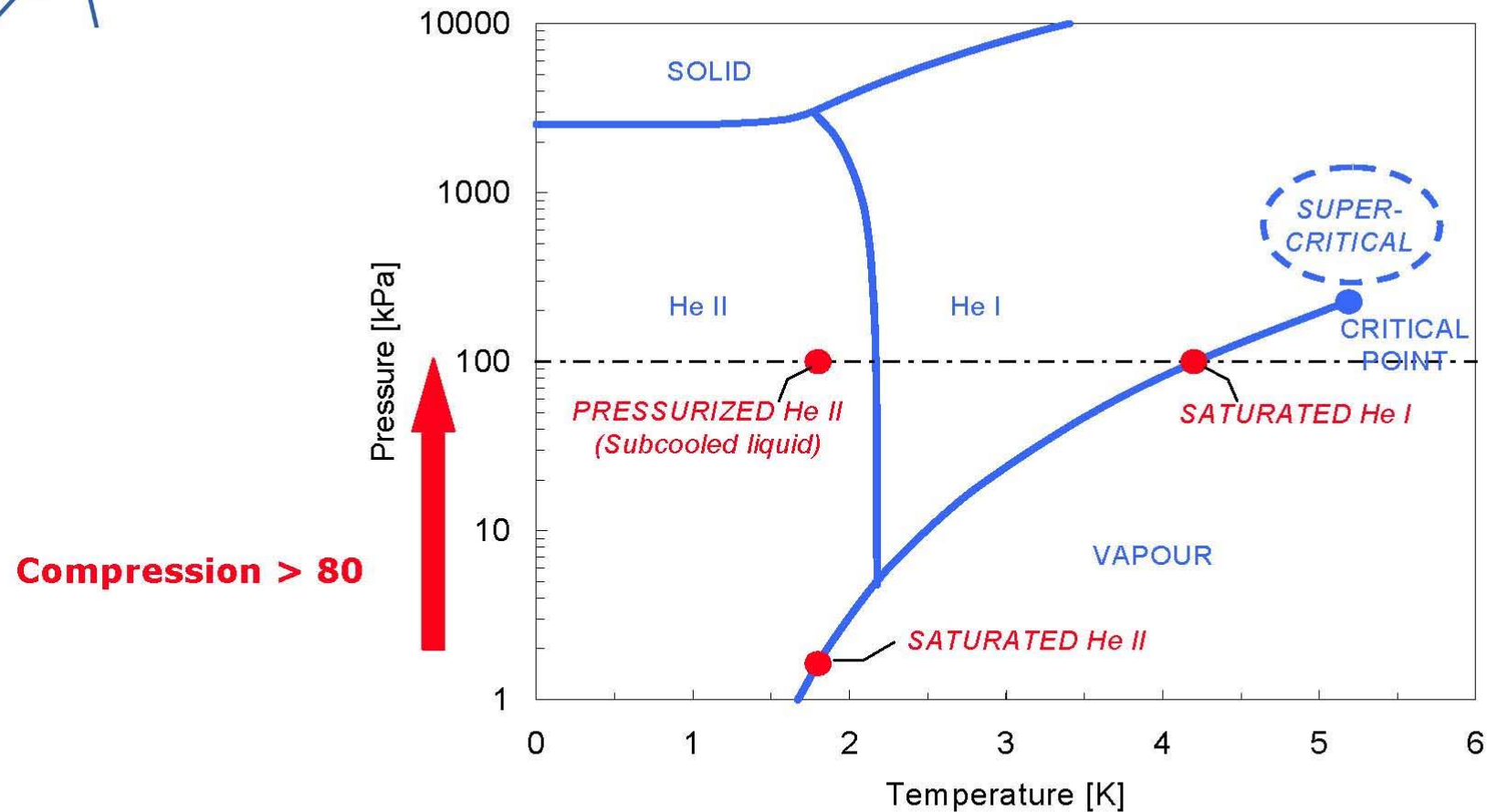
Wheel diameter: 5-15 cm

Shaft length: 20 cm

Rotation: 60'000 to 150'000 rpm



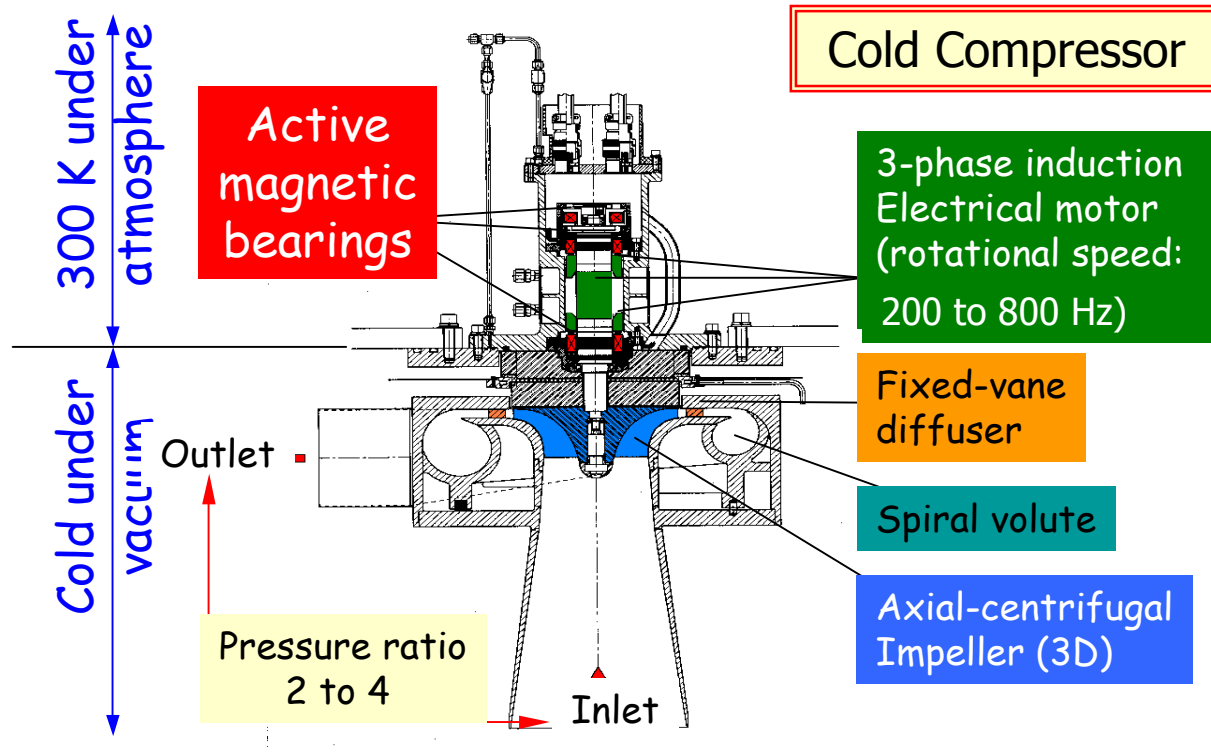
## Challenges of power refrigeration at 1.8 K



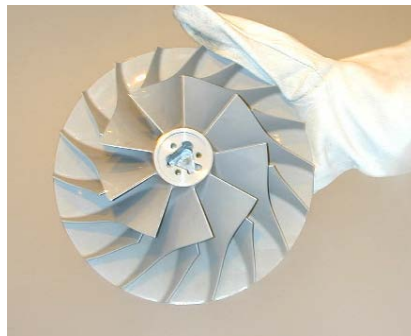
- Compression of large mass flow-rate of He vapor across high pressure ratio  
⇒ intake He at maximum density, i.e. cold
- Need contact-less, vane-less machine ⇒ hydrodynamic compressor
- Compression heat rejected at low temperature ⇒ thermodynamic efficiency



# 1.8K Units with cold compressors (x8)



Specific technology to allow large capacity below 2K



# Contents

- Introduction
- Cryogenic fluids
- Heat transfer & thermal insulation
- Cryogenic distribution & cooling schemes
- Refrigeration & liquefaction
- **Various complements**
- Concluding remarks, references

# Bulk Liquid & Gaseous cryogen storage solutions



## Deliveries in Liquid form:

- 60 trucks LN2 to cool a LHC sector to 80K (14 days-1'200t)
- 20 trucks for external storage of helium (4 months - 90tons)

250m<sup>3</sup> Gaseous He  
(20B - 850kg He)



## How to deal with impurities

- Any liquid or gas other than helium would solidify during the cooling process. This could block the helium flow or degrade moveable components (valves, turbines)
- Typical treatment applied for: Water, air (N<sub>2</sub> and O<sub>2</sub>), H<sub>2</sub> (adsorption on porous medium like activated charcoal, molecular sieve)

### Recommendation:

- ⇒ evacuation of air once circuits are leak-tight (pur helium)
- ⇒ on-line treatment of what could remain or arrive during operation, with target of fraction of ppm(v)



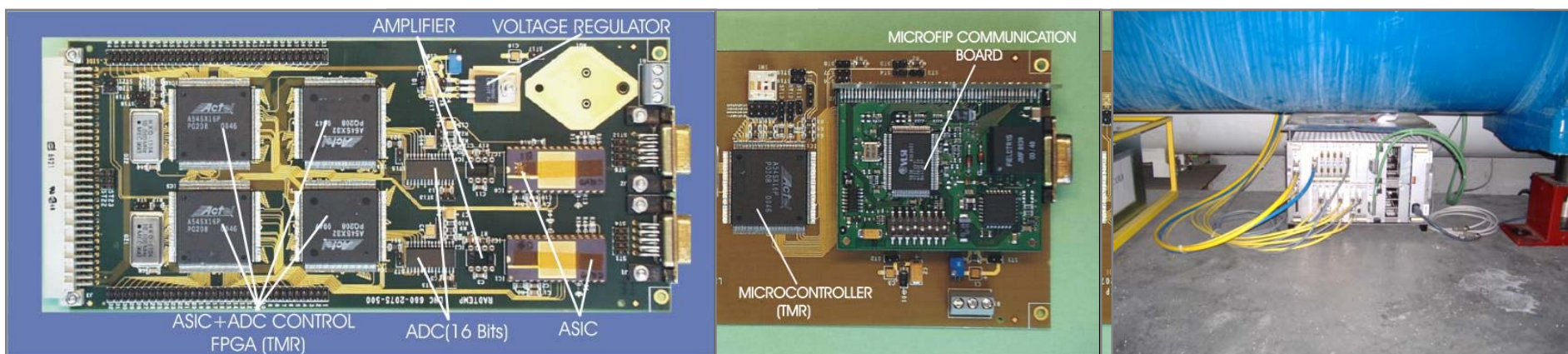
# Thermometry

Industrial instrumentation whenever possible, specific developments when necessary

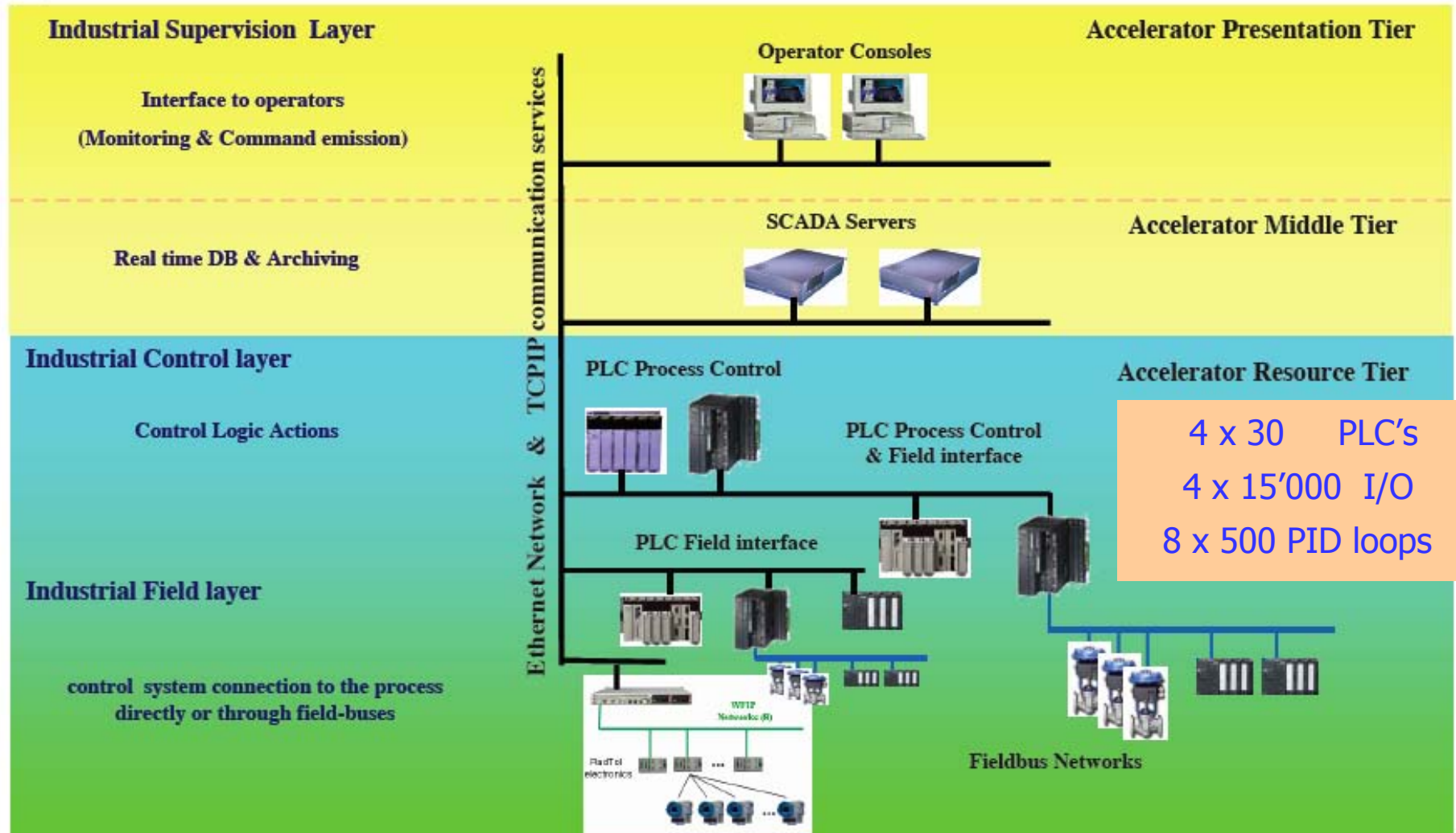
6'000 units, +/- 10 mK @ 2K in LHC radiation conditions



From 'sensor' to 'thermometer' with signal processing

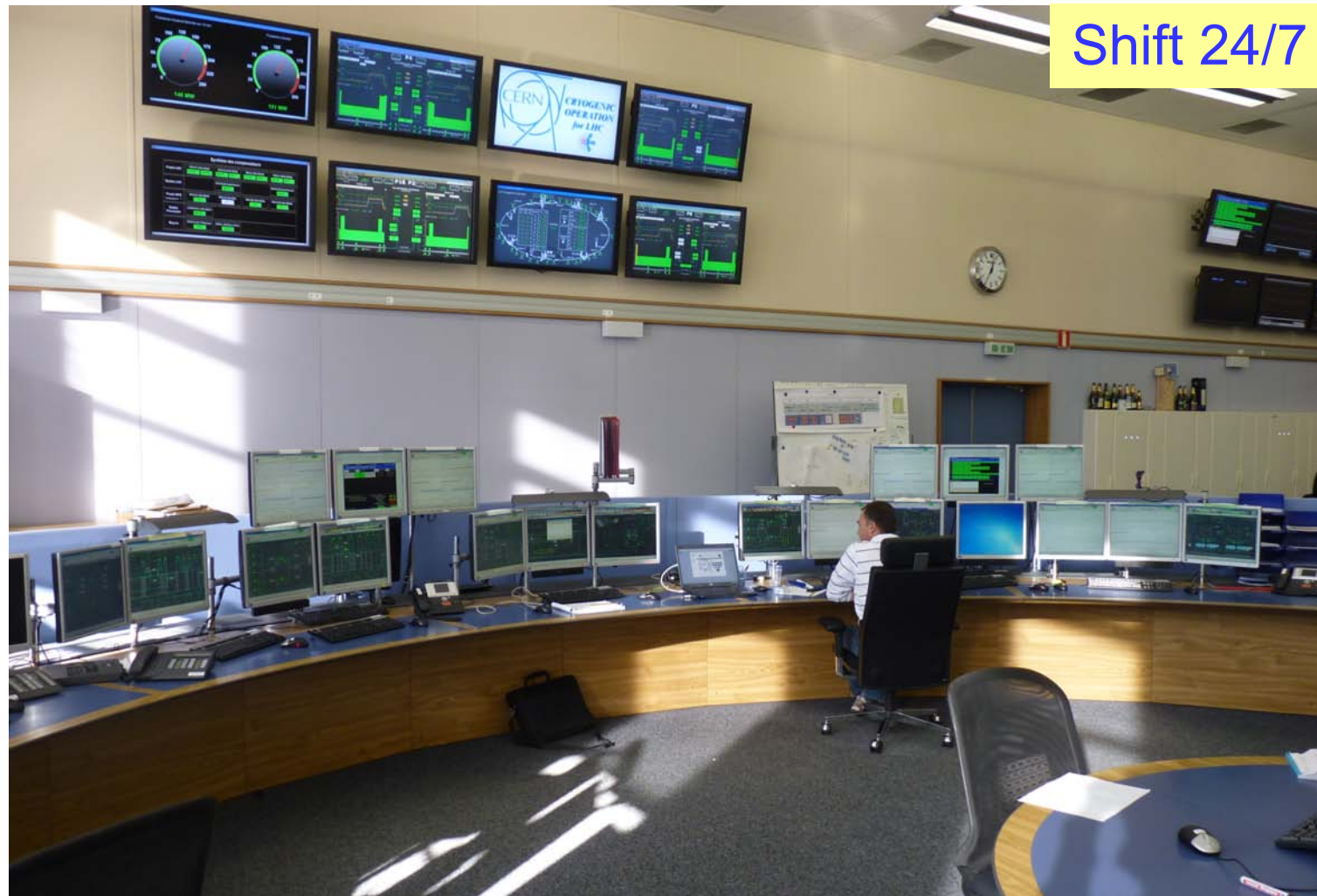


# Industrial Control Architecture





## Cryo operator in Cern Central Control room



Shift 24/7

Fully automated, supervised by a single operator



# Safety notes

- Major risks associated with cryogenic fluids at low temperatures:
  - **Asphyxia:** Oxygen is replaced by a pure
  - **Cold burns:** in case of contact with cold surfaces
  - **Explosion:** pressure rise in case of warm-up at constant volume (1l Liq  $\approx$  700 l gas)
  - **Embrittlement:** Thermal contractions, potential fragile at cold
- Be informed about valid standards, like for pressure vessels, safety devices, transport of cryogens, ...

## Concluding remarks

- Cryogenics serving superconducting systems is now part of all major accelerators and future projects.
- While advanced applications tend to favor “below 2K”, many almost industrial applications are based on “4.5K” and RnD (or demonstrators) continues for “high temperature” applications
- If cryogenic engineering follows well defined rules and standards, there are variants depending on boundary conditions, continents, time of a project...

I could only recommend that demonstrated experience be evaluated and adapted to specific requirements you may have !

*Thanks for your attention,  
and hoping you would (now) be more aware with cryogenics !!!*

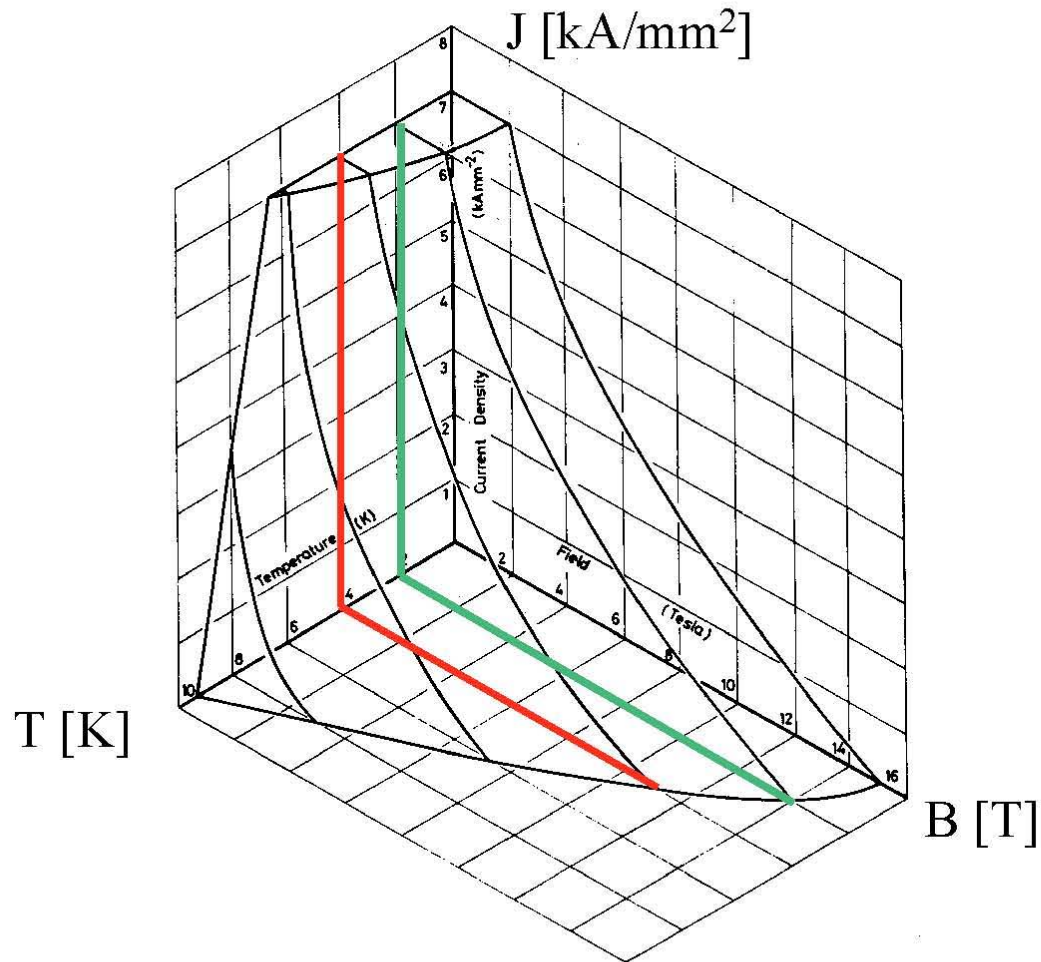
## Some references

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- R.B. Scott, *Cryogenic engineering*, Van Nostrand, Princeton (1959)
- G.G. Haselden, *Cryogenic fundamentals*, Academic Press, London (1971)
- R.A. Barron, *Cryogenic systems*, Oxford University Press, New York (1985)
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- S.W. van Sciver, *Helium cryogenics*, Plenum Press, New York (1986)
- K.D. Timmerhaus & T.M. Flynn, *Cryogenic process engineering*, Plenum Press, New York (1989)
- Proceedings of *CAS School on Superconductivity and Cryogenics for Particle Accelerators and Detectors*, Erice (2002) (+2013)
  - U. Wagner, *Refrigeration*
  - G. Vandoni, *Heat transfer*
  - Ph. Lebrun, *Design of a cryostat for superconducting accelerator magnet*
  - Ph. Lebrun & L. Tavian, *The technology of superfluid helium*
- Proceedings of ICEC and CEC/ICMC conferences

Bonus slides

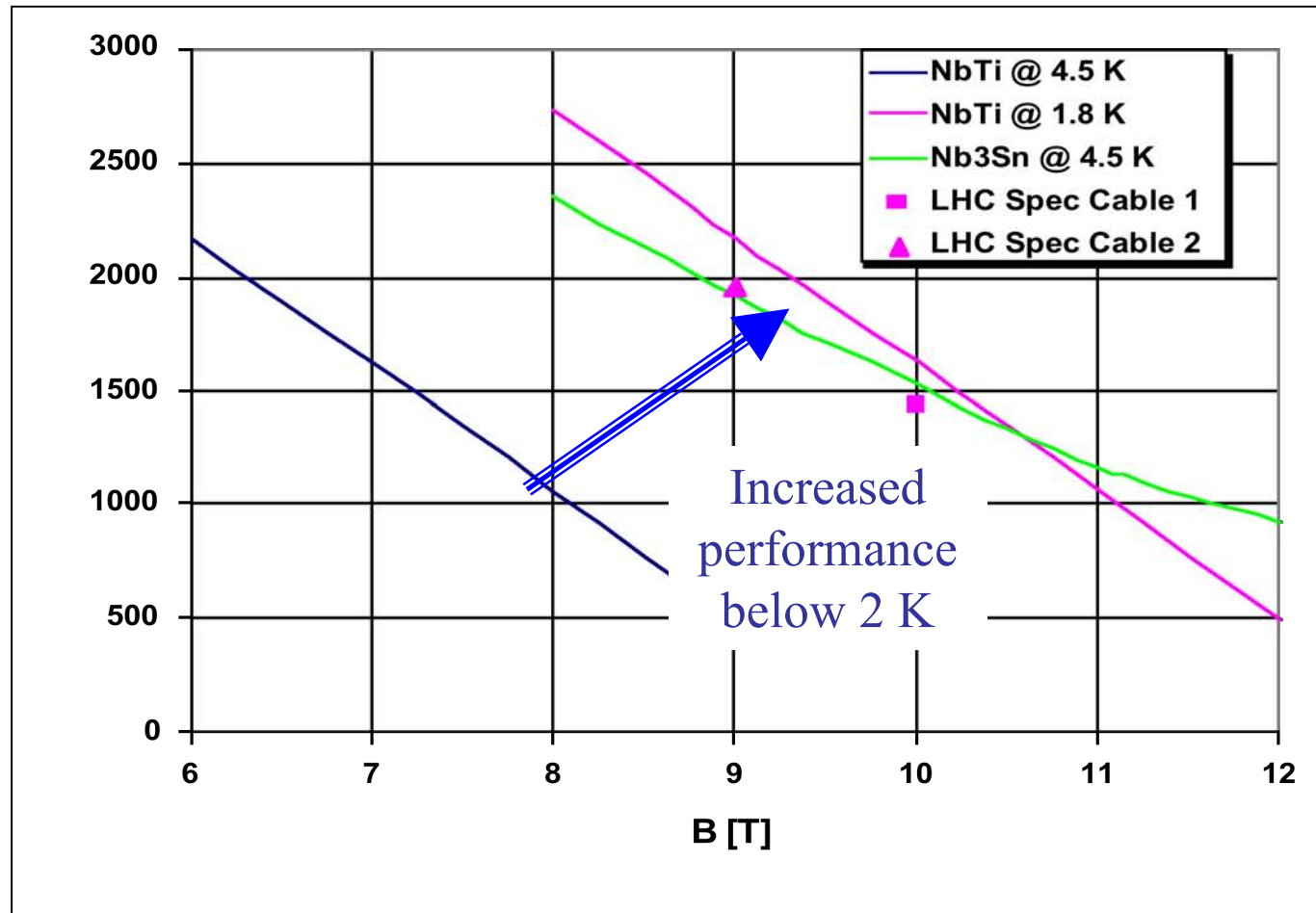


# Operating temperature & performance of superconductors



- Superconductivity only exists in a limited domain of temperature, magnetic field and current density
- Electrotechnical applications require transport current and magnetic field
- Operating temperature of the device must therefore be significantly lower than the critical temperature of the superconductor

# Superconducting magnets

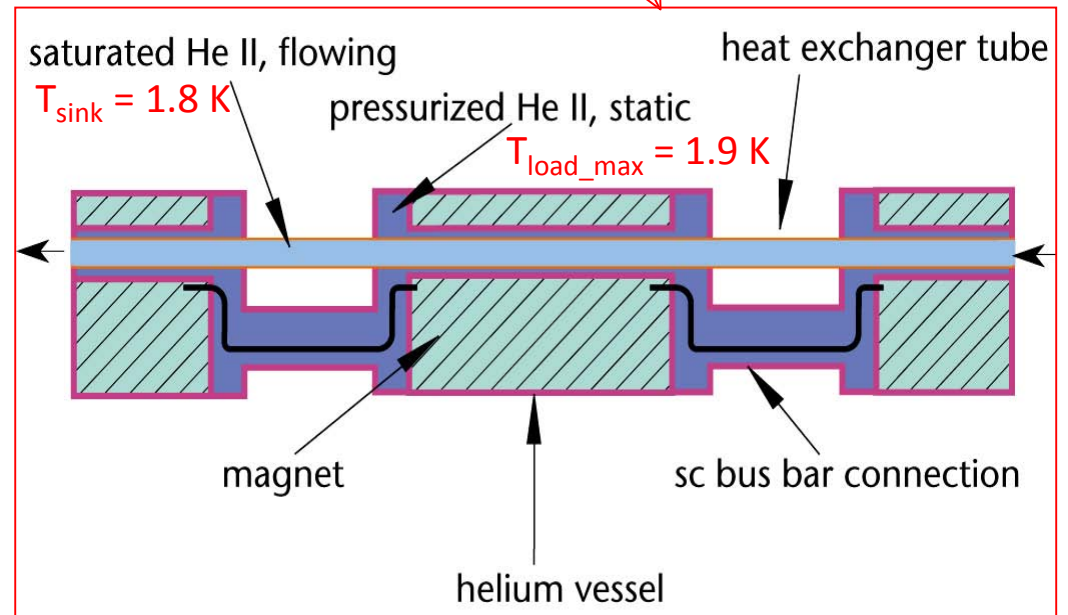
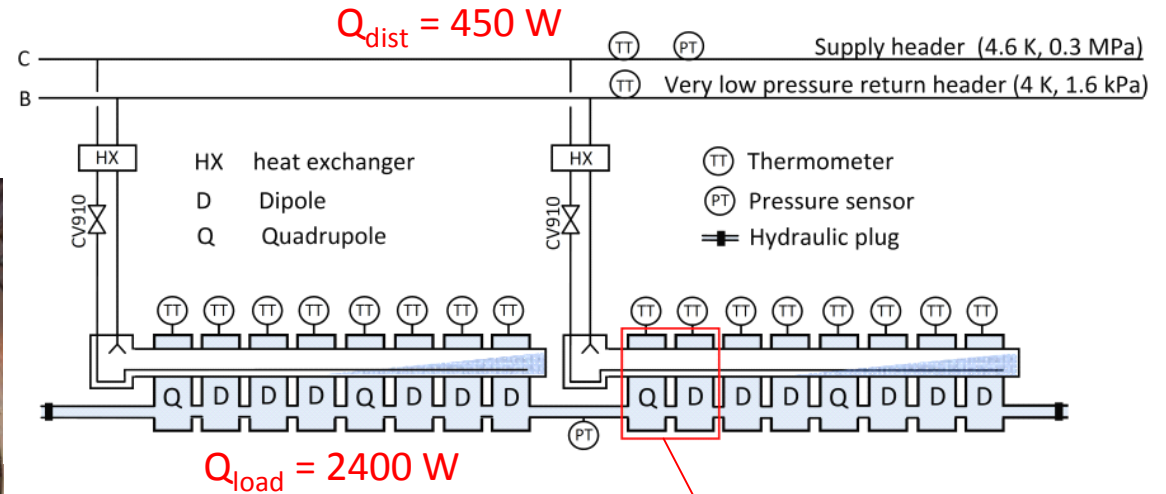


# LHC sector cooling scheme

## Pressurized/saturated He II



37'500 tons at 1.9 K

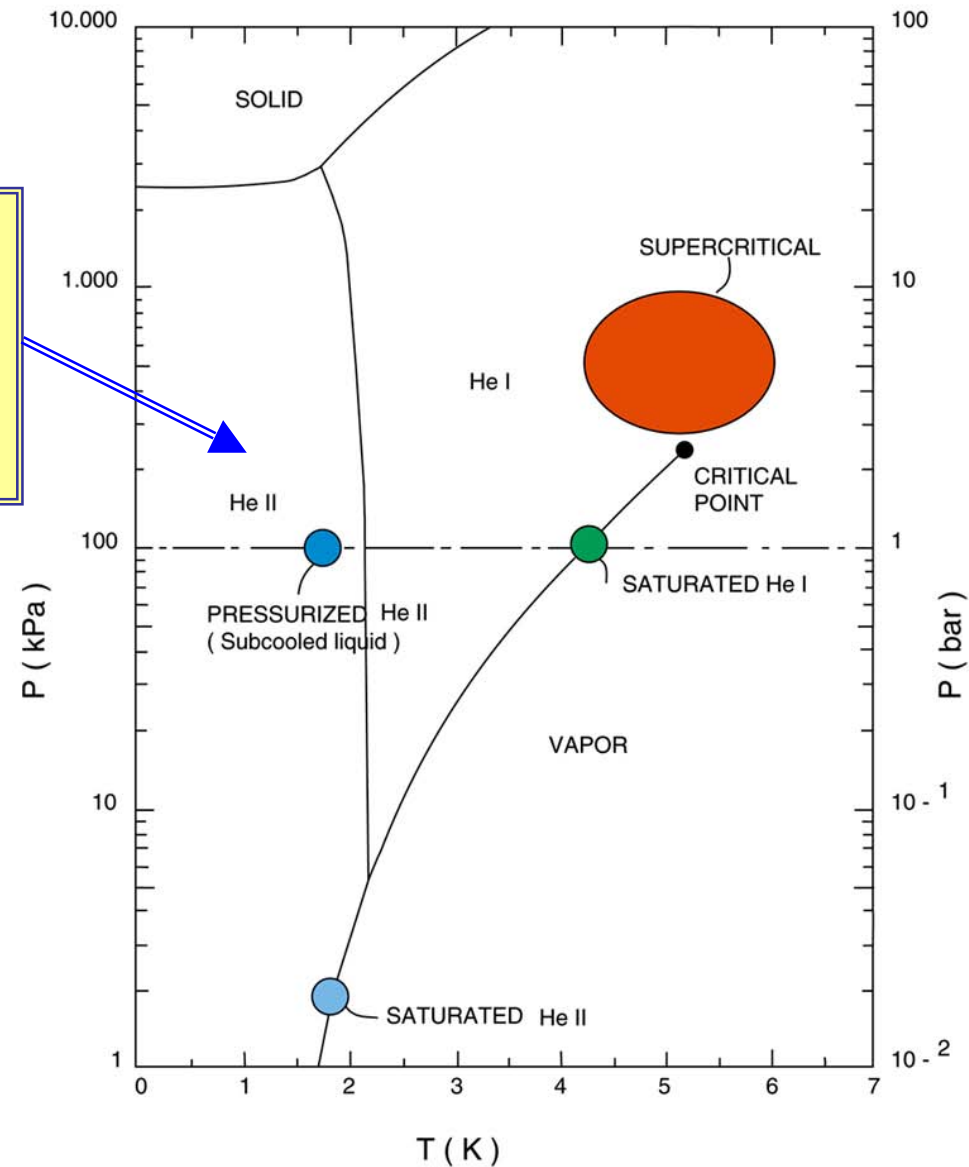




# Helium phase diagram

Superfluid Helium:

- Lower viscosity
- Larger heat transfer capacity



# Basic thermodynamics at low temperature

- Minimum refrigeration work  $W_{\min}$  to extract heat  $Q$  at temperature  $T$  and reject it at ambient temperature  $T_a$

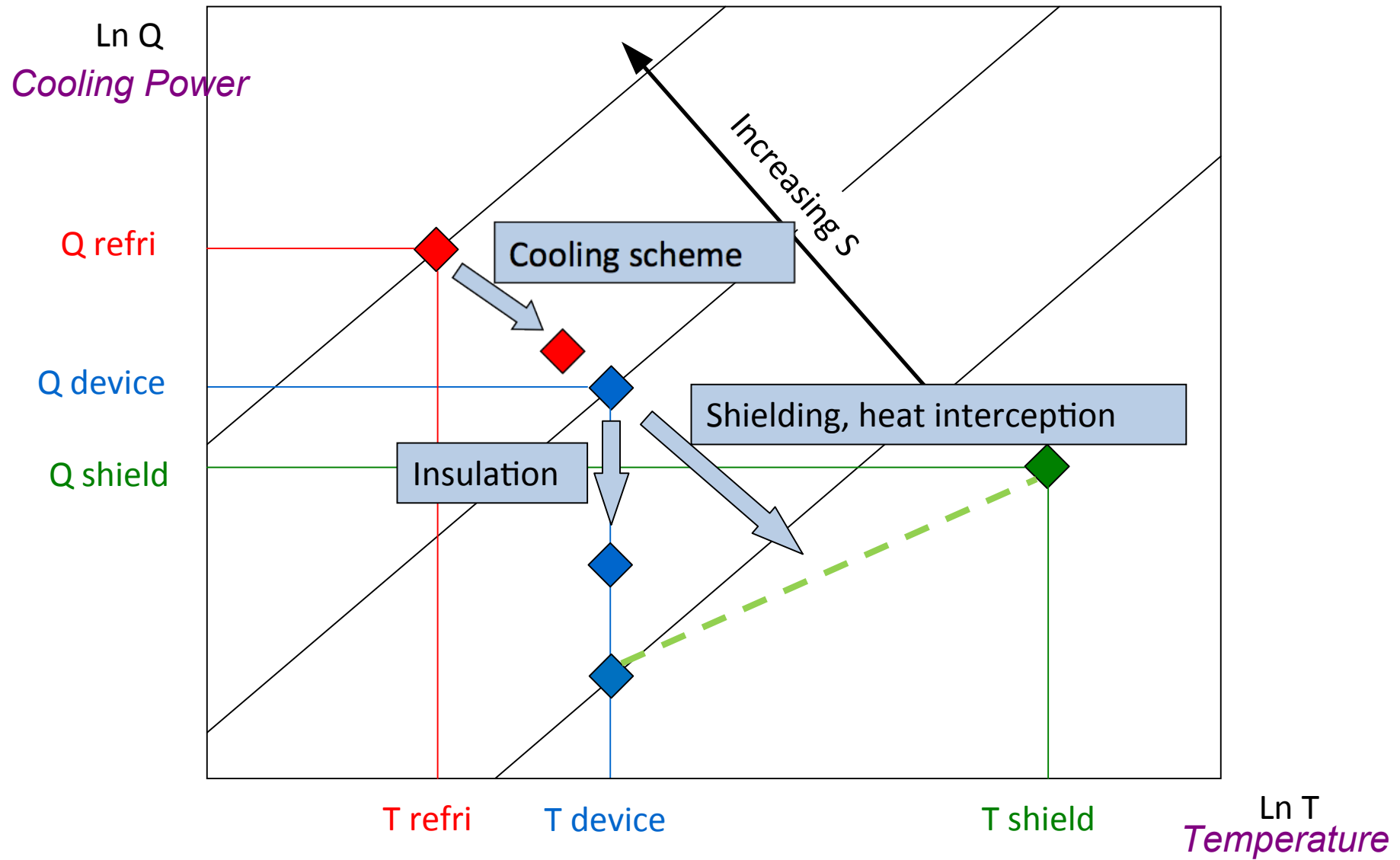
$$W_{\min} = Q (T_a/T - 1) = T_a \Delta S - Q$$

- At cryogenic temperature  $T \ll T_a$

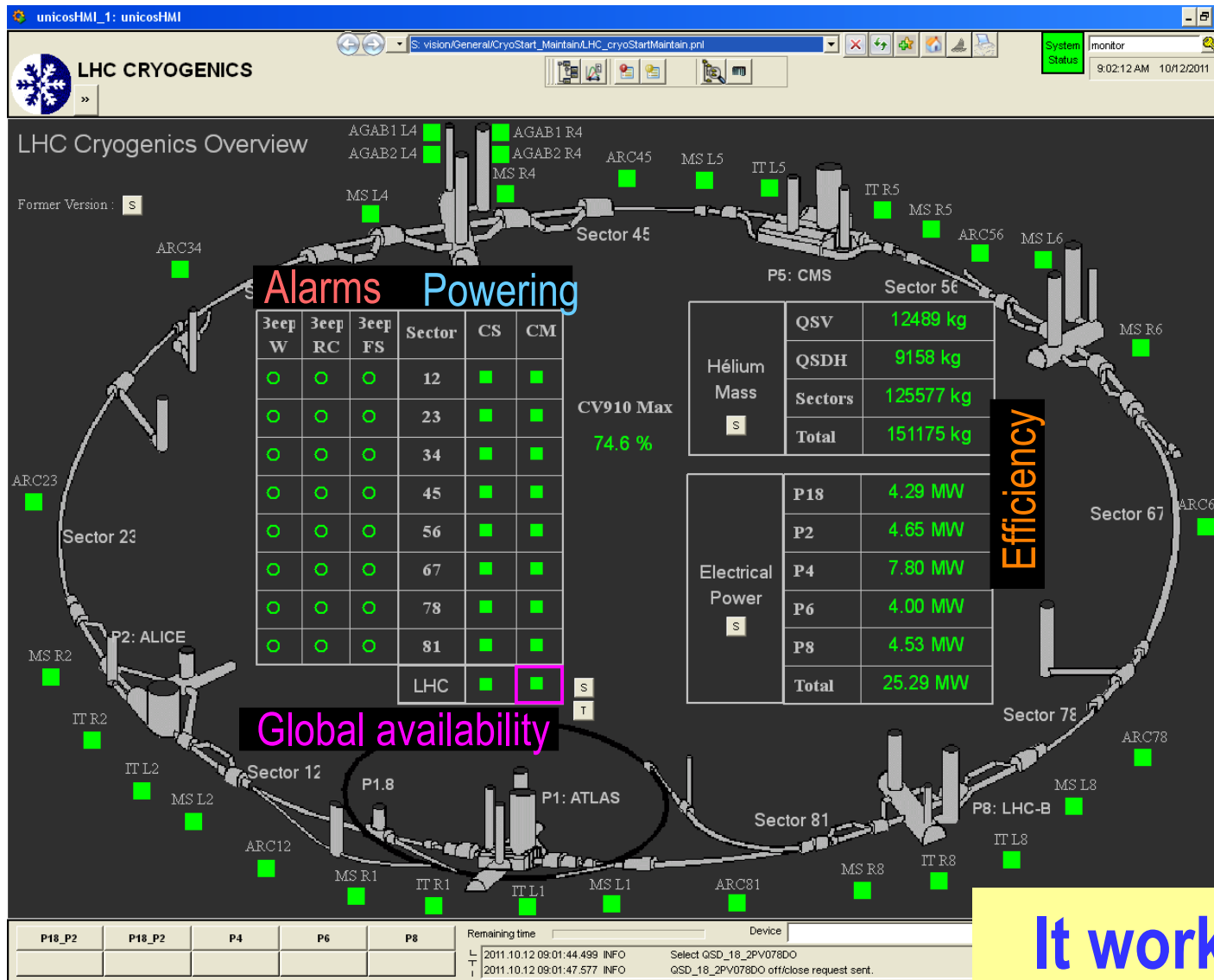
$$W_{\min} \approx Q T_a/T \approx T_a \Delta S$$

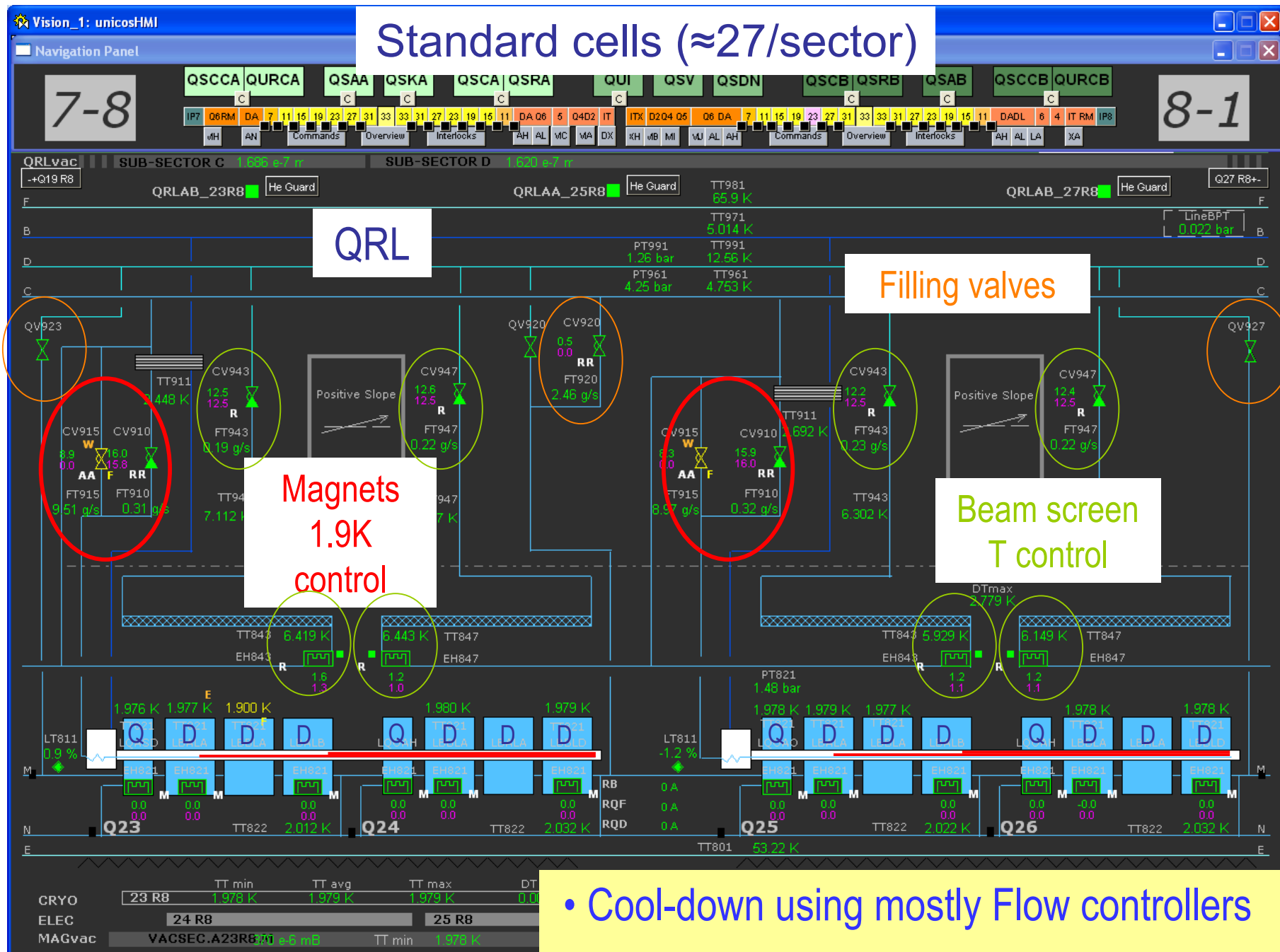
- entropy is a good measure of the cost of cryogenic refrigeration
- strategies minimizing  $\Delta S$  improve cryogenic design

# Cryogenic design strategies



# Operation, indicators





- Cool-down using mostly Flow controllers
- P, T, L controllers at operating conditions